

# Free surface flow over square bars at different Reynolds numbers

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## Abstract

Large-eddy simulations of free surface flow over bed-mounted square bars are performed for laminar, transitional and turbulent flows at constant Froude number. Two different bar spacings are selected corresponding to transitional and  $k$ -type (reattaching flow) roughness, respectively. The turbulent flow simulations are validated with experimental data and convincing agreement between simulation and measurement is obtained in terms of water surface elevations and streamwise velocity profiles. The water surface deforms in response to the underlying bed roughness ranging from mild undulation for transitional roughness to distinct standing waves for  $k$ -type roughness. The instantaneous water surface deformations increase with an increase in Reynolds number. Contours of the mean streamwise and wall-normal velocities, the total shear stress and the streamfunction reveal the presence and extension of recirculation zones in the trough between two consecutive bars. The flow is governed by strong local velocity gradients as a result of the rough bed and the deformed water surface. The local Froude number at the free surface increases for low Reynolds number in the flow over transitional roughness and decreases for low Reynolds number in the flow over  $k$ -type roughness. The transitional and turbulent flows exhibit a very similar distribution of the pressure coefficient  $C_p$  in both cases, whilst  $C_p$

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is generally lower for the laminar flow. Regarding the friction coefficient,  $C_f$ , it is significantly lower in the turbulent case than in the transitional and laminar cases. The bar spacing does not affect significantly the relative contribution of friction and pressure forces to the total force, neither does the Reynolds number. The friction factor is greater for transitional roughness and decreases with increasing Reynolds number.

*Keywords:* Free surface flow, large-eddy simulation, rough-bed flow, turbulent flow, transitional flow, laminar flow.

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## 1. Introduction

Shallow flows over rough beds are ubiquitous in nature, in particular in relatively steep channels or when traveling at high Froude numbers. These flows are characterised by significant deformation of the water surface, localised standing waves or even hydraulic jumps. The nature of deformation of the water surface of shallow flows over rough beds is influenced by several parameters including the Froude and Reynolds number, the channel bed slope as well as the bed's roughness topography. The cause of water surface deformations is the bulk flow and its turbulence structure. Scrutinizing the water surface deformation as a result of bulk flow characteristics and turbulence structures in response to the channel's rough bed can lead to a detailed understanding of shallow free surface flow hydrodynamics and hence to advanced non-intrusive river gauging methods or aerial flow measurements.

The geometrical characteristics of the bed roughness manipulates the flow's quantities and their distributions such as the velocity profile and the total drag. Here, a train of two-dimensional square bars, placed on a bed perpendicular to the flow at a given bar spacing, comprises the roughness. The main two types of this roughness, known as  $d$ -type (closely spaced bars) and  $k$ -type (widely spaced bars), were first identified by Perry et al. (1969). They summarised that the effective roughness, which is used to characterise the drag increment due to the roughness (Jiminez, 2004), is proportional to the integral length scale of

22 the flow for the  $d$ -type roughness and to the roughness height for the  $k$ -type  
23 roughness. Thus, at a given bulk Reynolds number, the transition between  $d$ -  
24 and  $k$ -type roughness over two-dimensional roughness elements depends only  
25 on the streamwise spacing of the bars. The mean and turbulent flow charac-  
26 teristics over square bar roughness have been investigated both experimentally  
27 (Okamoto, Seo, Nakaso and Kawai, 1993; Djenidi, Elavarasan and Antonia,  
28 1999; Krogstad, Andersson, Bakken and Ashrafian, 2005; Djenidi, Antonia,  
29 Amielh and Anselmet, 2008; Roussinova and Balachandar, 1969) and numer-  
30 ically (Cui, Patel and Lin, 2003; Stoesser and Rodi, 2004; Ikeda and Durbin,  
31 2007; Stoesser and Nikora, 2008; McSherry, Chua and Stoesser, 2017). Although  
32 this roughness geometry is rather simple, it has been helpful in studying the flow  
33 features over rough surfaces such as the transition from  $d$ - to  $k$ -type roughness  
34 which occurs at around  $\lambda/k = 4 - 5$ , where  $\lambda$  is the crest-to-crest bar spacing  
35 and  $k$  is the bar height (Simpson, 1973; Tani, 1987; Jiminez, 2004). Although  
36 many of the works mentioned focused on single-phase flows (closed channels),  
37 some investigated free surface flows over bars where the submergence plays a  
38 key role in the formation of the flow structure. In flows with large submergence,  
39 the entire cavity between roughness elements is occupied by a stable vortex for  
40  $d$ -type roughness; however, for  $k$ -type roughness the flow reattaches to the bed  
41 after a wake region between two bars (Stoesser and Rodi, 2004). Stoesser and  
42 Nikora (2008) used large-eddy simulation (LES) for free surface flow with in-  
43 termediate submergence and showed that for  $\lambda/k > 8$  the bars are isolated and  
44 the flow reattaches to the bed between two square bars. This was also shown  
45 in single-phase flows (Leonardi, Orlandi, Smalley, Djenidi and Antonia, 2003).  
46 Experiments of the flow with different submergence for two  $k$ -type spacings ( $\lambda/k$   
47 = 9 and 18) revealed that for the larger spacing the roughness effect extends  
48 up to a distance of  $3k$  from the bed while for smaller spacing the outer layer is  
49 affected by the roughness too. In flows with low submergence, the free surface  
50 experience changes due to the roughness especially in the form of standing wave  
51 or hydraulic jump at the free surface. These surface modulations are mainly  
52 generated over  $k$ -type roughness. Large scale turbulent structures, strong tur-

53 bulence production and energy dissipation contribute to these deformation at  
54 the free surface (Chanson and Brattberg, 2000; Chanson, 2009). McSherry et al.  
55 (2018) carried out several LES and laboratory experiments to study the hydro-  
56 dynamics of shallow flows and quantified the effects of roughness spacing and  
57 relative submergence on hydraulic resistance in such flows. Six flow cases, with  
58 transitional (between  $d$ - and  $k$ -type) and  $k$ -type roughness types and varying  
59 relative submergence were investigated. Flow features such as free surface de-  
60 formation, double averaged velocities and shear stresses and the contributions  
61 to the overall momentum balance were presented.

62 The research reported here investigates shallow flows over a transitional and  
63 a  $k$ -type roughness at three Reynolds numbers resulting in laminar, transitional  
64 and turbulent flow at the same Froude number. The objective of the present  
65 work is to investigate the effect of Reynolds number on the free surface char-  
66 acteristics, the underlying hydrodynamics, the distribution of the fluid stress  
67 and the relative contributions of pressure and friction drag to the total drag of  
68 shallow flows over bar-roughened beds. This paper is organized in four sections.  
69 Section 2 describes the applied numerical method and details of the simulations,  
70 section 3 presents and discusses the results. The manuscript is wrapped up by  
71 a summary and conclusion in section 4.

## 72 **2. Numerical framework and simulations performed**

73 Large-eddy simulation (LES) is employed in this research, a method per-  
74 ceived to be particularly suitable to simulate flows which are affected by large-  
75 scale turbulence structures (Stoesser, 2014). Due to the eddy-resolving na-  
76 ture of this approach energetic large-scale structures of the flow are calcu-  
77 lated directly and the effect of the small scales on the large scales is mod-  
78 elled. Another time-resolved computational method, unsteady Reynolds-aver-  
79 aged Navier-Stokes (URANS), is deemed unsuitable for this due its inability to  
80 predict accurately and reliably flow separation and reattachment, an important  
81 feature of the flows under consideration. Direct numerical simulation (DNS)

82 is the only approach which resolves all scales of turbulence but it is compu-  
 83 tationally uneconomical and requires enormous computational resources. The  
 84 in-house LES code HYDRO3D is employed to simulate the flows reported in  
 85 this study. It has been validated thoroughly for a large number of flows of  
 86 similar complexity (Stoesser, 2010; Bomminayuni and Stoesser, 2011; Stoesser,  
 87 McSherry and Fraga, 2015; Fraga, Stoesser, Lai and Socolofsky, 2016b; Fraga  
 88 and Stoesser, 2016a; Ouro, Harrold, Stoesser and Bromley, 2017a; Ouro and  
 89 Stoesser, 2017b). The code solves the spatially filtered Navier-Stokes equations

$$\nabla \cdot \mathbf{u} = 0 \quad (1)$$

$$\frac{\partial \mathbf{u}}{\partial t} + \mathbf{u} \cdot \nabla \mathbf{u} = -\nabla p + \frac{1}{Re} \nabla^2 \mathbf{u} - \nabla \cdot \tau \quad (2)$$

90 for turbulent, incompressible, three-dimensional flow. In equations (1) and (2)  
 91  $\mathbf{u}$  is the velocity vector with the three components  $u$ ,  $v$ ,  $w$  in the streamwise  
 92 ( $x$ ), spanwise ( $y$ ) and wall-normal ( $z$ ) directions,  $p$  is the pressure,  $\tau$  is the  
 93 subgrid scale stress tensor,  $Re = U_b H / \nu$  is the Reynolds number where  $U_b$  is  
 94 the bulk velocity,  $H$  is the depth defined as the distance between the mean water  
 95 surface position before starting the simulation and the mean bed elevation (see  
 96 Fig. 1), and  $\nu$  is the fluid kinematic viscosity. The effects of the small-scale  
 97 turbulence on the large eddies is approximated using the Wall-Adapting Local  
 98 Eddy-viscosity (WALE) (Nicoud and Ducros, 1999) model. There are several  
 99 advantages of using this model compared to the classical Smagorinsky model  
 100 such as detecting all the turbulent structures relevant to kinetic energy dissi-  
 101 pation and/or no need of a damping function or constant adjustments near the  
 102 wall as the eddy-viscosity goes naturally to zero in that region. A fractional-step  
 103 method with a second order Runge-Kutta time integration scheme is used to  
 104 solve equations (1) and (2) on a staggered Cartesian grid. In the predictor step  
 105 a second order finite difference method is used to compute diffusive terms and  
 106 a fifth-order weighted, essentially non-oscillatory (WENO) scheme is used to  
 107 compute the convective terms (Shu, 2009). The WENO scheme offers the nec-  
 108 essary compromise between numerical accuracy and algorithm stability which

109 is especially important for the free-surface algorithm (Kara, Kara, Stoesser and  
 110 Sturm, 2015b). A multi-grid method is employed in the corrector step to solve  
 111 the pressure Poisson equation and to achieve a divergent flow field at the end  
 112 of each time step. The accuracy and credibility of LES results are more sensi-  
 113 tive to the treatment of boundary conditions, also called super-grid modeling,  
 114 and use of high-order spatial discretization schemes together with sufficiently  
 115 fine grids (Stoesser, 2014; Rodi, Constantinescu and Stoesser, 2013). Validating  
 116 the present results with similar experiments (Fig. 2) shows that the numeri-  
 117 cal implementation of the Navier-Stokes equations are accurate enough for the  
 118 applied grid resolution. The Level Set Method (LSM), proven to be success-  
 119 ful in the description of complex air-water interfaces, is used for free surface  
 120 capturing (Sussman, Smereka and Osher, 1994; Kang and Sotiropoulos, 2012;  
 121 Kara, Stoesser, Sturm and Mulahasan, 2015c; Kara, Kara, Stoesser and Sturm,  
 122 2015b). A level set signed distance function,  $\phi$ , which is zero at the phase  
 123 interface, negative in air and positive in water is employed. The interface is  
 124 tracked by solving a pure advection equation (Sethian and Smereka, 2003)

$$\frac{\partial \phi}{\partial t} + \mathbf{u} \cdot \nabla \phi = 0 \quad (3)$$

A transition zone is introduced at either side of the phase interface to avoid  
 numerical instability. This instability may be caused due to sudden change  
 and discontinuity of density and viscosity of the two immiscible fluids. In the  
 transition zone, with the thickness of two grid spacings,  $\phi$  is calculated as  $|\phi| \leq$   
 $\varepsilon$  where  $\varepsilon$  is half of the thickness of the interface. The density  $\rho$  and dynamic  
 viscosity  $\mu$  are calculated as

$$\rho(\phi) = \rho_g + (\rho_l - \rho_g)\mathcal{H}(\phi) \quad (4)$$

$$\mu(\phi) = \mu_g + (\mu_l - \mu_g)\mathcal{H}(\phi) \quad (5)$$

125 where  $\mathcal{H}(\phi)$  is a Heaviside defined as  $\mathcal{H}(\phi) = 0$  for  $\phi < -\varepsilon$ ,  $\mathcal{H}(\phi) = 1/2(1$   
 126  $+ \phi/\varepsilon + \sin(\pi\phi/\varepsilon)/\pi)$  for  $|\phi| < \varepsilon$  and  $\mathcal{H}(\phi) = 1$  for  $\phi > \varepsilon$ . Equation (3) is

127 solved using the WENO scheme as this is a pure advection problem and us-  
128 ing central differencing schemes cause stability problem (Rodi, Constantinescu  
129 and Stoesser, 2013). The solution domain is decomposed into a certain number  
130 of sub-domains and the code solves the equations in each domain separately  
131 via parallel computing. The Message Passing Interface (MPI) is used to allow  
132 communication between sub-domains. More details of the code can be found in  
133 (Cevheri, McSherry and Stoesser, 2016; Ouro, Fraga, Lopez-Novoa and Stoesser,  
134 2018; Kara, Stoesser and McSherry, 2015a)

135 Large-eddy simulations of the flow over two-dimensional square bars of height  
136 ( $k$ ) at two different bar spacings,  $\lambda/k = 5.2$  and  $\lambda/k = 10.4$ , where  $\lambda$  is the dis-  
137 tance from bar to bar, are carried out at three different  $Re$ , corresponding to  
138 laminar, transitional and turbulent flow are performed. Table 1 provides an  
139 overview of the various simulations as well as geometrical and hydraulic pa-  
140 rameters for each case. The  $\lambda/k = 5.2$  case is neither  $d$ -type (skimming flow)  
141 nor  $k$ -type (reattaching flow) roughness so it is considered transitional (wake  
142 interference flow) roughness, while the  $\lambda/k = 10.4$  case is classified as  $k$ -type  
143 roughness. The global Froude number,  $Fr = U_b/\sqrt{gH}$ , is kept constant for all  
144 cases ( $g$  is the gravity acceleration). The two turbulent flows are carried out at  
145 two different grid resolutions (medium and fine) and also double-sized domain  
146 (double the length and width in streamwise and spanwise directions, respec-  
147 tively) to investigate the sensitivity of the simulation results to grid resolution  
148 and domain size. Fig. 1 presents the schematic of the computational domains.  
149 For both bars spacings the length of the domain,  $L_x$ , is  $10.4k$ , the width,  $L_y$   
150 , and height (including both water and air),  $L_z$ , are both  $5k$  and hence for  
151 the  $\lambda/k = 5.2$  case the domain included two troughs while for  $\lambda/k = 10.4$  it  
152 only contained one trough. For the double-sized domain the streamwise and the  
153 spanwise extent of the domain are doubled without changing the grid resolution  
154 and for the fine grid the grid numbers are doubled in all directions on the reg-  
155 ular-sized domain. In all cases the grid spacing is uniform in all directions. The  
156 geometrical configurations and flow variables are generally similar to the exper-  
157 iments and simulations of McSherry et al. (2018) while some details regarding

158 boundary conditions and flow simulations differs. A no-slip boundary condition  
 159 is applied at the bottom and at the surface of the bars. Periodic boundary con-  
 160 dition are applied in the streamwise and spanwise directions to ensure that the  
 161 flow is quasi two-dimensional (i.e. homogeneous in the spanwise direction) and  
 162 without secondary currents (McSherry et al., 2018). The turbulent and tran-  
 163 sitional flows simulations are driven by a constant pressure gradient of similar  
 164 value to the flume experiments discussed in McSherry et al. (2018) while the  
 165 laminar simulations are driven by an adjusted pressure gradient to maintain a  
 166 constant mass flow rate to ensure that  $U_b$  is similar to the turbulent cases. The  
 167 friction velocity, as listed in Table. 1, used to calculate the grid spacings in  
 168 wall units, is calculated from  $U_\tau = (dp/dx)(h/\rho)$  where  $dp/dx$  is the pressure  
 169 gradient in the streamwise direction and  $h$  is the mean water surface elevation  
 170 from the bed at still water level (see Fig. 1). All simulations are initiated with a  
 171 free-slip boundary condition at the still water level and the simulations are run  
 172 for 10 to 15 flow through periods,  $T_f (= L_x/U_b)$ , to allow for the flow to develop  
 173 fully. The simulations are then restarted with the level set algorithm to track the  
 174 free surface. Averaging of the flow quantities is begun after another 4 to 6 flow  
 175 through periods when the free-surface flow is fully developed, and continued for  
 176 between 40 and 60 further flow through periods to obtain converged turbulence  
 177 statistics. The differences in  $Fr$  and  $Re$  between the work of McSherry et al.  
 178 (2018) and the present simulations for turbulent flows is due to uncertainties  
 179 in the experiments. However, the difference in  $Fr$  or  $Re$  is less than 10% or  
 180 5%, respectively for the turbulent flows suggesting adequacy of grid, domain  
 181 size and boundary conditions. In this following, time-averaged quantities are  
 182 denoted with an overbar, the double- (temporal- and spatial-) averaged quanti-  
 183 ties are denoted by both the overbar and brackets  $\langle \rangle$ , the small symbols with  
 184 prime are fluctuations and small symbols without any of these are instantaneous  
 185 quantities.



### 186 3. Results and discussion

187 The credibility of the simulations is assessed first by comparing computed  
188 two-dimensional water surface elevation and the temporal mean streamwise ve-  
189 locity of the two turbulent cases with data reported in McSherry et al. (2018).  
190 Fig. 2 (a,b) present time- and spanwise-averaged water surface elevations for all  
191 grid spacings and domain sizes together with experimental and numerical data  
192 (case C2 and C5) in McSherry et al. (2018). There is a good agreement between  
193 the present LES results and the reference data. Fig. 2 (c,d) show the adequacy  
194 of the domain size and grid spacing chosen for the turbulent case as well. Fig.  
195 3 and Fig. 4 show profiles of the time- and spanwise-averaged streamwise ve-  
196 locity together with the corresponding data reported in McSherry et al. (2018)  
197 from both experiments and LES. There were two measurement locations for  
198  $\lambda/k = 5.2$  and four for  $\lambda/k = 10.4$  (Fig. 1). Both figures show good agreement  
199 of the present results with McSherry et al. (2018)'s data. The results of the  
200 simulations on the fine grid and with the double domain (not shown here) are  
201 similar to the results obtained on the chosen domain and grid size and are hence  
202 not worth presenting. The simulation outputs in terms of turbulence statistics  
203 are not sensitive to the grid resolution and domain size. The grid resolution of  
204 the transitional  $Re$  is identical to the grid resolution of the turbulent cases while  
205 the the laminar cases were carried out on a coarser grid, however due to the  
206 low Reynolds number the grid is effectively finer than for the other two cases.  
207 Table 1 demonstrates the adequacy of the grid size for all simulated cases.

#### 208 3.1. Water surface

209 The time- and spanwise-averaged water surface profile is plotted in Fig. 5 for  
210 all  $Re$  simulated. There was no considerable difference in water surface between  
211 the three flow cases over the  $\lambda/k = 5.2$  bars except for some weak undulation  
212 between two consecutive bars in the turbulent and transitional flows, whilst the  
213 free surface of the laminar flow is almost flat. Over the  $\lambda/k = 10.4$  bars the  
214 general feature of the water surface is similar for all  $Re$ , exhibiting a standing

215 wave between the bars. Increasing the  $Re$  results in the standing wave becoming  
 216 steeper with the trough of the wave deeper. Fig. 6 shows the instantaneous three  
 217 dimensional free surface of all six flows considered highlighting the differences  
 218 in water surface behaviour of the simulated cases. The main difference for both  
 219  $\lambda/k = 5.2$  and  $\lambda/k = 10.4$  is that the water surface is the more disturbed  
 220 the higher the  $Re$ . Instantaneous water surface disturbances are absent for the  
 221 laminar cases and featured only a very weak undulation and a small standing  
 222 wave for every two bars, Fig. 6(a) and (d) respectively. In the transitional  $Re$ ,  
 223 water surfaces are disturbed and the undulations over the  $\lambda/k = 5.2$  bars and the  
 224 standing wave over the  $\lambda/k = 10.4$  bars are distinguished more clearly. In the  
 225 turbulent flows, instantaneous water surface disturbances, the undulations and  
 226 standing waves are well-established for both roughness arrangements. Due to  
 227 the temporal- and spanwise-averaging instantaneous disturbances of the water  
 228 surface are not seen in Fig. 5. The main features of the water surfaces are  
 229 similar for all  $Re$  and for both  $\lambda/k = 5.2$  and  $\lambda/k = 10.4$ , by increasing  $Re$  the  
 230 instantaneous perturbations of the water surface are enhanced and modulate  
 231 the time-averaged water surfaces.

### 232 3.2. Mean velocity contours and streamfunction

233 The velocity and stress distributions are examined to provide a more detailed  
 234 understanding of the flow fields and momentum transport. Fig. 7 presents con-  
 235 tours of the time- and spanwise-averaged streamwise velocity normalized by the  
 236 bulk velocity for all six flow cases. Also plotted is the local  $Fr$  calculated at  
 237 the free surface. In all cases the flow separates at the bars leading to the gen-  
 238 eration of recirculation bubbles just downstream of the bars. The recirculation  
 239 zones include significant negative velocity near the bed in the transitional and  
 240 turbulent flows over both bars spacing. In the flow over the  $\lambda/k = 5.2$  bars this  
 241 recirculation bubble occupied the entire trough between two consecutive bars  
 242 while over  $\lambda/k = 10.4$  this zone extends to a reattachment point after which the  
 243 flow recovers to a boundary layer. The recirculation zone is the longest in the  
 244 transitional flow and shortest in the laminar flow. **The large shear stress in the**

245 laminar flow leads to boundary layer formation near the wall earlier than two  
 246 other types of flow and this pushes the reattachment point back towards the  
 247 bars which generates smaller recirculation zone. In turbulent flow, where there  
 248 are the strongest disturbances, the intense mixing near the wall increase the  
 249 momentum exchange. This affects the weak negative streamwise velocity at the  
 250 end of the recirculation bubble hence contributes to a small recirculation zone.  
 251 In the transitional flow though, the shear stress and the turbulent motions are  
 252 not at their maximum strength thus the negative streamwise velocity induced by  
 253 the bar extends to a larger area and makes the recirculation zone the longest.  
 254 In the flow over  $\lambda/k = 5.2$  streamwise gradients of the streamwise velocity are  
 255 almost absent above the bars while they are quite significant above the bars  
 256 over  $\lambda/k = 10.4$ . The flow accelerates just above the bars near the free surface  
 257 leading to a local increase of  $Fr$  until it reaches a maximum, after that the flow  
 258 decelerates as the height of the recirculation bubble decreases relatively and  
 259 the water depth increases suddenly hence a decrease in the streamwise velocity  
 260 keeps a constant mass flow rate. This sudden deceleration is evidenced by the  
 261 collapse in the local  $Fr$ . Local acceleration followed by a sudden deceleration  
 262 results in the formation of the standing wave at the water surface (McSherry,  
 263 Chua, Stoesser and Mulahasan, 2018). Fig. 7a, b, c show that the variations  
 264 of the local  $Fr$  is significantly less in the  $\lambda/k = 5.2$ , and hence local variations  
 265 of the water depth for these cases are much smaller, resulting in the afore-  
 266 mentioned mild undulations. Noteworthy are high local  $Fr$  in the laminar case  
 267 over the  $\lambda/k = 5.2$  bars, which is due to the significant viscous stress, resulting  
 268 in a quasi-parabolic velocity profile and consequently high streamwise velocities  
 269 near the water surface.

270 The distribution of the total shear stress (viscous plus turbulent) signifies  
 271 momentum flux. Contours of the the sum of the time- and spanwise-averaged  
 272 shear stresses, viscous and Reynolds shear stress, normalized by the squared  
 273 (global) shear velocity are shown in Fig. 8. In the laminar flow over  $\lambda/k = 5.2$   
 274 the time-averaged shear stress is negative near the water surface, whereas in the  
 275 transitional and turbulent flows it is near zero almost everywhere. The ever-so-

276 slightly deformed water surface in this case creates local form drag and hence  
277 the flow decelerates very close to the water surface. The negative shear stress  
278 is the result of negative streamwise velocity gradients and this is particularly  
279 pronounced in the laminar flow in which the viscosity is very high. In the  
280 laminar flow over  $\lambda/k = 10.4$  the mean shear stress is negative just below  
281 the water surface as well and also in the transitional flow while it is positive  
282 upstream of the standing wave in the turbulent flow. The standing wave creates  
283 significant drag below the water surface, again leading to decelerated flow near  
284 the surface. In the laminar flow the extent of the negative shear stress is quite  
285 significant as there are large changes in the surface elevation especially around  
286 the standing wave. Due to high viscosity the effects of large water surface de-  
287 formations in this case extend to larger flow depth hence the negative shear  
288 stress in this case is weaker than that in the laminar flow over  $\lambda/k = 5.2$  as  
289 the boundary layer is thicker in the former than the latter. In the transitional  
290 flow the negative shear stress is only significant just downstream of the stand-  
291 ing wave as the viscous effects are still considerable whereas in the turbulent  
292 flow it is strongest only immediately downstream of the wave as larger distur-  
293 bances especially at the standing wave induce large disturbances in the flow  
294 field and contribute to larger turbulent shear stress. In the laminar flow in  
295 both geometries the maximum of the time- and space-averaged shear stress is  
296 at the crest of the bars where the velocity gradient is the largest but in the  
297 transitional and turbulent flows this maximum is downstream of the bars, due  
298 to only small contributions of the viscous stress and large contributions of the  
299 turbulent stress at higher  $Re$ . The effect of high viscosity is also observed near  
300 the bed. In the laminar flows pockets of the negative time- and space-averaged  
301 total shear stress are visible downstream of the bar (as a result of recirculation).  
302 In the transitional flows though, these pockets are visibly smaller, due to the  
303 viscous stress being less dominant and in the turbulent flows, where the viscous  
304 stress contribution is very small, they are reduced to a very thin layer adjacent  
305 to the wall in the recirculation bubbles.

306       Contours of the time- and spanwise-averaged wall-normal velocity normal-

307 ized by the bulk velocity are presented in Fig. 9 and they highlight significant  
 308 interaction of the near bed flow with the flow near the water surface, evidenced  
 309 by pronounced variations of  $\langle \bar{w} \rangle$  over the water depth and in the streamwise  
 310 direction. In the flows over  $\lambda/k = 5.2$ , alternating regions of positive and neg-  
 311 ative  $\langle \bar{w} \rangle$  are found above and below the crest of the bars. In the laminar  
 312 flow, under the crest of the bars, there are mainly patches of positive  $\langle \bar{w} \rangle$  in  
 313 the wake of the bars followed by negative  $\langle \bar{w} \rangle$ . The positive patches occupy  
 314 almost the entire cavity between bars in the transitional and turbulent flows  
 315 and the regions of negative  $\langle \bar{w} \rangle$  are small but strong near the leading edge of  
 316 the bars. Positive patches near the bed between the bars result in negative ones  
 317 above extending up to the water surface. At the leading edge of the bars there  
 318 is local increase in  $\langle \bar{w} \rangle$  resulting in regions of positive  $\langle \bar{w} \rangle$  above extending  
 319 to the water surface. Alternating regions of positive and negative time- and  
 320 spanwise-averaged wall-normal velocity are present in the flow over  $\lambda/k = 10.4$   
 321 too. The patches of negative  $\langle \bar{w} \rangle$  above the crest of the bars results in regions  
 322 of strong mean wall-normal velocity near the water surface and extending close  
 323 to the bed between bars in the larger-spacing-simulations. Positive regions of  
 324  $\langle \bar{w} \rangle$  near the leading edge of the bars extend all the way to the water sur-  
 325 face. A second peak of positive  $\langle \bar{w} \rangle$  is observed just below the standing wave.  
 326 Clearly, maxima and minima of  $\langle \bar{w} \rangle$  are greater in the flow over the  $\lambda/k = 10.4$   
 327 bars than over the  $\lambda/k = 5.2$  bars. Fig. 7 and Fig. 9 suggest that the abrupt  
 328 change of the water surface correlate with local acceleration/deceleration of the  
 329 flow. These two figures also show that in the flow over  $\lambda/k = 5.2$  under the  
 330 crest of the bars there is flow circulation between two bars while in the flow over  
 331  $\lambda/k = 10.4$  there is not such flow circulation.

332 Fig. 10 presents contour lines of the spanwise-mean streamfunction for all  
 333 simulated cases. The dashed lines denote negative values. Consistent with Fig.  
 334 7 these plots show that for small bar spacing the recirculation zone occupies the  
 335 cavity between bars while the mean flow reattaches at the bed for the wide bar  
 336 spacing beds. Small regions of positive streamfunction, a secondary recirculation  
 337 zone, driven by the primary recirculation, exists just downstream of the the

338 bars and they are of similar size in all transitional and turbulent cases.  
 339 No such secondary recirculation zone exists in the laminar flow as a result of  
 340 the primary recirculation being short and weak. There is a small region of  
 341 negative streamfunction at the leading edge of the bars in the transitional and  
 342 turbulent cases as the flow accelerates around the leading edge of the bars and  
 343 detaching from the bed. The streamwise velocity near the bed in the laminar  
 344 case is significantly smaller than in the other two flows and hence this upstream  
 345 bubble is absent in the laminar flow. The longest recirculation bubble is found  
 346 in the transitional flow due to the positive mean shear stress downstream of the  
 347 bar as shown in Fig. 8. In the turbulent flow the streamwise extension of the  
 348 recirculation zone is smaller due to the larger contribution of Reynolds shear  
 349 stress to the mean shear stress which results in the larger region of positive  
 350 mean shear stress downstream of the bar.

### 351 3.3. Pressure and friction forces

352 To investigate the interaction of the bulk flow with the bed, the main forces  
 353 applied on the bed were studied. Pressure coefficient,  $C_p$ , which is the ratio of  
 354 pressure difference over the dynamic pressure calculated by the bulk velocity  
 355 and friction coefficient,  $C_f$ , which is the wall shear stress normalized by the  
 356 dynamic pressure calculated by the bulk velocity were examined.

357 Fig. 11 plots  $C_p = (\langle \overline{P} \rangle - \langle \overline{P_{ref}} \rangle) / (0.5\rho U_b^2)$  where  $\langle \overline{P_{ref}} \rangle$  is the  
 358 temporal and spanwise mean pressure over the free surface at  $x = x_{ref}$  (Fig.  
 359 1). In both roughness spacings, the pressure has its maximum value at the  
 360 leading edge of the bars. In the transitional and turbulent flows there are no  
 361 changes in  $C_p$  downstream of the bars up to some certain streamwise locations  
 362 but it increased monotonously after this length with an abrupt sudden increase  
 363 at the leading edge of the next bar. However, the increasing trend is found  
 364 in the entire cavity for  $C_p$  in the laminar flows. In the flow over  $\lambda/k = 5.2$   
 365 the length over which the pressure is constant is longer in the transitional and  
 366 turbulent flows than that in those flows over  $\lambda/k = 10.4$ .  $C_p$  in the transitional  
 367 and turbulent flows over both roughness types have similar values with the

368 maximum  $C_p$  corresponding to the transitional flow. In the laminar flow,  $C_p$   
 369 is smaller while the difference of  $C_p$  between the laminar flow and two other  
 370 flow cases is greater in the flow over  $\lambda/k = 10.4$  than  $\lambda/k = 5.2$ . Since there is  
 371 not a large difference between the mean relative submergence for all  $Re$  in both  
 372 geometries (see Fig 5) and there is no dynamic pressure over the surface,  $C_p$  can  
 373 be interpreted as the static pressure over the bed. In the flow over  $\lambda/k = 5.2$   
 374 the values of the wall-normal velocity near the bed are smaller compared to the  
 375 value and peaks of  $\langle \bar{w} \rangle$  in the flow over  $\lambda/k = 10.4$  which are also closer to the  
 376 bed (Fig. 9). This contributes to the generation of a larger pressure coefficient  
 377 on the bed in the flow over  $\lambda/k = 10.4$  as seen in Fig. 11.

378 Fig. 12 presents the friction coefficient  $C_f = (\tau_{wall}) / (0.5\rho U_b^2)$  where  $\tau_{wall}$   
 379 is the wall shear stress. The sign of  $C_f$  for all cases is consistent with the mean  
 380 streamwise velocity distribution shown in Fig. 7. Here the friction coefficients  
 381 in the two transitional flows are of the same order as those of the laminar flow  
 382 cases. The negative peak of  $C_f$  in the transitional flow in Fig. 12(a) are further  
 383 from the the trailing edge of the bar on the left compared to the same peak  
 384 in the laminar flow due to the presence of the relatively larger region of the  
 385 positive streamwise velocity in the wake of that bar (Fig. 7(a,b)). Having a  
 386 smaller region of positive  $\langle \bar{u} \rangle$  in the turbulent flow in this case leads to the  
 387 displacement of this peak back towards the upstream bar. Similarly, in Fig.  
 388 12(b) the first negative peak ( $1.5 < x/k < 3$ ) is the furthest from the left bar  
 389 in the transitional flow as the region of the positive streamwise velocity is the  
 390 largest in this type of flow. The small recirculation region in  $9 < x/k < 10$  in  
 391 Fig. 7(d,f) leads to the generation of the second negative peak in Fig. 7(b)  
 392 in the transitional and turbulent flows which causes a decrease in the total  $C_f$   
 393 in this geometry. Since the viscosity is larger in the laminar flow, the largest  
 394 value of  $C_f$  is expected for both peaks in the laminar flow between the three  
 395 simulated  $Re$  which is the case in Fig. 12 (b). But in the flow over  $\lambda/k = 5.2$   
 396 the value of the negative peak is similar in the laminar and transitional flows  
 397 representing the similar maximum tangential velocity above the bed between  
 398 these two flow types.

399 The pressure and friction drag forces in the streamwise direction are shown  
400 in Fig. 13 a. These forces are normalized by the total force  $F_T = F_\tau + F_P$ .  
401 Friction drag is calculated at the bed and at the top of the bars and pressure  
402 drag is calculated at the front and back faces of the bars. Friction forces are  
403 entirely negative in flow over the  $\lambda/k = 5.2$  bars (as the recirculation zone  
404 occupies the entire through), irrespective of  $Re$ , and just positive in all flows  
405 over the  $\lambda/k = 10.4$  bars due to the recirculation zone being finite. In all  
406 cases, pressure drag is significantly larger than friction drag independent of the  
407 Reynolds number. Also, the bar spacing does not affect the distribution of the  
408 forces for the three  $Re$  investigated. Fig. 13 b plots friction factors (in the  
409 form of the Darcy Weisbach friction factor) as a function of Reynolds number  
410 for both roughness types. The friction factor of the flow over  $k$ -type roughness  
411 is consistently greater than for the flow over transitional roughness and this  
412 is in line with previous research. Also in line with previous research of flows  
413 over rough beds is that flow resistance decreases with an increase in Reynolds  
414 number.

#### 415 4. Conclusions

416 Results of large-eddy simulations of flows over spanwise aligned square bars  
417 in an open channel flow were presented. Two bar spacings corresponding to  
418 transitional and  $k$ -type roughness were considered each at three  $Re$  correspond-  
419 ing to laminar, transitional and turbulent flows at a constant  $Fr$ . The time- and  
420 spanwise-averaged water surface profiles and the instantaneous three-dimensional  
421 water surface visualisations showed that the mean water surface deformations  
422 are similar for all three  $Re$  except for the effects of turbulence which results  
423 in perturbations of the instantaneous water surface. In the flow over  $k$ -type  
424 roughness a standing wave is established at all  $Re$  at the water surface and it  
425 is steepest in the turbulent flow. The flows over the transitional roughness are  
426 characterised by mild water surface undulations at all  $Re$  with these being most  
427 pronounced for the turbulent flow. Significant streamwise gradients of the time-



428 averaged streamwise velocity, mainly occurring near the free surface and below  
429 the crest of the bars, are present in the flow over the  $k$ -type roughness. It was  
430 also revealed that the variations of the local  $Fr$  explained the formation of the  
431 standing waves, signifying a sudden drop in streamwise velocity near the water  
432 surface and hence a sudden deceleration of the flow leading to a sudden increase  
433 in water depth. The mean total shear stress is negative just below the water  
434 surface in the laminar flows over both bar spacing and its maximum value was  
435 in the flow over transitional roughness. This led to the rise of  $Fr$  to its largest  
436 value at the free surface in this flow type. The time-averaged wall-normal ve-  
437 locity varies significantly throughout the domain regardless of bar arrangement  
438 and Reynolds number. There is flow circulation below the crest of the bars over  
439 transitional roughness occupying the entire trough between bars. The largest  
440 recirculation bubble in the flow over  $k$ -type roughness was found in the transi-  
441 tional flow as the positive mean shear stress in the wake of the bars shifted the  
442 recirculation zone away from them. The larger contribution of Reynolds shear  
443 stress to the mean shear stress in the turbulent flow resulted in the decrease of  
444 the streamwise extension of the recirculation zone in this flow type. The max-  
445 imum value of pressure on the bed corresponds to the transitional flows and  
446 the minimum value to the laminar flows in flow over both bar spacings. The  
447 peak of the pressure in every flow case occurs at the leading edge of the bars.  
448 Larger wall-normal velocity in the flow over  $k$ -type roughness contributes to  
449 larger  $C_p$  in this geometry. Small regions of positive streamwise velocity shifts  
450 the peak of  $C_f$  in the streamwise direction. The tangential velocity adjacent  
451 to the wall is larger in the laminar flow over transitional roughness than  $k$ -type  
452 roughness, the cases in which the friction coefficients are maximum between all  
453 investigated  $Re$ . In the flow cases studied here, increasing the  $Re$  leads to a  
454 small increase in the pressure force applied on the bed and a small decrease in  
455 the viscous force while the bar spacing had even a smaller effect on the value of  
456 these forces. Generally pressure drag dominates this flow. In terms of friction  
457 factors, as expected  $k$ -type roughness features larger flow resistance and flow  
458 resistance generally decreases with an increase in Reynolds number.

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$\lambda/k = 5.2, H/k = 2.5$							
Case	$U_b$	$U_\tau$	$Re$	$Fr$	$\Delta x^+$	$\Delta y^+$	$\Delta z^+$
Turbulent	0.24	0.073	$7.2 \times 10^3$	0.44	58.3	68.3	36.4
Turbulent <sub>fine</sub>	0.25	0.073	$7.5 \times 10^3$	0.46	29.5	34.8	18.1
Turbulent <sub>double domain</sub>	0.25	0.073	$7.5 \times 10^3$	0.46	59.3	68.7	36.0
Transitional	0.23	0.073	$6.9 \times 10^2$	0.42	5.8	6.8	3.6
Laminar	0.24	0.081	$7.2 \times 10^1$	0.44	1.0	1.5	0.8
McSherry et al. (2018)	0.28	0.077	$8.3 \times 10^3$	0.51	75.1	71.8	38.8

$\lambda/k = 10.4, H/k = 2.9$							
Case	$U_b$	$U_\tau$	$Re$	$Fr$	$\Delta x^+$	$\Delta y^+$	$\Delta z^+$
Turbulent	0.23	0.074	$8.0 \times 10^3$	0.39	59.5	69.7	37.2
Turbulent <sub>fine</sub>	0.24	0.075	$8.3 \times 10^3$	0.41	30.2	35.4	18.9
Turbulent <sub>double domain</sub>	0.24	0.075	$8.3 \times 10^3$	0.41	60.4	70.7	37.7
Transitional	0.22	0.075	$7.7 \times 10^2$	0.38	5.9	7.0	3.7
Laminar	0.23	0.11	$8.0 \times 10^1$	0.39	1.3	2.0	1.1
McSherry et al. (2018)	0.24	0.083	$8.3 \times 10^3$	0.42	80.3	76.9	41.0

Table 1: Hydraulic conditions and computational details.

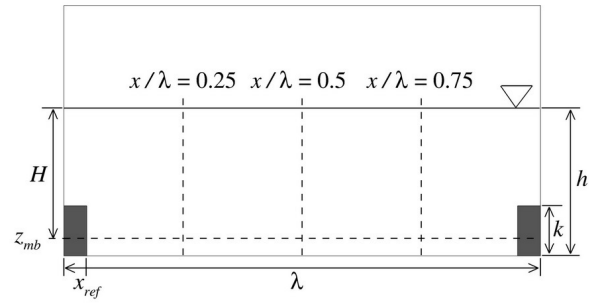


Figure 1: Schematic of the computational domain.

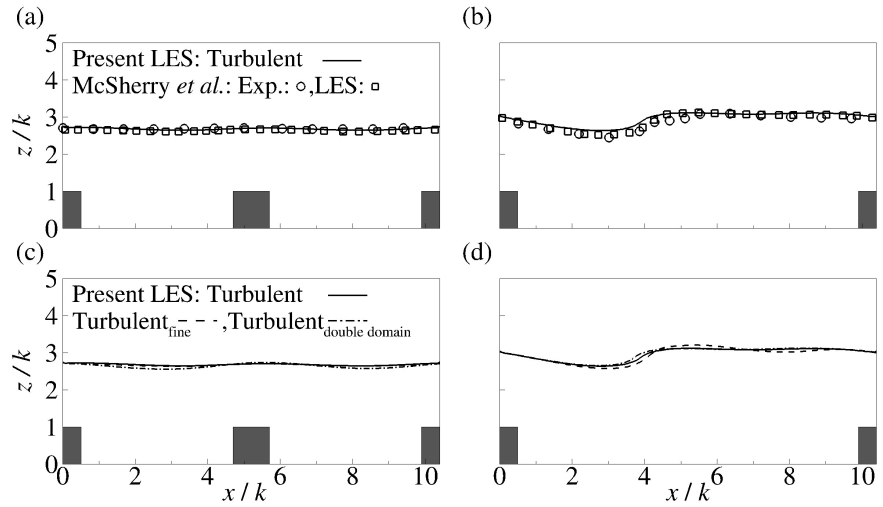


Figure 2: Time- and spanwise-averaged free surface elevations for (a,c)  $\lambda/k = 5.2$  and (b,d)  $\lambda/k = 10.4$ .



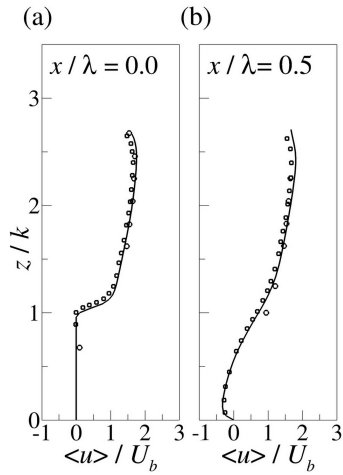


Figure 3: Temporal and spanwise mean streamwise velocity for  $\lambda/k = 5.2$  at selected streamwise locations.

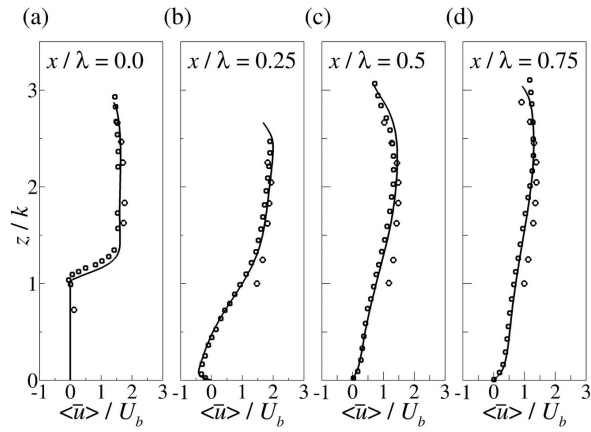


Figure 4: Time- and spanwise-averaged streamwise velocity for  $\lambda/k = 10.4$  at selected streamwise locations.

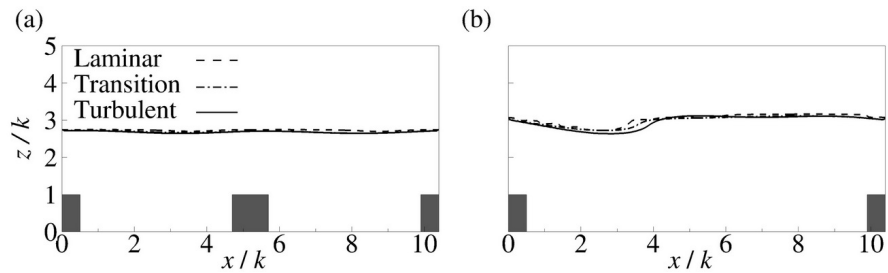


Figure 5: Time- and spanwise-averaged water surface profiles in turbulent, transitional and laminar flow for (a)  $\lambda/k = 5.2$  and (b)  $\lambda/k = 10.4$ .

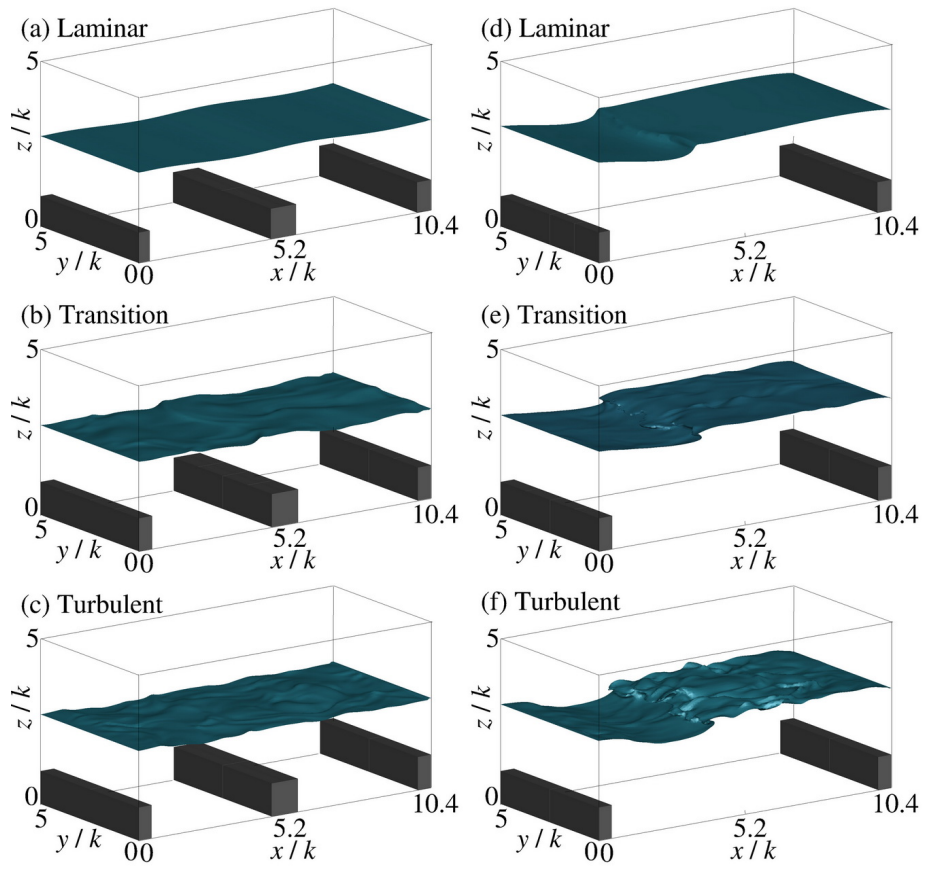


Figure 6: Instantaneous water surface in turbulent, transitional and laminar flow for (a-c)  $\lambda/k = 5.2$  and (d-f)  $\lambda/k = 10.4$ .

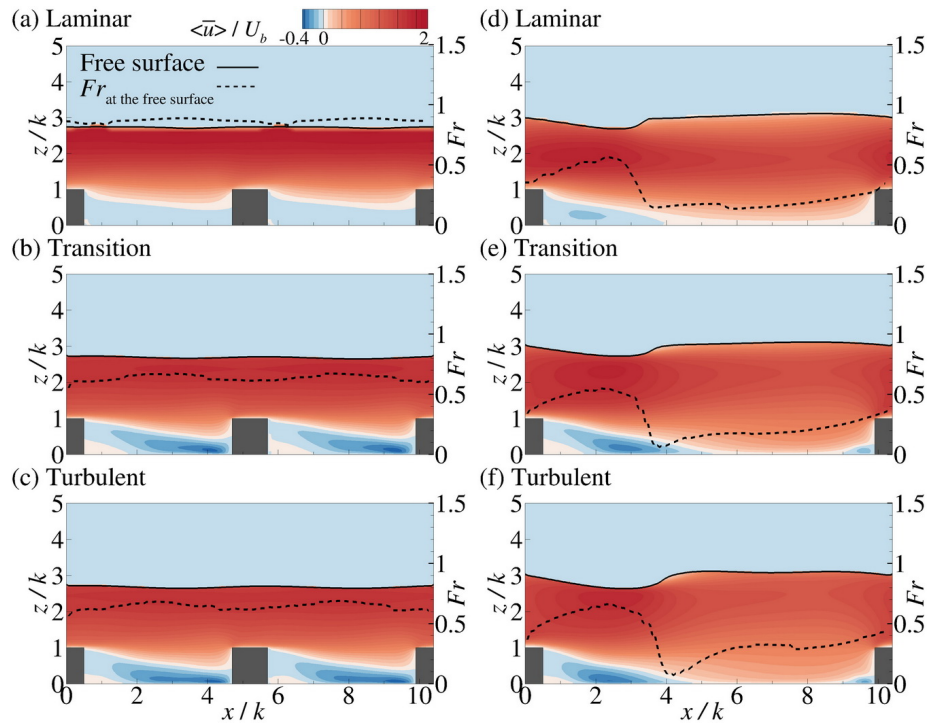


Figure 7: Contours of the time- and spanwise-averaged streamwise velocity together with the local Froude number at the water surface for turbulent, transitional and laminar flow for (a-c)  $\lambda/k = 5.2$  and (d-f)  $\lambda/k = 10.4$ .

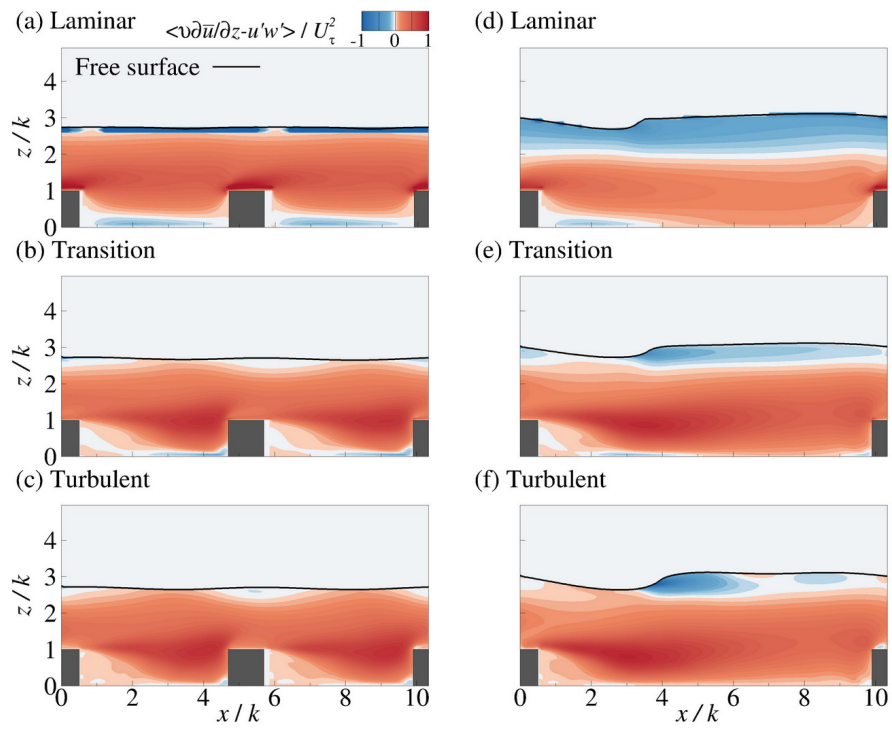


Figure 8: Contours of time- and spanwise-averaged viscous and Reynolds shear stresses in turbulent, transitional and laminar flow for (a-c)  $\lambda/k = 5.2$  and (d-f)  $\lambda/k = 10.4$ .

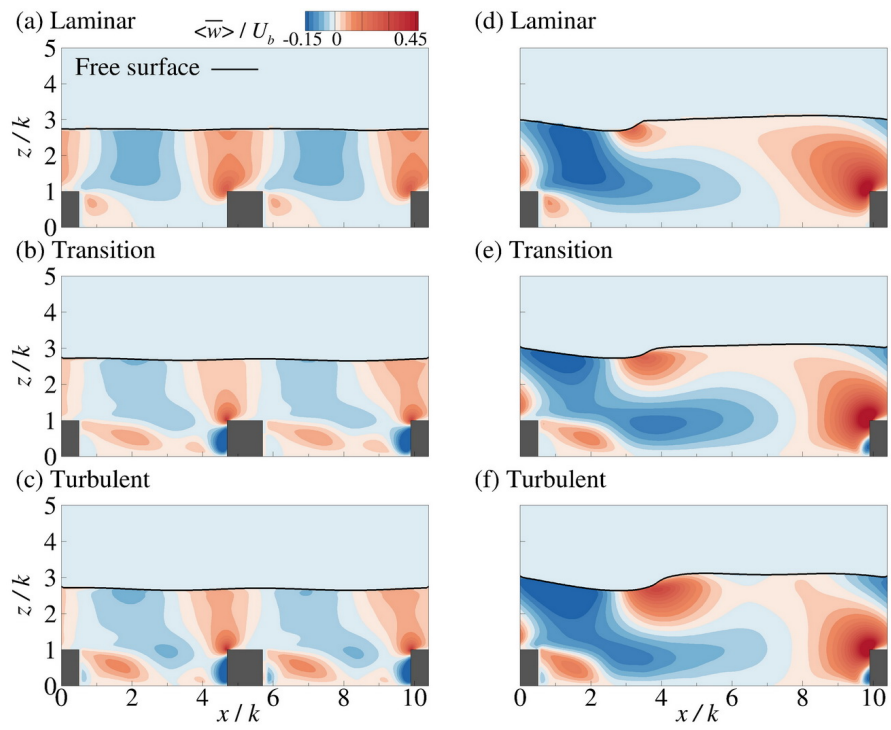


Figure 9: Time- and spanwise-averaged wall-normal velocity in turbulent, transitional and laminar flow for (a-c)  $\lambda/k = 5.2$  and (d-f)  $\lambda/k = 10.4$ .

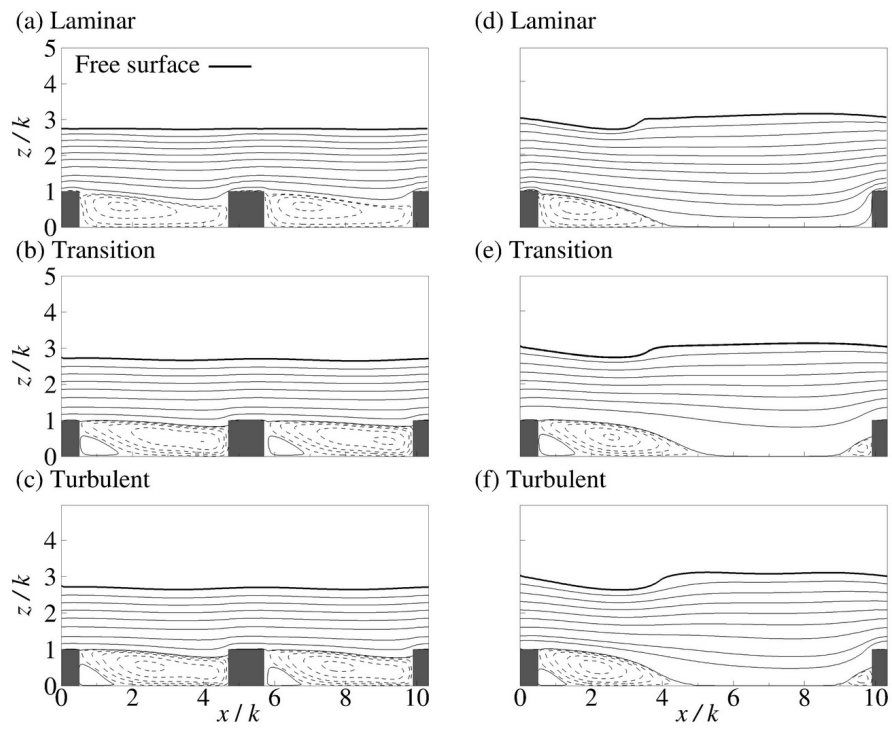


Figure 10: Contour lines of the spanwise-averaged streamfunction in turbulent, transitional and laminar flow for (a-c)  $\lambda/k = 5.2$  and (d-f)  $\lambda/k = 10.4$ .

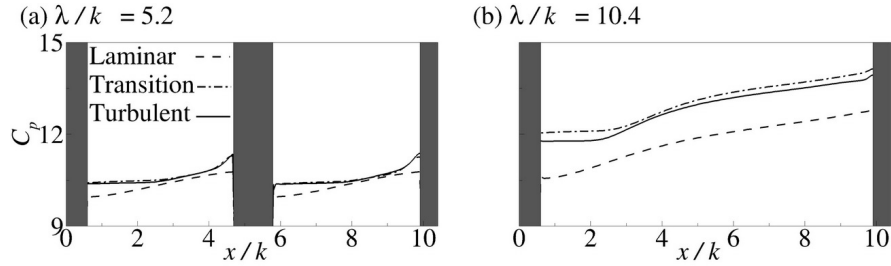


Figure 11: Pressure coefficient in turbulent, transitional and laminar flow for (a)  $\lambda/k = 5.2$  and (b)  $\lambda/k = 10.4$ .

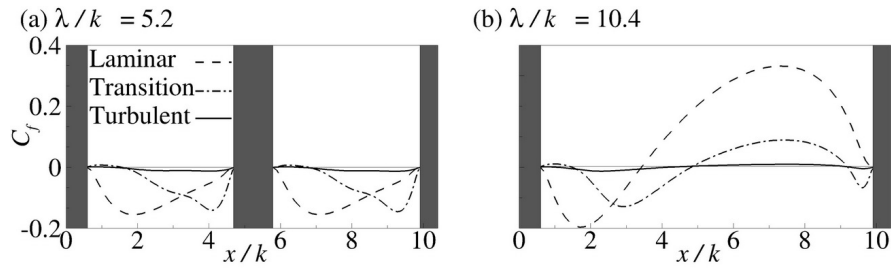


Figure 12: Friction coefficient in turbulent, transitional and laminar flow for (a)  $\lambda/k = 5.2$  and (b)  $\lambda/k = 10.4$ .

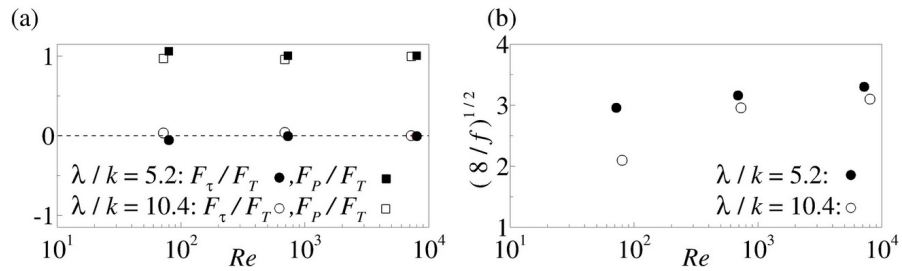


Figure 13: (a) Friction and pressure forces, (b) variation of friction factor with Reynolds number.