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Full Length Article Pore-scale study of CO₂ desublimation in a contact liquid

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ABSTRACT

Cryogenic carbon capture (CCC) designed to operate in a contact liquid is an innovative technology for capturing CO₂ from industrial flue gases, helping mitigate climate change. Understanding CO₂ desublimation properties in a contact liquid is crucial to optimizing CCC, but is challenging due to the complex physics involved. In this work, a multiphysics lattice Boltzmann (LB) model is developed to investigate CO₂ desublimation in a contact liquid for various operating conditions, with the multiple and fully-coupled physics being incorporated (i.e., two-phase flow, heat transfer across three phases, CO₂ transport between the gas and liquid, homogeneous and heterogeneous desublimation of CO_2 , and solid CO_2 generation). The CO_2 desublimation process in a contact liquid is well reproduced. Moreover, parametric studies and quantitative analyses are set out to identify optimal conditions for CCC. The decreasing liquid temperature (T_i) and flue gas temperature (T_0) are found to accelerate the CO₂ desublimation rate and enhance the CO₂ capture velocity (v_c) . However, excessively low T_i and T_0 values should be avoided. These conditions increase the energy consumption of cooling while only marginally improving $v_{\rm c}$, due to the limited CO $_2$ supply. The CCC system performs effectively when purifying flue gases with high CO $_2$ content (Y_0). This is because the large Y_0 accelerates the CO₂ desublimation rate and enhances the overall CO₂ capture efficiency. A high gas injection velocity (or Pe) is beneficial for amplifying v_c by increasing the gasliquid interfaces and enhancing the CO₂ supply. Nevertheless, too high a Pe should be avoided, as it hinders the transport of CO₂ to the liquid or solid CO₂ surfaces, ultimately restricting the amount of CO₂ available for desublimation and inhibiting the enhancement of v_c . This study develops a viable LB methodology to investigate CO₂ desublimation in a contact liquid for varying conditions, which advances the knowledge base of CCC and facilitates its industrial applications.

1. Introduction

Climate change is widely recognized as one of the most pressing environmental challenges. There is a consensus among researchers that such an issue is mainly caused by human activities that yield critical CO2 emissions (Kamkeng et al., 2021; Lei et al., 2023; Mac Dowell et al., 2017). To mitigate anthropogenic CO₂ emissions, various carbon capture technologies have been developed, including absorption, membrane, adsorption, and cryogenic processes (Bose et al., 2024; Naquash et al., 2022). Cryogenic carbon capture (CCC), as a relatively emerging technology, has garnered increasing research interest. Among the existing CCC designs, the one utilizing direct contact heat exchangers (or cryogenic liquids) is operationally simple (Fazlollahi et al., 2017; Jensen, 2015). During the operation of liquid-based CCC, the flue gas containing multiple components is injected into the CCC system and cooled by the liquid to low temperatures. This subsequently induces the desublimation of CO₂ and produces solid CO₂ particles within the liquid. Finally, these solid CO₂ particles are separated from the liquid and recycled for further applications. This system presents significant application potentials and research interests due to its numerous advantages, like low energy penalty, process simplicity, cost-effectiveness, and absence of chemicals (Font-Palma et al., 2021; Jensen et al., 2015). Nevertheless, the development of CCC is still in its nascent stage because of technical problems and knowledge gaps, including inappropriate operating conditions, hydrocarbon emissions, and unclear desublimation kinetics (Jensen, 2015; Pan et al., 2013). Therefore, investigating CO_2 desublimation in the liquid-based CCC system is crucial to advancing the understanding of CCC and providing guidance for its development.

The problem of CO_2 desublimation in a contact liquid involves several fully-coupled physics, i.e., two-phase flow, heat transfer across three phases (i.e., contact liquid, flue gas, and solid CO_2), multicomponent transport between the flue gas and contact liquid, homogeneous desublimation within the contact liquid, heterogeneous desublimation on solid CO_2 surfaces, and solid CO_2 generation. For such a complex problem, some experimental and numerical efforts have been devoted to gaining a comprehensive understanding.

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Tuinier et al. (2011a, 2010, 2011b) designed a CCC system using three dynamically operated packed beds for purifying flue gases at the atmospheric pressure. Their experiments demonstrated that the operational cost strongly depended on the initial bed temperature and the CO₂ concentration in the flue gas. Meanwhile, they proposed a onedimensional (1D) model to solve the mass and energy balance during the desublimation of CO2 on packed bed surfaces. They indicated that an initial bed temperature above -120°C could exponentially diminish the amount of solid CO₂ captured and strongly amplify the specific cooling duty (Tuinier et al., 2011b). However, such CCC systems were constrained to the atmospheric pressure and gases with low CO₂ contents. For this concern, Ali et al. (2016, 2014) designed a multiple cryogenic desublimation based pipeline network to remove H₂O and CO₂ from flue gases with high CO₂ contents and high pressures. In parallel, another CCC system based on Stirling coolers (SC) was developed for CO2 capture (Song et al., 2012a; 2014; 2013; 2012b). After conducting extensive experiments, the optimal gas flow rate and SC temperatures were identified for the gas cooling stage to optimize the system performance. A two-dimensional (2D) model was built to examine the growth of CO₂ frost layers on SC surfaces, demonstrating that the accumulated frost layer on SC increased the thermal resistance and raised the surface temperature (Song et al., 2013).

The above findings advanced the knowledge base of CO₂ desublimation on solid surfaces, but failed to explore CO₂ desublimation properties within a contact liquid. To cover this void, the feasibility and efficiency of the liquid-based CCC system was explored. Jensen et al. Jensen (2015), Jensen et al. (2015) proposed a CCC system with an external cooling loop to cool the contact liquid for subsequent CO₂ desublimation. They employed computational simulations and experimental validations to demonstrate that this system was capable of removing up to 99% of CO₂ from the flue gas continuously while maintaining low energy penalties. A CCC natural-gas treatment system was then designed for capturing CO₂ from the natural gas (Fazlollahi et al., 2017). Through experimental and numerical investigations, it was recommended that the high operating pressure and the low methane content in natural gas were beneficial for improving the CO2 capture performance. More recently, a 1D model incorporating a new mass deposition scheme was proposed to consider both the heterogeneous nucleation of solid CO_2 on the gas-solid interface and the homogeneous nucleation within the gas phase (Debnath et al., 2019). The condensed CO₂ thickness was simulated and mapped over the space-time domain, based on which spatiotemporal evolutions of the gas phase fraction and the CO₂ separation strength were further calculated.

These existing studies have evaluated the CO_2 capture performance in a contact liquid to a certain extend. Despite these achievements, the multiple physics behind CO_2 desublimation and their complex interactions have not been thoroughly examined across extensive operating conditions. Moreover, the current simulations were conducted on volume-averaged scales, ignoring pore-scale details, such as the flow of gas bubbles within liquid, intricate structures of solid CO_2 , conjugate heat transfer across three phases, and CO_2 transport from the gas to liquid phases. The pore-scale information, however, is crucial to accurately constructing empirical correlations required in volume-averaged simulations (Xu et al., 2018a; 2018b; Yang et al., 2021). Therefore, it is essential to develop pore-scale models for comprehensively investigating the underlying physics of CO_2 desublimation in a contact liquid.

Over the past three decades, the lattice Boltzmann (LB) method has been developed into a powerful alternative to conventional solvers for simulating complex fluid flows with phase change in porous media at the pore scale (Chen et al., 2022; Lei et al., 2019; Li et al., 2016; Wei et al., 2022). This is attributed to its distinctive advantages. For instance, the LB method, based on the kinetic theory, can seamlessly handle multiple physics, including multiphase flow, phase change, and heat and mass transfer. Additionally, elementary mechanical rules (e.g., bounce-back) make it easy for the LB method to treat complex boundary conditions in pore structures. The simple implementation and high parallelism of

the LB method further enhance its computational efficiency, making it well-suited for pore-scale simulations. Accordingly, there exist plentiful LB models for independent investigations of the multiple physics behind CO₂ desublimation in a contact liquid, including two-phase flow, multicomponent transport, conjugate heat transfer, homogeneous and heterogeneous solidification, and solid structure evolutions. For instance, LB models were proposed for simulating two-phase flows at the pore scale, where the multiple-relaxation-time (MRT) scheme and the multiple distribution functions were incorporated to improve the model stability and consider the multicomponent transport, respectively (Li et al., 2016; Wang et al., 2022; 2023). The feasibility of LB model was also verified in simulating species transport across gas-liquid interfaces, with the mass jump described by Henry's law. This mass jump was successfully implemented by incorporating either the continuum species transport source term or the boundary scheme in LB simulations (Chen et al., 2020; Yang et al., 2022). In the LB community, the conjugate heat transfer across different phases could be solved without resorting to conventional extrapolations or iterations (He et al., 2019). A half-lattice division scheme was proposed to model the conjugate heat transfer, but it was limited to cases with steady-state solutions or uniform heat capacities (Wang et al., 2007; Wang and Pan, 2008). Considering this, a new source term derived from the energy conservation equation was introduced in LB equations to realize the conjugate heat transfer during coke combustion and methane hydrate dissolution (Karani and Huber, 2015; Lei et al., 2021; Zhang et al., 2019). Finally, the homogeneous CO₂ desublimation in the contact liquid can be described by the classical nucleation theory, while the heterogeneous CO₂ desublimation on solid CO₂ surfaces can be treated by boundary conditions for mass conservation and conjugate heat transfer (Jensen, 2015; Lei et al., 2023; 2024). In applications like salt precipitation and CO₂ sequestration, the homogeneous nucleation in the liquid phase was successfully solved by introducing source terms in LB simulations (Chen et al., 2014; Yang et al., 2023). To enforce the heterogeneous solidification on solid surfaces, a general bounce-back scheme was built to treat the mass conservation condition, which displayed the second-order accuracy at complex boundaries (Zhang et al., 2012). Along with these solidification simulations, the volume of pixel (VOP) scheme coupled with the mass balance equation was applied to explicitly track the pore-scale solid structure evolutions (Chen et al., 2014; Kang et al., 2006).

In general, separate LB models are available for individually modeling the key physics behind CO₂ desublimation in a contact liquid at the pore scale. However, the combination and interactions of these complex physics have not been achieved by a single LB model, which is indeed changeling. In our recent studies, an LB model has been proposed to simulate CO₂ desublimation on solid surfaces. This model successfully reproduced CO₂ desublimation properties under various conditions and identified different desublimation regimes (Lei et al., 2023, Lei et al., 2024). Nevertheless, the presence of contact liquid was not considered, thus neglecting the two-phase flow, transport of CO₂ between the gas and liquid phases, homogeneous CO₂ desublimation in the liquid phase, and temperature variations across three phases. To fill these gaps, this study aims to extend our recently proposed LB model to investigate CO₂ desublimation in a contact liquid during CCC at the pore level, accounting for the multiphysics involved and their interactions. The CO₂ capture performance of liquid-based CCC will be evaluated for different operating parameters, which are expected to elucidate the optimal conditions and enhance the understanding of CCC.

2. Mathematical and physical models

This study investigates the multiphysics behind CO_2 desublimation in a contact liquid during CCC. Fig. 1 depicts a sample desublimation column filled with contact liquid (Fazlollahi et al., 2017). From the inlet, an incompressible warm flue gas containing CO_2 and N_2 , with mass fractions Y_0 and $(1 - Y_0)$ respectively, is injected at temperature T_0 , pressure p_0 , and velocity u_0 . Initially, the temperature of the contact liquid (T_i)



is set above the freezing point of N₂ but below that of CO₂. As a result, N₂ exits the domain without undergoing phase change, but CO₂ desublimates and remains within the liquid. Specifically, the CO₂ component diffuses across the gas-liquid interface and desublimates to form solid CO₂ nuclei in the contact liquid. As CO₂ desublimation continues both in the liquid and on solid CO₂ surfaces, CO₂ nuclei gradually grow into large solid CO₂ particles. The generation of solid CO₂ particles affects the two-phase flow (i.e., the flue gas and contact liquid) in the desublimation column. The modified two-phase convection and gas-liquid interfaces impact the transport of CO₂ component. Meanwhile, considering the exothermic nature of desublimation and the incoming warm flue gas, the local temperature gradually increases with conjugate heat transfer through different phases. The variations in CO₂ mass fraction and temperature in turn influence the CO₂ desublimation rate. Therefore, the underlying physics of CO₂ desublimation, namely, the two-phase flow, heat and mass transfer, CO₂ desublimation mechanisms, and solid CO₂ structure evolution, are fully coupled.

Before constructing governing equations for describing desublimation of CO₂ at the pore scale, some simplifications and assumptions are made as follows. (1) The capture (or desublimation) of CO_2 is investigated, while the cooling of the system and the recovery of the desublimated CO_2 are neglected. (2) The incompressible flue gas obeys the ideal gas law, and it is represented as a mixture of CO_2 and N_2 . (3) Fick's law is applied to describe the mass diffusion of gas components in both the gas and liquid phases, and the mass jump across gas-liquid interfaces is determined by Henry's law. (4) The desublimation of CO₂ is simplified into two steps: nucleation and growth. The homogeneous desublimation (i.e., nucleation) within the contact liquid follows the classical nucleation theory, while the heterogeneous desublimation (i.e., growth) on solid CO₂ surfaces is determined by the local deviation from the gas/liquid-solid equilibrium state. (5) Thermophysical properties of the gas, liquid, and solid phases are set as constants in relation to the initial condition. (6) The movement of solid CO_2 is not considered.

Based on the aforementioned assumptions and introductions, a set of governing equations has been developed to model the desublimation of CO_2 during CCC at the pore scale. This includes the continuity Eq. (1), the incompressible Navier–Stokes Eq. (2), the conservation Allen–Cahn Eq. (3) for the two-phase flow, the species conservation Eq. (4) for CO_2 transport in the gas and liquid phases, and the energy balance Eq. (5) for

Fig. 1. The schematic diagram of CO_2 desublimation in a contact liquid at the pore scale.

heat transfer through the gas, liquid, and solid phases (Lei et al., 2023; Liang et al., 2018; Yang et al., 2023).

$$\nabla \cdot (\rho \boldsymbol{u}) = \boldsymbol{0},\tag{1}$$

$$\frac{\partial(\rho \boldsymbol{u})}{\partial t} + \nabla \cdot (\rho \boldsymbol{u} \boldsymbol{u}) = -\nabla p + \nabla \cdot \left[\rho \nu \left(\nabla \boldsymbol{u} + (\nabla \boldsymbol{u})^T\right)\right] + \boldsymbol{F}_s + \boldsymbol{F}_g, \tag{2}$$

$$\frac{\partial \phi}{\partial t} + \nabla \cdot (\phi u) = \nabla \cdot [M(\nabla \phi - \lambda n)], \qquad (3)$$

$$\frac{\partial Y}{\partial t} + \nabla \cdot (Y\boldsymbol{u}) = \nabla \cdot \left[D \left(\nabla Y + \Phi_{CST} \right) \right] + S_n,$$

$$\Phi_{CST} = -\frac{Y(H-1)}{H\phi + (1-\phi)} \nabla \phi,$$
(4)

$$\frac{\partial(\rho c_p T)}{\partial t} + \nabla \cdot \left(\rho c_p T u\right) = \nabla \cdot \left(\rho c_p \alpha \nabla T\right) + Q.$$
(5)

Here, u, ρ , and v represent fluid velocity, density, and kinematic viscosity, respectively. t is time. ϕ is the phase indicator, with $\phi = 0.0$ for gas, $\phi = 1.0$ for liquid, and $\phi = 0.5$ for the gas–liquid interface. The unit vector normal to the interface is calculated by,

$$\boldsymbol{n} = \frac{\nabla \phi}{|\nabla \phi|}.\tag{6}$$

M denotes mobility, and λ is a function of ϕ as,

$$\lambda = \frac{4\phi(1-\phi)}{W},\tag{7}$$

with *W* representing the gas–liquid interface thickness. The volumetric body force $F_g = (\rho - \rho_l)g$ is imposed on the system, with ρ_l being the liquid density and *g* being the gravitational acceleration. One of the widely used potential forms in the phase-field method is applied to determine the surface tension force F_s as,

$$\boldsymbol{F}_{s} = \boldsymbol{\mu}_{\phi} \nabla \phi. \tag{8}$$

The chemical potential μ_{ϕ} is defined as,

$$\mu_{\phi} = 4\beta\phi(\phi - 1)(\phi - 0.5) - \kappa\nabla^2\phi, \qquad (9)$$

where $\kappa = 3\sigma W/2$, $\beta = 12\sigma/W$, and σ is the surface tension.

In the mass and energy conservation Eqs. (4)–(5), *Y* and *D* are the CO_2 mass fraction and diffusion coefficient, respectively. S_n is the source term caused by homogeneous desublimation of CO_2 , and Φ_{CST} is the source term for CO_2 mass jump at the gas–liquid interface. Such a mass jump is described by $Y_l = HY_g$, with *H* being the Henry coefficient, and Y_l and Y_g representing the CO_2 mass fractions in the liquid and gas phases, respectively. *T* is the local temperature, c_p is the specific heat capacity at constant pressure, α is the thermal diffusivity, and *Q* is the heat released from CO_2 desublimation.

As shown in Fig. 1, the exothermic CO_2 desublimation is expressed as,

$$\operatorname{CO}_2(g) \to \operatorname{CO}_2(s) + Q. \tag{10}$$

Here, the gas and solid phases of CO_2 are denoted by g and s, respectively. The liquid is gradually warmed up by the exothermic desublimation process and the incoming warm gas. Once the local temperature reaches the freezing point of CO_2 , the desublimation of CO_2 stops. From such a desublimation process, the released heat Q is calculated as,

$$Q = m_r a_r h_r, \tag{11}$$

with h_r being the enthalpy change of CO₂ desublimation and a_r being the specific surface area per unit volume. Considering that CO₂ desublimates into solid through either the homogeneous nucleation within the contact liquid or the heterogeneous solidification on solid CO₂ surfaces, the mass transfer rate m_r is calculated as (Debnath et al., 2019; Lei et al., 2023; Yang et al., 2023),

$$m_r = \begin{cases} k_n \exp\left(\Delta G/k_B T\right), & \text{in the contact liquid,} \\ k_r(y_i p - p_e), & \text{on solid CO}_2 \text{ surfaces.} \end{cases}$$
(12)

Here, k_n and k_r are mass desublimation rate constants, ΔG is the Gibbs free energy, k_B is the Boltzmann constant, and y_i is the CO₂ mole fraction. The gas/liquid–solid equilibrium pressure is estimated as (Tuinier et al., 2010),

$$p_e = \exp\left(10.257 - \frac{3082.7}{T} + 4.08\ln T - 2.2658 \times 10^{-2}T\right).$$
 (13)

Based on the mass transfer rate in (12), the nucleation source term is $S_n = Y m_r$. During CO₂ desublimation, the structure of solid CO₂ is tracked by the mass balance equation (Kang et al., 2014),

$$\rho_s \partial_t V_s = m_r a_r V_r, \tag{14}$$

where V_s and ρ_s represent the volume and density of solid CO₂, respectively, and V_r is the active volume for desublimation.

To solve the above governing equations, boundary conditions at both internal and external boundaries of the computational domain are introduced. On the one hand, external boundary conditions are set as follows. First, from the inlet (y = 0), the flue gas is injected into the domain at a given operating condition. Accordingly, the gas compositions, temperature, pressure, and velocity are set as specified values. Then, at the outlet ($y = l_y$), a fully developed flow is considered and the zero-gradient velocity and the no-flux temperature and mass fraction are applied. Finally, the periodic conditions are imposed at the lateral sides (x = 0, l_x and z = 0, l_z). On the other hand, the desublimation of CO₂ takes place on solid CO₂ surfaces (I). Such internal boundary conditions are described by,

No-slip velocity:
$$\boldsymbol{u}^{I} = (0, 0),$$
 (15)

Mass conservation:
$$\mathbf{n}^{I} \cdot D\rho \nabla Y^{I} = m_{r},$$
 (16)

Conjugate heat transfer:

$$\begin{cases} T^{I,+} = T^{I,-}, \\ \mathbf{n}^{I} \cdot \left(k\nabla T + \rho c_{p} \mathbf{u} T\right)^{I,+} = \mathbf{n}^{I} \cdot \left(k\nabla T + \rho c_{p} \mathbf{u} T\right)^{I,-} + q. \end{cases}$$
(17)

In the above equations, n^{I} is the surface normal pointing to the gas or liquid phase, + and - denote parameters on either side of I, $k = \alpha \rho c_{p}$ is

the thermal conductivity, and *q* is the heat flux caused by CO_2 desublimation. Once the local temperature increases to the freezing point of solid CO_2 , the desublimation of CO_2 stops with $m_r = 0$ and q = 0. Thus, the no-slip and no-flux boundary conditions are implemented.

In order to model CO_2 desublimation using the LB method, the above physical parameters should be converted to those in lattice units. For this purpose, dimensionless parameters are derived to act as the conversion criterions between the two systems of units. By introducing the characteristic length *L*, velocity *U*, and temperature T_{ch} , dimensionless parameters marked by asterisks are derived as,

$$\mathbf{x}^{*} = \frac{u}{L}, t^{*} = \frac{t}{L/U}, \ \mathbf{u}^{*} = \frac{u}{U}, \ \rho^{*} = \frac{\rho}{\rho_{g}}, \ p^{*} = \frac{p}{\rho_{g}U^{2}}, \ T^{*} = \frac{T}{T_{ch}}, \\ m_{r}^{*} = \frac{m_{r}}{\rho_{g}U}, \ h_{r}^{*} = \frac{h_{r}}{c_{\rho,g}T_{ch}}, \ k_{r}^{*} = k_{r}U, \ \text{Re} = \frac{LU}{v_{g}}, \ \text{Pe} = \frac{LU}{D_{g}}, \ \text{Pr} = \frac{v}{a_{g}}.$$
(18)

The subscript g refers to physical properties of the flue gas. From such a dimensionless derivation, key characteristic numbers are obtained: the Reynolds number Re, the Péclet number Pe, and the Prandtl number Pr. In the following LB simulations, a match of these dimensionless variables ensures the same desublimation properties between the lattice space and the real physical coordinate.

3. Numerical method

A multiphysics LB model is built to solve the conservation Eqs. (1)–(5). Considering the varying thermophysical properties of gas, liquid, and solid phases, the energy conservation Eq. (5) is recast as (Lei et al., 2021),

$$\partial_t T + \nabla \cdot (T \boldsymbol{u}) = \nabla \cdot (\alpha \nabla T) + F_t,$$

$$F_t = \frac{Q}{\rho c_p} + \frac{\nabla (\rho c_p) \cdot (\alpha \nabla T - T \boldsymbol{u})}{\rho c_p} - \frac{T \partial_t (\rho c_p)}{\rho c_p},$$
(19)

with F_t being the new source term.

To solve the two-phase flow (Eqs. (1)-(3)), species transport (Eq. (4)), and heat transfer (Eq. (19)), four sets of LB evolution equations are built as follows (Guo and Shu, 2013; Lei et al., 2023; Liang et al., 2018; Yang et al., 2023),

$$f_i(\mathbf{x} + \mathbf{e}_i \delta_i, t + \delta_i) - f_i(\mathbf{x}, t)$$

= $-(\mathbf{M}^{-1} S \mathbf{M})_{ij} \Big[f_j(\mathbf{x}, t) - f_j^{eq}(\mathbf{x}, t) \Big] + \delta_i F_i,$ (20)

$$p_i \left(\mathbf{x} + \mathbf{e}_i \delta_t, \ t + \delta_t \right) - p_i (\mathbf{x}, \ t)$$

= $- \left(\mathbf{M}^{-1} \mathbf{S}_{\phi} \mathbf{M} \right)_{ij} \left[p_j (\mathbf{x}, \ t) - p_j^{eq} \ (\mathbf{x}, \ t) \right] + \delta_t F_{p,i},$ (21)

$$g_i(\mathbf{x} + \mathbf{e}_i \delta_t, t + \delta_t) - g_i(\mathbf{x}, t)$$

= $-(\mathbf{M}^{-1} \mathbf{S}_y \mathbf{M})_{ij} [g_j(\mathbf{x}, t) - g_j^{eq}(\mathbf{x}, t)] + \delta_t F_{y,i},$ (22)

$$h_i \left(\mathbf{x} + \mathbf{e}_i \delta_i, \ t + \delta_i \right) - h_i (\mathbf{x}, \ t)$$

= $- \left(\mathbf{M}^{-1} \mathbf{S}_t \mathbf{M} \right)_{ij} \left[h_j (\mathbf{x}, \ t) - h_j^{eq} (\mathbf{x}, \ t) \right] + \delta_t F_{t,i},$ (23)

where *i* and *j* are discrete directions. For fluid moving with discrete velocity e_i at position **x** and time *t*, $f_i(\mathbf{x}, t)$, $p_i(\mathbf{x}, t)$, $g_i(\mathbf{x}, t)$, and $h_i(\mathbf{x}, t)$ are distribution functions for the hydrodynamics, phase indicator, CO_2 mass fraction, and temperature fields, respectively. The equilibrium distribution functions are given as Lei et al. (2023), Liang et al. (2018),

$$r_{i}^{eq} = \begin{cases} \frac{p}{c_{s}^{2}} (w_{i} - 1) + \rho s_{i}, & i = 0, \\ \frac{p}{c_{s}^{2}} w_{i} + \rho s_{i}, & i \neq 0, \end{cases}$$

with $s_{i} = w_{i} \left(\frac{e_{i} \cdot u}{c_{s}^{2}} + \frac{(e_{i} \cdot u)^{2}}{2c_{s}^{4}} - \frac{u^{2}}{2c_{s}^{2}} \right),$ (24)

$$p_i^{eq} = w_i \phi \left(1 + \frac{\boldsymbol{e}_i \cdot \boldsymbol{u}}{c_s^2} \right), \tag{25}$$

$$g_{i}^{eq} = w_{i}Y\left[1 + \frac{e_{i} \cdot u}{c_{s}^{2}} + \frac{\left(e_{i} \cdot u\right)^{2}}{2c_{s}^{4}} - \frac{u^{2}}{2c_{s}^{2}}\right],$$
(26)

$$h_{i}^{eq} = w_{i}T\left[1 + \frac{e_{i} \cdot u}{c_{s}^{2}} + \frac{(e_{i} \cdot u)^{2}}{2c_{s}^{4}} - \frac{u^{2}}{2c_{s}^{2}}\right].$$
(27)

Here, $c_s = e/\sqrt{3}$ is the lattice sound velocity and *e* is the lattice speed. *S*, S_{ϕ} , S_{y} , and S_t are diagonal relaxation matrices. *M* is the transformation matrix to map distribution functions from the physical space to the moment space. The distribution functions for source terms are (Lei et al., 2023; Liang et al., 2018; Yang et al., 2022),

$$F_{i} = w_{i} \left(1 - \frac{1}{2\tau} \right) \left[\boldsymbol{u} \cdot \nabla \rho + \frac{\boldsymbol{e}_{i} \cdot \left(\boldsymbol{F}_{s} + \boldsymbol{F}_{g} \right)}{c_{s}^{2}} + \frac{\boldsymbol{u} \nabla \rho : \left(\boldsymbol{e}_{i} \boldsymbol{e}_{i} - c_{s}^{2} \boldsymbol{I} \right)}{c_{s}^{2}} \right], \quad (28)$$

$$F_{p,i} = w_i \left(1 - \frac{1}{2\tau_p} \right) \frac{\boldsymbol{e}_i \cdot \left[\partial_l (\boldsymbol{\phi} \boldsymbol{u}) + c_s^2 \lambda \boldsymbol{n} \right]}{c_s^2}, \tag{29}$$

$$F_{y,i} = w_i \left(1 - \frac{1}{2\tau_y} \right) \boldsymbol{e}_i \cdot \boldsymbol{\Phi}_{CST} + w_i \boldsymbol{S}_n, \tag{30}$$

$$F_{t,i} = w_i F_t, \tag{31}$$

with τ , τ_p , and τ_y being relaxation times. At each time step, after the above evolutions, the macroscopic variables are calculated as (Lei et al., 2023; Liang et al., 2018),

$$p = \frac{c_s^2}{1 - w_0} \left[\sum_{i \neq 0} f_i + \frac{\delta t}{2} \boldsymbol{u} \cdot \nabla \rho + \rho s_0 \right], \quad \rho \boldsymbol{u} = \sum_i \boldsymbol{e}_i f_i + \frac{\delta t}{2} \left(\boldsymbol{F}_s + \boldsymbol{F}_g \right), \quad (32)$$

$$\phi = \sum_i p_i, \quad \boldsymbol{Y} = \sum_i g_i, \quad \boldsymbol{T} = \sum_i h_i.$$

To enforce the CO₂ desublimation on solid CO₂ surfaces *I*, LB boundary schemes are developed. The no-slip velocity in Eq. (15) is addressed by the halfway bounce-back scheme and the conjugate heat transfer in Eq. (17) is implemented by solving the energy conservation Eq. (19). As for the mass conservation condition in Eq. (16), the surface mass fraction gradient ∇Y^I is calculated based on the finite-difference scheme as (Zhang et al., 2012),

$$\boldsymbol{n}^{I} \cdot \nabla \boldsymbol{Y}^{I} = \frac{\boldsymbol{Y} - \boldsymbol{Y}^{I}}{0.5\boldsymbol{n} \cdot \boldsymbol{e}_{i}\delta_{x}},\tag{33}$$

where *Y* is the CO_2 mass fraction at the gas or liquid grid neighboring *I*. By inserting Eq. (33) into Eq. (16) and using the ideal gas law, the value of Y^I is calculate as,

$$Y^{I} = \frac{D\rho Y + 0.5\mathbf{n} \cdot \mathbf{e}_{i}\delta_{x}k_{r}p_{e}}{D\rho + 0.5\mathbf{n} \cdot \mathbf{e}_{i}\delta_{x}k_{r}p_{0}T/T_{0}}.$$
(34)

The mass conservation boundary is thus deduced into a boundary with a given mass fraction Y^I , implemented using the halfway bounce-back scheme (Lei et al., 2023).

Furthermore, the VOP method is adopted to treat the evolution of solid CO₂ in Eq. (14) (Kang et al., 2006; Wang et al., 2019). Explicitly, a fine enough mesh is applied to cover the computational domain. Thus, each grid represents only one material: solid CO₂, flue gas, or contact liquid. Each grid node is located at the center of a unit volume. Initially, the volume of solid CO₂ is set as $V_s = 0$ for gas and liquid grids. As CO₂ desublimation happens, the value of V_s is updated at each time step by,

$$V_s(t+\delta_t) = V_s(t) + m_r a_r V_r / \rho_s.$$
(35)

As V_s increases to $V_s = 1$ at a liquid or gas grid, this grid transforms into a solid CO₂ grid. Then, when V_s doubles to $V_s = 2$ at a solid CO₂ grid, one of its neighboring gas or liquid grids expands into a solid CO₂ grid. After model validations, the developed multiphysics LB model is applied to investigate CO₂ desublimation within a contact liquid at the Table 1

Parameters	for po	re-scale	simulations	of	CO_2	desublima-
tion in a con	ntact li	quid.				

Physical properties	Values		
Contact liquid temperature, T_l	[135, 160] K		
Flue gas temperature, T_0	[165, 250] K		
CO_2 mass fraction, Y_0	[0.1, 1.0]		
Flue gas injection velocity, Pe	[1.4, 14]		
CO_2 diffusion coefficient in gas, D_g	$1.63 \times 10^{-5} \text{ m}^2/\text{s}$		
CO_2 diffusion coefficient in liquid, D_1	$2.91 \times 10^{-9} \text{ m}^2/\text{s}$		
Flue gas density, ρ_{g}	1.46kg/m ³		
Flue gas specific heat capacity, $c_{p,g}$	0.846 kJ/kgK		
Flue gas thermal diffusivity, α_{g}	$5.02 \times 10^{-6} \text{ m}^2/\text{s}$		
Flue gas kinematic viscosity, v_g	$1.01 \times 10^{-5} \text{ m}^2/\text{s}$		
Contact liquid density, ρ_l	620 kg/m ³		
Contact liquid specific heat capacity, $c_{p,l}$	2.25 kJ/kgK		
Contact liquid thermal diffusivity, α_l	$9.10 \times 10^{-8} \text{ m}^2/\text{s}$		
Contact liquid kinematic viscosity, v_l	$3.55 \times 10^{-7} \text{ m}^2/\text{s}$		
Solid CO ₂ density, ρ_s	$1.56 \times 10^3 \text{ kg/m}^3$		
Solid CO ₂ specific heat capacity, $c_{p,s}$	0.967 kJ/kgK		
Solid CO ₂ thermal diffusivity, α_s	$4.64 \times 10^{-7} \text{ m}^2/\text{s}$		
Mass desublimation rate constant, k_n	10 ⁻⁶ kgm/s		
Mass desublimation rate constant, k_r	10 ⁻⁶ s/m		
Enthalpy change of desublimation, h_r	$5.682 \times 10^5 \text{ J/kg}$		

pore scale. More details about the proposed LB model are provided in Appendices Appendix A–Appendix B.

Compared to our recent model for simulating CO_2 desublimation on solid surfaces (Lei et al., 2023), the current multiphysics LB model extends to include the contact liquid. The mathematical and numerical models developed here are advanced by incorporating two-phase flow, both homogeneous and heterogeneous CO_2 desublimation, CO_2 transport between air and liquid phases, and temperature transfer across all three phases. These newly integrated physical processes cover the limitations of our recent model, thereby allowing for a comprehensive treatment of the multiphysics involved in liquid-based CCC.

4. Results and discussion

For the desublimation system in Fig. 1, key geometrical parameters are set as: length $l_x = 43$ cm, height $l_y = 50$ cm, width $l_y = 43$ cm, and central hole diameter $l_d = 2$ cm. Initially, the system is filled by a contact liquid at T_l and the incompressible flue gas at (T_0, Y_0, u_0, p_0) is injected from the inlet. Considering that T_l is lower than the freezing point of CO₂, the CO₂ component desublimates and remains within the liquid for collection. In the subsequent simulations, desublimation parameters and thermalphysical properties are set as listed in Table 1. These physical parameters can be converted into lattice units based on a match of dimensionless parameters in Eq. (18), where characteristic parameters are,

$$L = l_{v}, \quad U = u_{0}, \quad T_{ch} = 175 \text{ K.}$$
 (36)

Prior studies have emphasized the importance of the liquid temperature T_1 , gas temperature T_0 , CO₂ mass fraction Y_0 , and gas injection velocity u_0 to the carbon capture performance of CCC (Fazlollahi et al., 2017; Jensen, 2015; Lei et al., 2023). Therefore, operating conditions are evaluated for ranges of these parameters as listed in Table 1. Note that, the gas injection velocity u_0 is characterized by the dimensionless Péclet number Pe. The contact liquid is set as isopentane in the present study and can be substituted with other suitable cryogenic liquids by adjusting the corresponding physical properties in future work. In this study, a mesh of size $256 \times 300 \times 256$ is selected to describe the 3D desublimation domain in Fig. 1. From this 3D domain, an xy cross-section at $z = 0.5l_z$ is selected for 2D simulations and the mesh size is 256×300 . Each simulation test continues until the CO₂ content in the outflow flue gas reaches the termination condition of $Y = 0.1Y_0$. The duration of the desublimation process until the termination point is termed as the operation time period t_e .



Fig. 2. CO_2 desublimation in a contact liquid at $T_l = 145$ K, $T_0 = 175$ K, $Y_0 = 1.0$, and Pe = 1.4. Contours of (a) solid CO_2 and (b) CO_2 mass fraction *Y* at six time instants t = 12.2, 47.2, 82.6, 122.0, 161.3, 175.0 min.

4.1. CO₂ Desublimation properties

The initial objective is to investigate CO_2 desublimation properties in a contact liquid. A 2D test with the liquid temperature $T_l = 145$ K, flue gas temperature $T_0 = 175$ K, CO_2 mass fraction $Y_0 = 1.0$, and Péclet number Pe = 1.4 (or gas injection velocity) is simulated. The obtained evolutions of solid CO_2 and CO_2 mass fraction Y over time are illustrated in Fig. 2.

The growth and distribution of solid CO₂ in a contact liquid are displayed over time in Fig. 2(a). At 12.2 min, small solid CO₂ nuclei are visible. This occurs because, after injection, the CO₂ component from flue gas diffuses across the gas-liquid interfaces and enters the contact liquid, where the homogeneous CO2 desublimation takes place and produces solid CO₂ nuclei. These nuclei gradually grow and show a pronounced presence of solid CO2 particles by 47.2 min. After that, the solid CO₂ regions further expand and become prominent, indicating the continued CO₂ desublimation process. Finally, from 161.3 min to 175.0 min, the expansion of solid CO₂ areas slows down, highlighting the decreased CO₂ desublimation rate over the period. Fig. 2(b) depicts contours of CO_2 mass fraction (Y) at varying time instants, where the gas-liquid interfaces are indicated by dashed lines. At 12.2 min, Y is uniformly low, indicating that the injected CO₂ component efficiently diffuses into the contact liquid and undergoes desublimation at this early stage. By 82.2 min, there is a noticeable increase in Y around the solid CO₂ areas. This stems from the fact that the continuous CO₂ desublimation increases the local temperature and reduces the available sites for further desublimation. As a result, near the solid CO_2 areas, the CO_2 desublimation rate slows down and the consumption of the CO₂ component diminishes. Correspondingly, the diffusion of CO₂ from the gas phase to the liquid phase becomes degraded and Y within the gas phase is amplified in those areas. Following the continuous decrease in the CO₂ desublimation rate, Y steadily increases and spreads, particularly within the gas phases around the solid CO₂ regions.

For comparison, the above case is repeated with CO_2 desublimation turned off. Distributions of solid CO_2 and CO_2 mass fraction *Y* at six time instants are illustrated in Fig. 3. As expected, no solid CO_2 is generated in this case due to the absence of CO_2 desublimation. Meanwhile, Fig. 3(b) shows that *Y* is consistently high in the gas phase but significantly lower in the liquid phase. This is attributed to the slow CO_2 diffusion from the flue gas to the contact liquid as well as the CO_2 mass fraction drop at the gas–liquid interfaces. These results differ significantly from those in Fig. 2, demonstrating that the existence of CO_2 desublimation can effectively capture CO_2 from the flue gas. To quantify the CO₂ capture capacity of the contact liquid, the volume fraction of solid CO₂ captured (ϕ_c) is calculated by,

$$\phi_c = \frac{1}{l_x l_y} \int_0^{l_x} \int_0^{l_y} V_s(x, y) dy dx \times 100\%.$$
(37)

Furthermore, considering the crucial role of monitoring the termination of the desublimation process, the CO₂ content in the outgoing gas phase (η_o) is estimated as,

$$\eta_o = \frac{1}{l_{x,g}Y_0} \int_0^{l_{x,g}} Y(x, l_y) dx \times 100\%,$$
(38)

where $l_{x,g}$ is the length occupied by the flue gas at the outlet. As introduced above, each simulation test stops once η_o reaches 10%. The calculated values of ϕ_c and η_o are plotted against time in Fig. 4(a)–(b). These plots are explained using the averaged temperature (\overline{T}) and the overall desublimation rate (m_r^*), as shown in Fig. 4(c)–(d). Here \overline{T} is defined as the averaged temperature over the entire computational domain,

$$\overline{T} = \frac{1}{l_x l_y} \int_0^{l_x} \int_0^{l_y} T(x, y) dy dx.$$
(39)

The overall desublimation rate m_r^* is calculated as,

$$m_r^* = \frac{1}{\rho_g u_0} \int_0^{l_x} \int_0^{l_y} m_r(x, y) dy dx.$$
(40)

In Fig. 4(a), the volume fraction of solid CO₂ captured (ϕ_c) is plotted over time. The black line represents the scenario with CO₂ desublimation, showing a steady increase in the value of ϕ_c . This indicates the continuous CO₂ desublimation process within the contact liquid. Insets within the plot illustrate the morphological evolutions of solid CO₂ captured. The growth rate of ϕ_c is observed to decline over time, which corresponds to the decreasing desublimation rate m_r^* in Fig. 4(d). This decline is driven by the rising temperature \overline{T} in Fig. 4(c), which slows down the CO₂ desublimation intensity (m_r^*) as described by Eq. (40). Fig. 4(b) depicts the outgoing CO₂ content in the flue gas (η_o). The case with CO₂ desublimation exhibits a relatively low level of η_o (i.e., $\eta_o < 10\%$), suggesting the effective capture of the CO_2 component. In this scenario, η_o increases over time and its growth rate accelerates as time progresses. The rise in T and the decline in m_r^* are responsible for such an increasing trend in η_o . As the value of η_o increases to reach 10%, marked by the grey dash line in Fig. 4(b), the simulation arrives at its termination point. At this point, the maximum amount of solid CO2 captured and the operation time period are recorded as ϕ_{cm} and t_e , respectively.



Fig. 3. CO_2 transport in a contact liquid without desublimation at $T_i = 145$ K, $T_0 = 175$ K, $Y_0 = 1.0$, and Pe = 1.4. Contours of (a) solid CO₂ and (b) CO₂ mass fraction *Y* at six time instants *t* = 1.2, 2.8, 12.0, 36.6, 55.1, 83.8 min.

Fig. 4. Analyses of CO₂ desublimation in a contact liquid at $T_l = 145$ K, $T_0 = 175$ K, $Y_0 = 1.0$, and Pe = 1.4. Temporal evolutions of (a) volume fraction of solid CO₂ captured ϕ_c , (b) outgoing CO₂ content in the gas phase η_o , (c) averaged temperature \overline{T} , and (d) overall desublimation rate m_r^* .

For comparison, the scenario without CO_2 desublimation is represented by red lines in Fig. 4. In this case, ϕ_c remains zero and η_o stays at a high level (i.e. $\eta_o > 85\%$). It suggests that the injected CO_2 remains in the gas phase and exits the desublimation system without being captured as solid CO_2 . Regarding the averaged temperature \overline{T} shown in Fig. 4(c), it increases at a slower rate in the scenario without CO_2 desublimation compared to that with desublimation. This is because the heat released from CO_2 desublimation contributes to warming up the desublimation system. The comparison in Fig. 4 highlights the fact that CO_2 diffusion from the gas phase to the liquid phase is inadequate for flue gas purification. In contrast, CO_2 desublimation within the contact liquid can effectively capture the injected CO_2 component from the flue gas. Taken together, these quantitative analyses corroborate the above qualitative observations in Figs. 2–3 and provide insights into the CO_2 desublimation process within a contact liquid.

4.2. Comparison of 2D and 3D simulations

In the above 2D simulations, the generated solid CO_2 particles form porous structures within the contact liquid. Such structures potentially increase the active solid CO_2 surfaces for heterogeneous CO_2 desublimation but reduce the available liquid sites for homogeneous CO_2 nucleation. Additionally, the presence of solid CO_2 particles obstruct the movement of flue gas and hinder the transport of CO_2 component from the gas phase to the liquid phase, thereby impacting the CO_2 component available for desublimation. The complexities and effects of these porous solid CO_2 structures are expected to vary between 2D and 3D scenarios. Therefore, to verify the accuracy of 2D simulations, it is essential to investigate CO_2 desublimation within a contact liquid in three dimensions. The 3D simulation setup, shown in Fig. 1, mirrors the parameters used in Section 4.1 to ensure comparability.

Fig. 5 presents the results of 3D simulations of CO₂ desublimation in a contact liquid, conducted at the liquid temperature $T_l = 145$ K, flue gas temperature $T_0 = 175$ K, CO₂ mass fraction $Y_0 = 1.0$, and Péclet number Pe = 1.4 (or gas injection velocity). Fig. 5(a) displays 3D contours of solid CO₂ at various time instants, illustrating the nucleation and growth of solid CO₂ within the contact liquid. Initially, small solid CO₂ nuclei appear and progressively grow and merge into large porous structures, with the growth rate decreasing over time. An *xy*-plane view of the solid CO₂ distribution at $z = 0.5I_z$ is provided in Fig. 5(b). This perspective clearly depicts the progression from small and discrete solid CO₂ nuclei to large and interconnected solid CO₂ regions within the contact liquid. Fig. 5(c) illustrates distributions of the CO₂ mass fraction (*Y*) in the same *xy*-plane. At the early stage, the value of *Y* in the gas phase decreases significantly due to the desublimation of CO₂. Over time, however, it gradually increases since the CO₂ desublimation rate declines.

Temporal evolutions of the solid CO₂ captured (ϕ_c) and the outgoing CO₂ content in the gas phase (η_o) from 3D simulations are recorded in Fig. 6, which also includes results from 2D simulations for comparison. Note that, due to the 3D structure and the central hole, both ϕ_c and η_o



Fig. 5. 3D simulations of CO₂ desublimation in a contact liquid at $T_i = 145$ K, $T_0 = 175$ K, $Y_0 = 1.0$, and Pe = 1.4. Contours of (a) solid CO₂, (b) *xy*-plane view of solid CO₂ at $0.5l_z$, and (c) *xy*-plane view of CO₂ mass fraction *Y* at $0.5l_z$, for six time instants *t* = 11.5, 55.3, 111.7, 172.0, 193.6, 211.3 min.

Fig. 6. Analyses of CO₂ desublimation in a contact liquid at $T_l = 145$ K, $T_0 = 175$ K, $Y_0 = 1.0$, and Pe = 1.4. Temporal evolutions of (a) volume fraction of solid CO₂ captured ϕ_c and (b) outgoing CO₂ content in the gas phase η_o .

in the 3D case are calculated as the average values of xy-planes ranging from $0.4l_z$ to $0.6l_z$. Fig. 6(a) shows that ϕ_c increases over time in both 2D and 3D simulations, implying the continuous CO₂ desublimation process. The 3D case consistently displays a higher ϕ_c compared to the 2D and thus the 3D model reproduces the more extensive desublimation of CO_2 in the contact liquid. This suggests that the additional z dimension allows for a more realistic representation of the CO₂ desublimation sites and their interactions, leading to a more effective description of the solid CO₂ generation in the contact liquid. The temporal evolution of η_0 is provided in Fig. 6(b), demonstrating an increasing trend over time in both 2D and 3D simulations. Additionally, the 3D case continuously exhibits a lower η_0 compared to the 2D one, which underscores the enhanced CO₂ capture efficiency through desublimation in three dimensions. This improved capture efficiency is attributed to the more realistic description of the complex solid CO₂ structures, species transport, and CO₂ desublimation processes in the 3D model.

Overall, compared to 2D studies, 3D simulations reveal the more intensive CO_2 desublimation dynamics and the higher CO_2 capture efficiency, stemming from the more realistic depiction of the porous CO_2 structures in three dimensions. Apart from these differences, CO_2 desublimation properties in the 3D case align well with the 2D results. It thus suggests that 2D simulations are sufficient for capturing the essential properties of CO_2 desublimation in a contact liquid. Moreover, 3D simulations are significantly more computationally demanding compared

to 2D studies. For example, to simulate CO₂ desublimation in a contact liquid until the termination point, a 2D simulation requires 2.4 hours of parallel computation using 1280 computing cores, whereas a 3D simulation necessities 46.7 hours of computation using 20,480 cores. Therefore, 2D simulations provide a simple and computationally efficient approach while still delivering accurate insights into CO₂ desublimation behaviors in a contact liquid. The subsequent simulations will be conducted in two dimensions.

4.3. Effects of operating conditions

Following the discussion of CO₂ desublimation properties in a contact liquid, a parametric study is set out to examine the impacts of operating conditions, specifically the liquid temperature T_l , flue gas temperature T_0 , CO₂ mass fraction Y_0 , and gas injection velocity (or Pe), on the CO₂ capture performance. The CO₂ desublimation process is simulated across a range of these operating parameters as listed in Table 1. From simulations in Section 4.1, two key metrics emerge to quantify the CO₂ capture performance: the maximum volume fraction of solid CO₂ capture (ϕ_{cm}) and the operation time period for CO₂ desublimation (t_e) . Using these two metrics, the CO₂ capture velocity is introduced as, $v_c = \phi_{cm}/t_e$. A higher v_c indicates a more efficient capture of CO₂, which is desirable.



Fig. 7. Analyses of CO₂ capture performance in a contact liquid at $T_l \in [135, 160]$ K, $T_0 =$ 175 K, $Y_0 = 1.0$, and Pe = 1.4. Temporal evolutions of (a) volume fraction of solid CO₂ captured ϕ_c and (b) outgoing CO₂ content in the gas phase η_o . (c) The maximum volume fraction of solid CO₂ captured ϕ_{cm} . (d) The operation time period for CO₂ desublimation t_e . (e) The CO₂ capture velocity v_c . (f) Distributions of solid CO₂ at t_e for simulations with $T_l = 145$ K and 135 K.

Fig. 7 illustrates the CO₂ capture performance in a contact liquid at various liquid temperatures T_l ranging from 135 K to 160 K, while keeping $T_0 = 175$ K, $Y_0 = 1.0$, and Pe = 1.4. Temporal evolutions of the volume fraction of solid CO₂ captured (ϕ_c) and the outgoing CO₂ content in the gas phase (η_o) are shown in Fig. 7(a)–(b). The simulation results exhibit a consistent CO_2 capture performance across all tested T_1 values: both ϕ_c and η_o increase over time as the desublimation process continues. Additionally, a lower T_l slightly speeds up the growth in ϕ_c and significantly slows down the rise in η_o . It is because a colder contact liquid increases the CO_2 desublimation rate according to Eq. (12), thereby accelerating the consumption of CO₂ in both the liquid and gas phases. Under the fixed flues gas injection condition (i.e., fixed Y_0 and Pe), this subsequently leaves less CO₂ in the flue gas (η_o). Note that, regardless of the enhanced CO_2 desublimation rate, a lower T_0 speeds up the increase in ϕ_c only to a limited extent. This is because the injected CO_2 for desublimation is restricted by the given Y_0 and Pe. Based on profiles of ϕ_c and η_o in Fig. 7(a)–(b), the termination point (ϕ_{cm} , t_e) at $\eta_o = 10$ % is determined for each tested T_l . Results of ϕ_{cm} and t_e are plotted against T_l in Fig. 7(c)–(d). A clear trend stands out in these figures that, as T_l drops from 160 K to 135 K, both ϕ_{cm} and t_e increase obviously, albeit at a reduced rate. This trend is expected because a colder contact liquid decelerates the growth in η_0 and thereby extends the CO₂ desublimation duration (t_e) . Along with the intensified CO₂ desublimation rate, the low- T_l scenarios ultimately result in the amplified ϕ_{cm} and t_e .

To quantify the correlation between ϕ_{cm} and t_e , the CO₂ capture velocity $v_c = \phi_{cm}/t_e$ is calculated and illustrated in Fig. 7(e). The profile exhibits the successive growth in v_c following the descending T_l , which highlights the improved CO₂ capture efficiency. However, as T_l declines to an extremely low level, the increase in v_c becomes less pronounced. For example, the decrease in T_l from 145 K to 140 K yields the significant increase in v_c from 0.351 to 0.403, but further drops to 135 K only brings about a slight growth in v_c from 0.403 to 0.408. A likely factor is that, although the theoretical desublimate rate increases with the decreasing T_l , the CO₂ supply for desublimation remains nearly constant at the given Pe and Y_0 . Correspondingly, at an extremely low T_l level (i.e., 135 K), the injected CO₂ becomes insufficient to satisfy the intensive desublimate rate, causing the system to transition from a CO₂-rich regime to a

CO₂-limited regime. Another important factor is the fast CO₂ desublimation rate under low- T_l scenarios, which introduces the rapid generation of solid CO₂ and the quick formation of highly porous solid structures within the contact liquid. As illustrated in Fig. 7(f), such porous structures may obstruct the flue gas flow and block the CO₂ desublimation sites. These factors contribute to the insufficient CO₂ desublimation and the slow increase in v_c under low- T_l conditions.

The CO₂ capture performance in a contact liquid is then examined for different flue gas temperatures T_0 ranging from 165 K to 250 K with fixed $T_l = 145$ K, $Y_0 = 1.0$, and Pe = 1.4. The simulation results are presented in Fig. 8, including volume fraction of solid CO₂ captured (ϕ_e), outgoing CO₂ content in the gas phase (η_o), the maximum volume fraction of solid CO₂ captured (ϕ_{em}), the operating time for CO₂ desublimation (t_e), and the CO₂ capture velocity (v_e).

Temporal evolutions of ϕ_c and η_o in Fig. 8(a)–(b) show that both ϕ_c and η_o increase over time for all T_0 values. Additionally, as T_0 drops off, the increase in ϕ_c exhibits only slight changes. It is caused by the restricted availability of CO_2 under the fixed Y_0 and Pe conditions. In contrast, the growth rate of η_o declines significantly with the descending T_0 . This can be explained as follows. The cold flue gas shortens the heat transfer period needed for the contact liquid to cool the flue gas from T_0 to the point where CO₂ desublimation begins, ultimately enhancing the CO₂ desublimation process. Combined with the quite stable CO₂ supply, the enhanced CO₂ desublimation process intensifies the CO₂ consumption and thereby yields the declined increase rate of η_o . Such effects of the low- T_0 conditions generally align with those of low- T_1 scenarios. By analyzing results of ϕ_c and η_o , the termination states (ϕ_{cm} , t_e) are identified for all tested T_0 values. The obtained ϕ_{cm} and t_e as a function of T_0 are illustrated in Fig. 8(c)–(d), revealing a clear trend that both ϕ_{cm} and t_e decrease as T_0 increases. This is because the cold flue gas intensifies the CO₂ desublimation process and degrades the CO₂ content in the flue gas, which ultimately boosts the generation of solid CO₂ (ϕ_{cm}) and prolongs the duration of CO_2 desublimation (t_e).

The calculated v_c is finally plotted against T_0 in Fig. 8(e), showing that T_0 has a substantial impact on the CO₂ capture efficiency. The decreasing T_0 yields the larger amount of solid CO₂ captured, the longer desublimation period, and the higher CO₂ capture velocity v_c . How-



Fig. 8. Analyses of CO₂ capture performance in a contact liquid at $T_1 = 145$ K, $T_0 \in [165, 250]$ K, $Y_0 = 1.0$, and Pe = 1.4. Temporal evolutions of (a) volume fraction of solid CO₂ captured ϕ_c and (b) outgoing CO₂ content in the gas phase η_o . (c) The maximum volume fraction of solid CO₂ captured ϕ_{cm} . (d) The operation time period for CO₂ desublimation t_e . (e) The CO₂ capture velocity v_c .

Fig. 9. Analyses of CO₂ capture performance in a contact liquid at $T_l = 145$ K, $T_0 = 175$ K, $Y_0 \in [0.1, 1.0]$, and Pe = 1.4. Temporal evolutions of (a) volume fraction of solid CO₂ captured ϕ_c and (b) outgoing CO₂ content in the gas phase η_o . (c) The maximum volume fraction of solid CO₂ captured ϕ_{cm} . (d) The operation time period for CO₂ desublimation t_e . (e) The CO₂ capture velocity v_c .

ever, as T_0 drops from 190 K to 165 K, the positive effect of T_0 becomes marginal, resulting in slight changes in v_c from 0.370 to 0.356. This trend stems from the fact that, although the cold flue gas can intensify the CO₂ desublimation process, the limited CO₂ supply from the injected flue gas restricts the continuous increase in v_c . The discussions in Figs. 7–8 highlight the importance of optimizing temperature levels of both the contact liquid T_l and the flue gas T_0 to enhance the CO₂ capture performance in a contact liquid. It is recommended to avoid excessively low T_l and T_0 values due to the high energy consumption of cooling and their minimal improvements on the CO₂ capture efficiency.

Fig. 9 illustrates the CO₂ capture performance in a contact liquid for varying CO₂ mass fractions Y_0 from 0.1 to 1.0, and fixed $T_l = 145$ K, $T_0 = 175$ K, and Pe = 1.4. The temporal evolutions of ϕ_c and η_o for different Y_0 values are illustrated in Fig. 9(a)–(b). The results exhibit a clear trend that the ascending Y_0 boosts the increase in ϕ_c and decelerates the rise in η_o . Consequently, the CO₂ desublimation process completes slowly and the amount of solid CO₂ captured rises rapidly. These substantial effects of Y_0 can be explained as follows. A large Y_0 amplifies the partial pressure $y_i p$ and the Gibbs free energy ΔG , thus speeding up the local CO₂ desublimation rate according to Eq. (12). This consequently contributes to the intensified CO₂ consumption (i.e., the diminished η_o) and the enhanced CO₂ nucleation and growth (i.e., the augmented ϕ_c). Different from the low- T_l or T_o conditions, the high- Y_0 scenarios can increase the CO₂ supply and thus accelerate the growth in ϕ_c obviously. After determining the termination states (ϕ_{cm} , t_e) by using results in Fig. 9(a)–(b), the evolutions of ϕ_{cm} , t_e , and v_c are plotted against Y_0 in Fig. 9(c)–(e). As Y_0 increases, there is a steady growth in all three metrics: ϕ_{cm} , t_e , and v_s . This tend corroborates the above explanation that the high- Y_0 conditions contribute to the improved CO₂ desublimation rate and the enhanced CO₂ capture efficiency.

Attention is finally turned to the influence of gas injection velocity (or Pe) on the CO₂ capture performance. The CO₂ desublimation process in a contact liquid is simulated for varying Pe \in [1.4, 14], and fixed T_l = 145 K, T_0 = 175 K, and Y_0 = 1.0. The modelled results are illustrated in Fig. 10, including evolutions of ϕ_c , η_o , ϕ_{cm} , t_e , and v_c .

Fig. 10 (a)–(b) plot the temporal evolutions of ϕ_c and η_o for different Pe numbers. The data indicates that the consecutive rise in Pe generally causes the rapid increase in ϕ_c . The sufficient supply of CO₂ is responsi-



Fig. 10. Analyses of CO₂ capture performance in a contact liquid at $T_1 = 145$ K, $T_0 = 175$ K, $Y_0 = 1.0$, and Pe $\in [1.4, 14]$. Temporal evolutions of (a) volume fraction of solid CO₂ captured ϕ_c and (b) outgoing CO₂ content in the gas phase η_o . (c) The maximum volume fraction of solid CO₂ captured ϕ_{cm} . (d) The operation time period for CO₂ desublimation t_e . (e) The CO₂ capture velocity v_c .

ble for this trend. To be specific, the fast gas injection associated with the high-Pe condition intensifies the gas supply and increases the gas phase fraction in the desublimation domain. This subsequently increases the gas-liquid interfaces for CO₂ transport into the liquid, amplifying the number of desublimation sites in the contact liquid. Meanwhile, the local CO₂ desublimation rate remains relatively stable due to the fixed temperature levels and the given CO_2 mass fraction. The combination of these two factors (i.e., the amplified desublimation sites and the stable desublimation rate) boosts the overall CO2 desublimation rate and accelerates the growth in ϕ_c . However, as Pe continues to ascend beyond an extremely high level (i.e., Pe = 14), the rise in ϕ_c begins to slow down. This occurs because the injected gas flows too fast to allow sufficient time for either the CO_2 transport into the liquid phase for homogeneous desublimation or the heterogeneous CO₂ desublimation on solid surfaces, thereby giving rise to the reduced growth in ϕ_c . On the other hand, a noticeable increase in η_o is detected with the growing Pe. Such an effect is driven by the fact that the barely changed local CO2 desublimation rate, combined with the significantly increased gas injection velocity, makes much CO₂ remaining in the outgoing flue gas. This subsequently allows the CO₂ desublimation process to complete quickly. The termination states (ϕ_{cm} , t_e , v_c) are plotted against Pe in Fig. 10(c)–(e). A clear trend stands out that both ϕ_{cm} and v_c initially increase with the ascending Pe, but they turn to decline after reaching their peaks at around Pe = 9.8. This pattern stems from the fact that, at an extremely high gas injection velocity (or Pe), the availability of CO2 in both the liquid phase or the solid surfaces becomes diminished and thereby the CO_2 desublimation efficiency is reduced. Differently, t_e declines progressively as Pe goes up, driven by the rapid rise in η_a under high-Pe conditions.

Based on findings in Figs. 9–10, it is suggested to appropriately select the CO_2 mass fraction Y_0 and the gas injection velocity (or Pe) to optimize the CO_2 capture efficiency of CCC. The CCC system is expected to perform effectively while purifying flue gas with a high CO_2 content (Y_0). This is because a high Y_0 leads to the improved CO_2 desublimation rate and the optimized CO_2 capture efficiency. Meanwhile, a large Pe is favorable since it boosts the overall CO_2 desublimation rate and then improves the CO_2 capture performance. It is emphasized that, however, Pe should not exceed a certain threshold. Beyond this point, the benefits of a high Pe will be degraded due to the limited time for CO_2 transport into the liquid phase or on solid CO_2 surfaces for desublimation. An appropriate Pe is thus essential to balance the enhanced desublimation rate and the degraded CO_2 availability, ensuring the efficient CO_2 capture.

5. Conclusions

In this work, a multiphysics lattice Boltzmann (LB) model has been proposed and validated for pore-scale simulations of CO₂ desublimation in a contact liquid. As a continuation of our recent studies on CO₂ desublimation on solid surfaces (Lei et al., 2023), this work newly introduces the two-phase flow, species transport across the gas and liquid phases, and homogeneous CO₂ desublimation in the contact liquid. Based on the proposed LB model, CO₂ desublimation in a contact liquid is investigated for varying conditions (i.e., the liquid temperature T_l , flue gas temperature T_0 , CO₂ mass fraction Y_0 , and gas injection velocity (or Pe).

The CO₂ desublimation process is firstly obtained by twodimensional (2D) simulations. The formation of small solid CO2 nuclei within the contact liquid and the growth of these nuclei into large solid CO₂ particles are reproduced. Moreover, 3D simulations are carried out to investigate the CO₂ desublimation. The results align well with those from 2D studies, although they come at significantly higher computational costs. It thus confirms the reliability and cost-effectiveness of 2D simulations. After that, a parametric study is carried out to examine effects of operating conditions on the CO2 capture performance in a contact liquid. With the descending T_l or T_0 , there is an obvious growth in the three performance metrics, namely the maximum volume fraction of solid CO₂ captured (ϕ_{cm}), the operation time period (t_e), and the CO₂ capture velocity ($v_c = \phi_{cm}/t_e$). The intensified CO₂ desublimation rate under low-temperature conditions is responsible for this trend. However, when T_l or T_0 drops to extremely low levels, the increase in v_c is constrained by the limited CO₂ supply at the given CO₂ mass fraction Y_0 and gas injection velocity (or Pe). A high Y_0 is found to result in the improved CO₂ desublimation rate and the augmented values of ϕ_{cm} , t_e , and v_c , hence the optimized CO₂ capture efficiency. Following the continuous rise in Pe, t_e decreases consistently, while ϕ_{cm} and v_c increase at first but change to decline after peaking at a threshold. These trends are attributed to the fact that a large Pe can boost the overall CO2 desublimation rate and improve the CO₂ capture performance. However, such benefits diminish once Pe exceeds the threshold, as the extremely fast gas flow towards the outlet limits the transport of CO₂ to the liquid or solid CO₂ surfaces for desublimation. For the optimal CO₂ capture performance, it is recommended to purify the flue gas with high Y_0 and Pe, while maintaining low T_l and T_0 . Nevertheless, extremely low temperatures and excessively high gas injection velocities should be avoided due to the insufficient CO₂ availability in the liquid or on solid surfaces for desublimation.

In summary, the proposed LB model successfully reproduced the CO_2 desublimation processes in a contact liquid during CCC under various operating conditions. The present findings provide valuable insights into the underlying physics of the liquid-based CCC operations and enhance the understanding of optimal conditions. This study demonstrates the capability of LB modelling to facilitate the optimization of CCC, which is a promising technology for mitigating climate change.

Declaration of competing interest

The authors declare that they have no known competing financial interests or personal relationships that could have appeared to influence the work reported in this paper.

CRediT authorship contribution statement

Timan Lei: Data curation, Formal analysis, Investigation, Methodology, Software, Visualization, Validation, Writing – original draft. Geng Wang: Investigation, Methodology, Software, Writing – review & editing. Junyu Yang: Investigation, Methodology, Software, Writing – review & editing. Jin Chen: Writing – review & editing. Kai H. Luo: Conceptualization, Formal analysis, Funding acquisition, Investigation,

Project administration, Resources, Supervision, Writing – review & editing.

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Appendix A. Multiphysics LB model

In this study, the two-dimensional nine-velocity (D2Q9) and threedimensional nineteen-velocity (D3Q19) MRT LB models are developed for simulating CO₂ desublimation within a contact liquid in two and three dimensions, respectively. The discrete velocities e_i , weight coefficients w_i and the transformation matrix **M** are set as (Guo and Shu, 2013),

D2Q9:
$$e_i = e \begin{bmatrix} 0 & 1 & 0 & -1 & 0 & 1 & -1 & -1 & 1 \\ 0 & 0 & 1 & 0 & -1 & 1 & 1 & -1 & -1 \end{bmatrix}$$
, (A.1)

Here $e = \delta_x / \delta_t$ is the lattice speed and it is set as e = 1 in this study. δ_x and δ_t denote the lattice spacing and time step, respectively.



Fig. 11. Model validation of CO₂ desublimation in a contact liquid. Comparison of the outgoing CO₂ content in the gas phase η_o between the present numerical results and the experimental measurements by Jensen et al. (2015).

The transformation matrix M maps distribution functions from the physical space ψ to the moment space as $\hat{\psi} = M \cdot \psi$. With this transformation, the evolution Eqs. (20)–(23) are performed in the moment space as,

$$\hat{f}(\mathbf{x} + e_i \delta_t, t + \delta_t) = \hat{f}(\mathbf{x}, t) - S \Big[\hat{f}(\mathbf{x}, t) - \hat{f}^{eq}(\mathbf{x}, t) \Big] + \delta_t \hat{F},$$
(A.6)

$$\hat{p}(\mathbf{x} + e_i \delta_t, t + \delta_t) = \hat{p}(\mathbf{x}, t) - S_{\phi} [\hat{p}(\mathbf{x}, t) - \hat{p}^{eq}(\mathbf{x}, t)] + \delta_t \hat{F}_p,$$
(A.7)

$$\hat{\boldsymbol{g}}\left(\boldsymbol{x}+\boldsymbol{e}_{i}\boldsymbol{\delta}_{t},\ t+\boldsymbol{\delta}_{t}\right)=\hat{\boldsymbol{g}}(\boldsymbol{x},\ t)-\boldsymbol{S}_{y}\left[\hat{\boldsymbol{g}}(\boldsymbol{x},\ t)-\hat{\boldsymbol{g}}^{eq}\left(\boldsymbol{x},\ t\right)\right]+\boldsymbol{\delta}_{t}\hat{\boldsymbol{F}}_{y},\tag{A.8}$$

$$\hat{\boldsymbol{h}}(\boldsymbol{x} + \boldsymbol{e}_i \delta_t, \ t + \delta_t) = \hat{\boldsymbol{h}}(\boldsymbol{x}, \ t) - \boldsymbol{S}_t \Big[\hat{\boldsymbol{h}}(\boldsymbol{x}, \ t) - \hat{\boldsymbol{h}}^{eq}(\boldsymbol{x}, \ t) \Big] + \delta_t \hat{\boldsymbol{F}}_t.$$
(A.9)

Through the Chapman-Enskog analysis on the proposed LB equations, the governing equations can be recovered with the relaxation times τ , τ_{ϕ} , τ_{y} , and τ_{t} being,

$$v = c_s^2(\tau - 0.5)\delta_t, M = c_s^2(\tau_\phi - 0.5)\delta_t, D = c_s^2(\tau_y - 0.5)\delta_t, \alpha = c_s^2(\tau_t - 0.5)\delta_t,$$
(A.10)

as well as the gradient terms of temperature being (Lei et al., 2017; 2021),

$$\nabla_x T = -\frac{\hat{h}_3 - Tu}{c_s^2 \tau_l \delta_l}, \qquad \nabla_y T = -\frac{\hat{h}_5 - Tv}{c_s^2 \tau_l \delta_l}.$$
 (A.11)

Except these calculations, other gradient terms and Laplace operates in Section 3 are calculated by the isotropic central scheme as (Guo et al., 2011),

$$\nabla \varsigma = \sum_{i} \frac{w_{i} \boldsymbol{e}_{i} \varsigma \left(\boldsymbol{x} + \boldsymbol{e}_{i} \delta_{t}\right)}{c_{s}^{2} \delta_{t}}, \qquad \nabla^{2} \varsigma = \sum_{i} \frac{2w_{i} \varsigma \left(\boldsymbol{x} + \boldsymbol{e}_{i} \delta_{t}\right) - \varsigma(\boldsymbol{x})}{c_{s}^{2} \delta_{t}^{2}}.$$
 (A.12)

Appendix B. Model validation

We have recently developed LB models for simulating CO_2 desublimation in a packed bed during CCC (Lei et al., 2023; 2024). Benchmark problems with analytical or experimental solutions have been conducted to test key sub-models of this proposed LB models. For brevity, these validation details are not repeated here. Building on our prior studies, this research extends the LB models to include two-phase flow, species transport across phases, and homogeneous desublimation, thereby allowing simulations of CO_2 desublimation in a contact liquid. To validate the reliability of the newly proposed LB model, we simulated the CO_2 desublimation process based on the desublimation domain in Fig. 1.

Our simulations are conducted as described in Section 3 of Jensen et al. (2015), with the same desublimation conditions applied. Specifically, the flue gas temperature $T_0 = 145$ K, CO₂ mass fraction $Y_0 = 0.15$, and the gas pressure $p_0 = 1$ atm. The thermophysical properties are set as in Section 4. The outgoing CO₂ content in the flue gas (η_o) is calculated and plotted against time in Fig. 11, where the experimental data from Jensen et al. (2015) is included for comparison. As shown,

the experimental data exhibits several peaks that are not present in our simulations. According to Jensen et al. (2015), these peaks are resulted from changes in operating conditions during experiments. Despite these discrepancies, the proposed LB model successfully replicates the overall CO_2 capture performance observed in experiments, demonstrating its accuracy in simulating CO_2 desublimation in a contact liquid.

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