
Dynamic output feedback sliding mode control for uncertain linear systems

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Abstract

In this paper, a class of uncertain linear systems with unmatched disturbances is considered, where the nominal system representation is allowed to be non-minimum phase. A sliding surface is designed which is dependent on the system output, observed state and estimated uncertain parameters. A linear coordinate transformation is introduced so that the stability analysis of the reduced-order sliding mode dynamics can be conveniently performed. A robust output feedback sliding mode control (OFSMC) is then designed to drive the considered system state to reach the sliding surface in finite time and maintain a sliding motion thereafter. A simulation example for a high incidence research model (HIRM) aircraft is used to demonstrate the effectiveness of the proposed method.

Keywords

Sliding mode control; Non-minimum phase; Dynamical output feedback; Regular form; Unmatched disturbances

Introduction

Physical systems are unavoidably affected by external disturbances and plant uncertainties which may in turn influence the system performance. Sliding mode control

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(Jiang et al., (2021b)) has been widely studied as an effective method to tackle this problem and has been employed in some industrial settings due to its conceptual simplicity and strong robustness properties. Much of the early work on sliding mode control assumed that all system states can be measured (Utkin (1977, 1992)). However, in actual engineering systems, this assumption may be unreasonable. This has motivated the current emphasis on the development of output feedback sliding mode control (OFSMC).

Many OFSMC algorithms have been proposed to stabilize uncertain systems. A static output feedback variable structure control has been investigated for linear systems without disturbances in Elkhazali and Decarlo (1995). A further static OFSMC has been designed for uncertain linear systems with matched disturbances in Edwards and Spurgeon (1995, 1998), where a convenient coordinate transformation has been given to obtain the regular form. However, unmatched disturbances act on many practical systems. These unmatched disturbances directly affect the dynamics of the system when the state moves on the sliding surface and exhibits a sliding mode. The well-known Constrained Lyapunov Problem (CLP) has been widely discussed in the literature and some effective methods have been proposed in Galimidi and Barmish (1986); Edwards et al., (2007) to constructively solve this problem. This work facilitated the development of static OFSMC schemes which tackle the problem of unmatched disturbances Yan et al., (2009a, 2013). However, the literature above requires that the plant under consideration is minimum phase. In practice, non-minimum phase systems may be encountered, such as flexible manipulators (Benosman and Vey (2003)) and the underactuated ship (Consolini and Tosques (2012)).

The study of OFSMC for non-minimum phase systems has recently received great attention. A dynamic OFSMC with a dynamic compensator has been proposed to stabilize a class of uncertain linear systems in Yan et al., (2004), where the nominal system may be non-minimum phase. The results have also been extended to time-delay systems in Yan et al., (2009b, 2010). Furthermore, based on the method proposed in Yan et al., (2004) and a reduced-order compensator, a class of large-scale interconnected systems has been stabilized in Yan et al., (2006). Although the case of more general unmatched disturbances with nonlinear bounds has been considered in the above literature, the unmatched disturbances are required to have zero steady-state values or bounded H_2 norms. In addition, some classic control tools, such as the Riccati approach (Kim et al., (2000)), the LMI-based approach (Park et al., (2007)) and an adaptive approach (Wen and Cheng (2008); Jiang et al., (2021a)) have been used to extend the sliding mode control approach to deal with unmatched disturbances. However, these methods also require that the unmatched disturbances must belong to somewhat restrictive classes of vanishing disturbance, which may be unreasonable in practice (Yang et al., (2013)).

By adding integral action to the sliding surface, integral sliding mode control has become well established and regarded as an effective method to counteract unmatched disturbances with nonzero steady-state value (Zhang and Yan (2019); Cao and Xu (2004)). However, the integral action may cause a large overshoot in the controlled system. More recently, based on the nonlinear disturbance observer proposed in Chen

(2003); Chen et al., (2000), the observed disturbances have been added to the sliding surface to counteract their effects in Yang et al., (2013). This method has attracted a lot of attention due to its simplicity and effectiveness but can only handle a class of second-order systems and lacks a strict stability proof of the sliding mode dynamics. The result has been extended for a nominal system representation that is a series of integrators in Ginoya et al., (2013), where a modified sliding surface has been presented to guarantee the practical stability of the overall system. An extended state observer-based sliding mode control has been designed for pulse-width modulation-based DC-DC buck converter systems in Wang et al., (2015). A disturbance observer-based continuous finite-time sliding mode control has been designed for input affine nonlinear systems in Nguyen et al., (2020), where a nonlinear sliding surface and supertwisting algorithm have been used to guarantee finite-time convergence of the considered system. In addition, disturbance observer-based control methods have also been designed for some uncertain nonlinear systems with unmatched disturbances (Yang et al., (2012); Zhang et al., (2017)). However, the literature mentioned above all requires that the considered system states are accessible. These solutions are frequently motivated by engineering practice and using a disturbance observer in dynamic OFSMC may be effective for uncertain non-minimum phase systems but it should be noted that the theoretical analysis may be challenging.

When the system states cannot be measured and the considered system is non-minimum phase, disturbance observer-based sliding mode control design will face new challenges which include: how to observe the system states and unmatched disturbances simultaneously and how to prove the stability of the associated extended system when the system is restricted on the designed sliding surface. For these challenges, a disturbance observer-based dynamic OFSMC strategy is proposed for a class of uncertain linear systems, where the nominal system representation is allowed to be non-minimum phase. Inspired by Yang et al., (2013), a novel sliding surface which is a function of an augmented space formed by the system output, the observed state and the estimated uncertain parameters, is designed. Furthermore, a new linear coordinate transformation has been derived so that the stability analysis of the reduced-order sliding mode dynamics can be conveniently performed. In comparison with existing static OFSMC methods (Gao et al., (2019); Ji et al., (2019)), the nominal part of the system considered in this paper is allowed to be non-minimum phase. In comparison with most of the existing dynamics OFSMC methods (Consolini and Tosques (2012); Yan et al., (2004, 2009b, 2010, 2006)), the system considered in this paper is allowed to have the unmatched disturbances with nonzero steady-state value. In comparison with most of the existing disturbance observer-based sliding mode control methods (Yang et al., (2013); Nguyen et al., (2020); Ginoya et al., (2013); Wang et al., (2015)), the system state in this paper is not required to be measurable while the unmatched disturbances have more general forms. The main theoretical contributions of this paper include: (i) a novel sliding surface is proposed to counteract the unmatched disturbances when only the system output information is available; (ii) a sufficient condition is given to guarantee the considered system exhibits an exponentially stable sliding motion while the state observed error and parameter estimation error converge to zero exponentially; (iii) a

new linear coordinate transformation is proposed to facilitate the stability analysis of the sliding motion dynamics.

The remainder of the paper is organized as follows. Section II formulates the problem and gives some assumptions that will be used in the following sections. In Section III, a novel sliding surface and the stability analysis of the sliding motion is given. A sliding mode control is designed in Section IV. A simulation example of a HIRM aircraft is presented to validate the proposed approach in Section V while the conclusions are given in Section VI.

Notation: For a square matrix A , $\lambda(A)$ and $\bar{\lambda}(A)$ denote the minimum and maximum eigenvalue respectively.

System description and problem formulation

Consider an uncertain linear system

$$\begin{aligned}\dot{\tilde{x}} &= \tilde{A}\tilde{x} + \tilde{B}u + \tilde{\Psi}(t)\theta \\ y &= \tilde{C}\tilde{x}\end{aligned}\quad (1)$$

where $\tilde{x} \in R^n, y \in R^p, u \in R^m$ are the system state, output and input vectors respectively with $m \leq p < n$, $\tilde{A}, \tilde{B}, \tilde{C}$ are constant matrices with appropriate dimensions, $\tilde{\Psi} \in R^{n \times q}$ is a known matrix, and $\theta \in R^q$ is an unknown constant parameter vector.

Remark 1. It should be noted that although the considered nominal system is linear, most existing sliding mode control methods for nonlinear systems cannot be directly applied to system (1). For instance, a novel second-order sliding mode control has been proposed for a class of nonlinear systems to handle the unmatched disturbances in Ding and Li (2017), but it is difficult to verify the sufficient conditions proposed to guarantee the stability of the sliding mode dynamics. Moreover, some control parameters in the sliding mode control of nonlinear systems may be difficult to obtain (Plestan et al., (2010); Mu et al., (2017)).

The following Assumptions are imposed on system (1).

Assumption 1. System (1) is controllable and observable.

Assumption 2. \tilde{B} and \tilde{C} are both full rank, $rank(\tilde{C}\tilde{B}) = m$.

Under Assumption 2, it follows from Lemma 5.3 in Edwards and Spurgeon (1998) that there exists a nonsingular coordinate transformation $x = \tilde{T}\tilde{x}$ such that the system (1) can be described as:

$$\begin{aligned}\dot{x} &= Ax + Bu + \Psi(t)\theta \\ y &= Cx\end{aligned}\quad (2)$$

where $A = \begin{bmatrix} A_{11} & A_{12} \\ A_{21} & A_{22} \end{bmatrix} = \tilde{T}\tilde{A}\tilde{T}^{-1}$ with $A_{22} \in R^{m \times m}$, $B = \begin{bmatrix} 0 \\ B_2 \end{bmatrix} = \tilde{T}\tilde{B}$ with nonsingular matrix $B_2 \in R^{m \times m}$, $C = \tilde{C}\tilde{T}^{-1}$ and $\Psi(t) = \tilde{T}\tilde{\Psi}(t)$.

The following Assumptions are imposed on system (2).

Assumption 3. $\Psi(t)$ is a known continuous differentiable function and $\Psi(t)$ is bounded with $\|\Psi(t)\| \leq \rho_1$.

Assumption 4. A_{12} is full row rank i.e. $\text{rank}(A_{12}) = n - m$.

Remark 2. It should be pointed out that the disturbances considered in this paper are nonzero time-varying and independent of the system states. The derivative of the unmatched disturbances $d(t)$ considered in Yang et al., (2013) is required to satisfy $\lim_{t \rightarrow \infty} \dot{d}(t) = 0$ while this condition is not required in this study.

Remark 3. It should be noted that Assumptions 1-3 are common and reasonable. Assumption 4 is not conservative and may be satisfied by many linear systems.

It follows from Assumption 1 that (A, C) is also observable. Thus there exists a matrix $L \in R^{n \times p}$ such that $A - LC$ is stable. Therefore for any symmetric matrix $Q_1 > 0$, the Lyapunov equation

$$(A - LC)^T P_1 + P_1(A - LC) = -Q_1 \quad (3)$$

has a unique symmetric solution $P_1 > 0$.

It follows from Proposition 3.3 in Edwards and Spurgeon (1998) that the pair (A_{11}, A_{12}) is controllable. Thus there exists a matrix $K \in R^{m \times (n-m)}$ such that $A_{11} - A_{12}K$ is stable. Then for any symmetric matrix $Q_2 > 0$, the Lyapunov equation

$$(A_{11} - A_{12}K)^T P_2 + P_2(A_{11} - A_{12}K) = -Q_2 \quad (4)$$

has a unique symmetric solution $P_2 > 0$.

The objective is to design a composite sliding surface formed by the system output, the observed state and the estimated parameters such that the reduced-order sliding mode is exponentially stable. For system (2), a dynamic output feedback control is defined of the following form

$$u = u(t, \hat{x}, \hat{\theta}, y) \quad (5)$$

where the joint state-parameter adaptive observers are described by

$$\dot{\hat{x}} = \hat{x}(t, u, \hat{x}, \hat{\theta}, y) \quad (6)$$

$$\dot{\hat{\theta}} = \hat{\theta}(t, u, \hat{x}, y) \quad (7)$$

The system will be designed such that the associated closed-loop system formed by (2) and (5)-(7) can be driven to the pre-designed sliding surface in finite time and a sliding motion can be maintained thereafter.

Sliding surface design and stability analysis of the sliding motion

Sliding surface design based on joint state-parameter adaptive observers

The following state-parameter adaptive observers have been introduced as in Zhang (2002); Zhang and Clavel (2001)

$$\dot{\hat{x}} = A\hat{x} + Bu + (\Upsilon\Gamma^{-1}\Upsilon^T C^T + L)(y - C\hat{x}) + \Psi(t)\hat{\theta} \quad (8)$$

$$\dot{\hat{\theta}} = \Gamma^{-1} \Upsilon^T C^T (y - C\hat{x}) \quad (9)$$

where L satisfies (3). The parameter gain Γ is designed by

$$\dot{\Gamma} = -\rho_2 \Gamma + \Upsilon^T C^T C \Upsilon \quad (10)$$

where the forgetting factor ρ_2 is a sufficiently large positive constant, the initial value $\Gamma(t_0)$ is symmetric positive definite, and Υ is generated by the following dynamical system

$$\dot{\Upsilon} = (A - LC) \Upsilon + \Psi(t) \quad (11)$$

For (11), it is obvious that Υ is bounded since $(A - LC)$ is stable and $\Psi(t)$ is bounded (Zhang (2002)). It follows that there exists a constant ρ_3 such that $\|\Upsilon\| \leq \rho_3$.

Define $e = x - \hat{x}$, $\tilde{\theta} = \theta - \hat{\theta}$ and notice that $\dot{\hat{\theta}} = 0$, then

$$\dot{e} = (A - LC - \Upsilon \Gamma^{-1} \Upsilon^T C^T C) e + \Psi(t) \tilde{\theta} \quad (12)$$

$$\dot{\tilde{\theta}} = -\Gamma^{-1} \Upsilon^T C^T C e \quad (13)$$

Lemma 1. (Zhang and Clavel (2001)) *If the forgetting factor $\rho_2 > 0$ and the initial gain matrix $\Gamma(t_0)$ is symmetric positive definite, then the gain matrix Γ generated by (10) is positive definite and uniformly bounded, i.e. there exists a positive constant ρ_4 such that $\|\Gamma\| \leq \rho_4$. Moreover, if the state-parameter adaptive observers are designed as (8)-(9), then there exist two positive constants χ_1, χ_2 such that $\|e\| \leq \chi_1$, $\|\tilde{\theta}\| \leq \chi_2$, where χ_1, χ_2 are determined by ρ_2, L and the initial values of e and $\tilde{\theta}$.*

In order to facilitate the subsequent proof of convergence, the following definition linear combination of e and $\tilde{\theta}$ is defined

$$\eta = e - \Upsilon \tilde{\theta} \quad (14)$$

where Υ has been defined in (11).

From (12)-(14), it follows that

$$\dot{\eta} = (A - LC) \eta \quad (15)$$

In the $(x, \tilde{\theta}, \eta)$ coordinate system, the following extended system can be described

$$\begin{bmatrix} \dot{x} \\ \dot{\tilde{\theta}} \\ \dot{\eta} \end{bmatrix} = \begin{bmatrix} A & 0 & 0 \\ 0 & -\Gamma^{-1} \Upsilon^T C^T C \Upsilon & -\Gamma^{-1} \Upsilon^T C^T C \\ 0 & 0 & A - LC \end{bmatrix} \begin{bmatrix} x \\ \tilde{\theta} \\ \eta \end{bmatrix} + \begin{bmatrix} B \\ 0 \\ 0 \end{bmatrix} u + \begin{bmatrix} \Psi(t) \theta \\ 0 \\ 0 \end{bmatrix} \quad (16)$$

Remark 4. It should be noted that the information obtained from the observer design has not been used for control design in Zhang (2002); Zhang and Clavel (2001) and no strict Lyapunov proof has been provided.

Remark 5. The extended system (16) contains all the necessary information on the variables and forms the basis for the subsequent design and stability analysis of the sliding mode control.

In order to design a stable sliding mode dynamics, a further transformation $z = Tx$ will be given, where $T = \begin{bmatrix} I_{n-m} & 0 \\ K & I_m \end{bmatrix}$ and K satisfies (4). In the new coordinates, system (2) can be described by:

$$\dot{z} = \begin{bmatrix} \bar{A}_{11} & \bar{A}_{12} \\ \bar{A}_{21} & \bar{A}_{22} \end{bmatrix} \begin{bmatrix} z_1 \\ z_2 \end{bmatrix} + \begin{bmatrix} 0 \\ B_2 \end{bmatrix} u + \begin{bmatrix} T_1 \Psi(t) \theta \\ T_2 \Psi(t) \theta \end{bmatrix} \quad (17)$$

where $z := \text{col} \begin{pmatrix} z_1 & z_2 \end{pmatrix}$, $z_1 \in R^{n-m}$, $z_2 \in R^m$, $\bar{A}_{11} = A_{11} - A_{12}K$, $\bar{A}_{12} = A_{12}$, $T_1 := \begin{bmatrix} I_{n-m} & 0 \end{bmatrix} \in R^{(n-m) \times n}$, $T_2 := \begin{bmatrix} K & I_m \end{bmatrix} \in R^{m \times n}$.

For system (2), design the sliding function as

$$\sigma(y, \hat{x}) = S_1 y + S_2 \hat{x} + \bar{A}_{12}^{\text{right}} T_1 \Psi(t) \hat{\theta} \quad (18)$$

where $\bar{A}_{12}^{\text{right}} := \bar{A}_{12}^T (\bar{A}_{12} \bar{A}_{12}^T)^{-1} \in R^{m \times (n-m)}$ is the right inverse of \bar{A}_{12} , $S_1 \in R^{m \times p}$ is a design parameter, and $S_2 = S - S_1 C$ with $S = \begin{bmatrix} 0 & I_m \end{bmatrix} T$.

Remark 6. Note that \bar{A}_{11} is stable which can make the stability analysis of the sliding mode dynamics more straightforward than with the traditional dynamic OFSMC design, such as Yan et al., (2004, 2009b, 2010, 2006). Furthermore, an additional advantage is that an explicit expression can be given for S, S_1, S_2 .

Stability analysis of the sliding motion

From (18), it follows that

$$\begin{aligned} \sigma(y, \hat{x}, \hat{\theta}) &= S_1 y + S_2 \hat{x} + \bar{A}_{12}^{\text{right}} T_1 \Psi(t) \hat{\theta} \\ &= S_1 C x + S_2 (x - e) + \bar{A}_{12}^{\text{right}} T_1 \Psi(t) \hat{\theta} \\ &= S x - S_2 e + \bar{A}_{12}^{\text{right}} T_1 \Psi(t) \hat{\theta} \\ &= S T^{-1} z - S_2 e + \bar{A}_{12}^{\text{right}} T_1 \Psi(t) \hat{\theta} \\ &= z_2 - S_2 e + \bar{A}_{12}^{\text{right}} T_1 \Psi(t) \hat{\theta} \end{aligned} \quad (19)$$

In the new coordinates, the sliding surface is given by:

$$z_2 = S_2 e - \bar{A}_{12}^{\text{right}} T_1 \Psi(t) \hat{\theta} \quad (20)$$

When system (16) is restricted to the sliding surface (20), the corresponding extended system dynamics are described by

$$\begin{bmatrix} \dot{z}_1 \\ \dot{\hat{\theta}} \\ \dot{\eta} \end{bmatrix} = \begin{bmatrix} \bar{A}_{11} & \bar{A}_{12} S_2 \Upsilon + T_1 \Psi(t) & \bar{A}_{12} S_2 \\ 0 & -\Gamma^{-1} \Upsilon^T C^T C \Upsilon & -\Gamma^{-1} \Upsilon^T C^T C \\ 0 & 0 & A - LC \end{bmatrix} \begin{bmatrix} z_1 \\ \hat{\theta} \\ \eta \end{bmatrix} \quad (21)$$

Theorem 1. Suppose Assumptions 1-4 are satisfied. Then, the extended system (21) is exponentially stable if $M > 0$ with M defined by

$$M = \begin{bmatrix} \underline{\lambda}(Q_2) & -\|P_2 \bar{A}_{12} S_2\| & -\rho_3 \|P_2 \bar{A}_{12} S_2\| - \rho_1 \|P_2 T_1\| \\ -\|P_2 \bar{A}_{12} S_2\| & \underline{\lambda}(Q_1) - \bar{\lambda}(C^T C) & 0 \\ -\rho_3 \|P_2 \bar{A}_{12} S_2\| - \rho_1 \|P_2 T_1\| & 0 & \rho_2 \rho_4 \end{bmatrix} \quad (22)$$

Proof:

For system (21), consider the Lyapunov function candidate

$$V(z_1, \eta, \tilde{\theta}, t) = z_1^T P_2 z_1 + \eta^T P_1 \eta + \tilde{\theta}^T \Gamma \tilde{\theta} \quad (23)$$

The time derivative of V along the trajectories of system (21) is given as

$$\begin{aligned} \dot{V} &= -z_1^T Q_2 z_1 + 2z_1^T P_2 \bar{A}_{12} S_2 (\eta + \Upsilon \tilde{\theta}) + 2z_1^T P_2 T_1 \Psi(t) \tilde{\theta} \\ &\quad - \eta^T Q_1 \eta - 2\tilde{\theta}^T \Gamma \Gamma^{-1} \Upsilon^T C^T C (\eta + \Upsilon \tilde{\theta}) + \tilde{\theta}^T (-\rho_2 \Gamma + \Upsilon^T C^T C \Upsilon) \tilde{\theta} \\ &= -z_1^T Q_2 z_1 + 2z_1^T P_2 \bar{A}_{12} S_2 (\eta + \Upsilon \tilde{\theta}) + 2z_1^T P_2 T_1 \Psi(t) \tilde{\theta} \\ &\quad - \eta^T Q_1 \eta - \rho_2 \tilde{\theta}^T \Gamma \tilde{\theta} - 2\tilde{\theta}^T \Upsilon^T C^T C \eta - \tilde{\theta}^T \Upsilon^T C^T C \Upsilon \tilde{\theta} \end{aligned} \quad (24)$$

Noted that there exists the following relationship

$$\begin{aligned} -2\tilde{\theta}^T \Upsilon^T C^T C \eta - \tilde{\theta}^T \Upsilon^T C^T C \Upsilon \tilde{\theta} &= -(\eta + \Upsilon \tilde{\theta})^T C^T C (\eta + \Upsilon \tilde{\theta}) + \eta^T C^T C \eta \\ &\leq \eta^T C^T C \eta \end{aligned} \quad (25)$$

Then (24) can be further described by

$$\begin{aligned} \dot{V} &\leq -z_1^T Q_2 z_1 + 2z_1^T P_2 \bar{A}_{12} S_2 (\eta + \Upsilon \tilde{\theta}) + 2z_1^T P_2 T_1 \Psi(t) \tilde{\theta} - \eta^T Q_1 \eta - \rho_2 \tilde{\theta}^T \Gamma \tilde{\theta} + \eta^T C^T C \eta \\ &\leq -\underline{\lambda}(Q_2) \|z_1\|^2 + 2\|P_2 \bar{A}_{12} S_2\| \|z_1\| \|\eta\| + 2(\rho_3 \|P_2 \bar{A}_{12} S_2\| + \rho_1 \|P_2 T_1\|) \|z_1\| \|\tilde{\theta}\| \\ &\quad + \{\bar{\lambda}(C^T C) - \underline{\lambda}(Q_1)\} \|\eta\|^2 - \rho_2 \rho_4 \|\tilde{\theta}\|^2 \\ &= -X^T M X \end{aligned} \quad (26)$$

where $X = \begin{bmatrix} \|z_1\| & \|\eta\| & \|\tilde{\theta}\| \end{bmatrix}^T$.

It follows that

$$\dot{V} \leq -\underline{\lambda}(M) \|X\|^2 \quad (27)$$

It should be pointed that although the gain matrix Γ is positive definite, it may be not symmetric. Hence,

$$\begin{aligned}\tilde{\theta}^T \Gamma \tilde{\theta} &= \tilde{\theta}^T \left(\frac{\Gamma + \Gamma^T}{2} + \frac{\Gamma - \Gamma^T}{2} \right) \tilde{\theta} \\ &= \tilde{\theta}^T \left(\frac{\Gamma + \Gamma^T}{2} \right) \tilde{\theta} \\ &\geq \lambda \left(\frac{\Gamma + \Gamma^T}{2} \right) \|\tilde{\theta}\|^2\end{aligned}\quad (28)$$

It follows from (23) and (28) that

$$k_1 \left(\|z_1\|^2 + \|\eta\|^2 + \|\tilde{\theta}\|^2 \right) \leq V(z_1, \eta, \tilde{\theta}, t) \leq k_2 \left(\|z_1\|^2 + \|\eta\|^2 + \|\tilde{\theta}\|^2 \right) \quad (29)$$

where $k_1 = \min \left\{ \lambda(P_2), \lambda(P_1), \lambda \left(\frac{\Gamma + \Gamma^T}{2} \right) \right\}$, $k_2 = \max \{ \bar{\lambda}(P_2), \bar{\lambda}(P_1), \rho_4 \}$.

It follows from (27) and (29) that

$$\dot{V} \leq -\frac{\lambda(M)}{k_2} V \quad (30)$$

Then,

$$V \leq V(t_0) \exp\{2k_3(t - t_0)\} \quad (31)$$

where $k_3 = -\frac{\lambda(M)}{2k_2}$ and $V(t_0)$ represents the initial value.

From (14), it is obvious that

$$\|e\| \leq \|\eta\| + \rho_3 \|\tilde{\theta}\| \quad (32)$$

Furthermore, it follows from (29), (31) and (32) that

$$\|e\| \leq (1 + \rho_3) \sqrt{\frac{V(t_0)}{k_1}} \exp\{k_3(t - t_0)\} \quad (33)$$

Hence, the extended system (21) is exponentially stable while the observation error and parameter estimation error converge to zero exponentially.

Sliding mode control design

Based on the observed state, the estimated parameters given by (8)-(9) and the system output, the following sliding mode control is proposed

$$\begin{aligned}u &= -(SB)^{-1} \left\{ SA\hat{x} + \left(S_2L + S_2\Upsilon\Gamma^{-1}\Upsilon^T C^T + \bar{A}_{12}^{right} T_1 \Psi(t) \Gamma^{-1} \Upsilon^T C^T \right) (y - C\hat{x}) \right. \\ &\quad \left. + \left(\bar{A}_{12}^{right} T_1 \dot{\Psi}(t) + S_2 \Psi(t) \right) \hat{\theta} + \omega \frac{\sigma}{\|\sigma\|} \right\}\end{aligned}\quad (34)$$

where ω is defined by

$$\omega = \mu + \chi_1 \|S_1 C A\| + \chi_2 \|S_1 C\| \|\Psi(t)\| + \|S_1 C\| \|\Psi(t)\| \|\hat{\theta}\| \quad (35)$$

with μ a positive constant.

Theorem 2. *Suppose Assumptions 1-4 are satisfied. The control (34) with ω defined in (35) will drive the system (16) to the sliding surface in finite time and maintain a sliding motion thereafter.*

Proof:

It follows from (16) and (18) that

$$\begin{aligned} \dot{\sigma} &= SBu + SAx + S\Psi(t)\theta - S_2(A - LC - \Upsilon\Gamma^{-1}\Upsilon^T C^T C)e \\ &\quad - S_2\Psi(t)\tilde{\theta} + \bar{A}_{12}^{right} T_1 \dot{\Psi}(t)\hat{\theta} + \bar{A}_{12}^{right} T_1 \Psi(t)\Gamma^{-1}\Upsilon^T C^T (y - C\hat{x}) \\ &= SBu + SAx + S\Psi(t)\theta - S_2 Ae + S_2 LCe + S_2 \Upsilon\Gamma^{-1}\Upsilon^T C^T Ce \\ &\quad - S_2\Psi(t)\tilde{\theta} + \bar{A}_{12}^{right} T_1 \dot{\Psi}(t)\hat{\theta} + \bar{A}_{12}^{right} T_1 \Psi(t)\Gamma^{-1}\Upsilon^T C^T (y - C\hat{x}) \\ &= SBu + SA\hat{x} + S_1 CAe + S_1 C\Psi(t)\theta + S_2\Psi(t)\hat{\theta} + S_2 L(y - C\hat{x}) \\ &\quad + S_2 \Upsilon\Gamma^{-1}\Upsilon^T C^T (y - C\hat{x}) + \bar{A}_{12}^{right} T_1 \dot{\Psi}(t)\hat{\theta} + \bar{A}_{12}^{right} T_1 \Psi(t)\Gamma^{-1}\Upsilon^T C^T (y - C\hat{x}) \end{aligned} \quad (36)$$

By applying the control (34) to (36),

$$\begin{aligned} \sigma^T \dot{\sigma} &= \sigma^T \left\{ SBu + SA\hat{x} + S_1 CAe + S_1 C\Psi(t)\theta + S_2\Psi(t)\hat{\theta} + S_2 L(y - C\hat{x}) \right. \\ &\quad \left. + S_2 \Upsilon\Gamma^{-1}\Upsilon^T C^T (y - C\hat{x}) + \bar{A}_{12}^{right} T_1 \dot{\Psi}(t)\hat{\theta} + \bar{A}_{12}^{right} T_1 \Psi(t)\Gamma^{-1}\Upsilon^T C^T (y - C\hat{x}) \right\} \\ &= \sigma^T \left\{ S_1 CAe + S_1 C\Psi(t)\theta - \omega \frac{\sigma}{\|\sigma\|} \right\} \\ &\leq -\|\sigma\| \left\{ \omega - \|S_1 CA\| \|e\| - \|S_1 C\| \|\Psi(t)\| \|\theta\| \right\} \\ &\leq -\|\sigma\| \left\{ \omega - \|S_1 CA\| \|e\| - \|S_1 C\| \|\Psi(t)\| \left(\|\tilde{\theta}\| + \|\hat{\theta}\| \right) \right\} \\ &\leq -\|\sigma\| \left\{ \mu + \chi_1 \|S_1 CA\| + \chi_2 \|S_1 C\| \|\Psi(t)\| + \|S_1 C\| \|\Psi(t)\| \|\hat{\theta}\| \right. \\ &\quad \left. - \|S_1 CA\| \|e\| - \|\tilde{\theta}\| \|S_1 C\| \|\Psi(t)\| - \|S_1 C\| \|\Psi(t)\| \|\hat{\theta}\| \right\} \\ &\leq -\mu \|\sigma\| \end{aligned} \quad (37)$$

According to the so-called η reachability condition (Edwards and Spurgeon (1998)), Theorem 2 is proved.

Remark 7. According to the principles of state transformation, Theorems 1 and 2 are appropriate for the original system (1). One can use $\hat{x} = \tilde{T}^{-1}\hat{x}$ and $\tilde{x} = \tilde{T}^{-1}x$ directly in the controller implementation.

Simulation example

The proposed approach will now be validated by a practically oriented example. Consider the lateral dynamics of the HIRM aircraft at the trim values of Mach 0.2 and height 5000 ft taken from Kureemun (2002). The system matrices are given by (see Yan et al., (2004))

$$\tilde{A} = \begin{bmatrix} -0.0080 & 0.4100 & -0.9047 & 0.1334 \\ -7.3235 & -0.4278 & 2.6462 & 0 \\ -0.1460 & -0.0247 & -0.1544 & 0 \\ 0 & 1.0000 & 0.4558 & 0 \end{bmatrix}, \tilde{B} = \begin{bmatrix} 0.0181 & -0.0094 \\ -3.1026 & 0.4024 \\ -0.4096 & -0.0833 \\ 0 & 0 \end{bmatrix}$$

$$\tilde{C} = \begin{bmatrix} 1 & 0 & 0 & 0 \\ 0 & 0 & 1 & 0 \end{bmatrix} \quad (38)$$

For the lateral dynamics of the HIRM aircraft, the system states $\tilde{x} := \text{col}(\tilde{x}_1, \tilde{x}_2, \tilde{x}_3, \tilde{x}_4)$ represent sideslip angle (rad), roll rate (rad/s), yaw rate (rad/s) and bank angle (rad) respectively, the system inputs $u := \text{col}(u_1, u_2)$ represent differential tailplane deflection (rad) and differential canard deflection (rad) respectively, the system outputs $y := \text{col}(y_1, y_2)$ represent sideslip angle (rad) and yaw rate (rad/s) respectively. The system zeros are 32.3877 and -0.3254 and thus it is a non-minimum phase system. Suppose that the system suffers from the disturbances $\tilde{\Psi}(t)\theta$ where $\tilde{\Psi}(t)$ is given by

$$\tilde{\Psi}(t) = \begin{bmatrix} 0.0053 \sin t \\ -0.0130 \sin t \\ 0.0990 \sin t \\ 0.1 \cos t \end{bmatrix}^T \quad (39)$$

and $\theta = 0.3$.

The coordinate transformation $x = \tilde{T}\tilde{x}$ is given by

$$\tilde{T} = \begin{bmatrix} 0.9986 & 0.0126 & -0.0516 & 0 \\ 0 & 0 & 0 & 1 \\ -0.0058 & 0.9914 & 0.1309 & 0 \\ 0.0528 & -0.1304 & 0.9901 & 0 \end{bmatrix} \quad (40)$$

Then in the new coordinates x , the system can be rewritten as

$$A = \begin{bmatrix} -0.0433 & 0.1332 & 0.2895 & -0.9112 \\ -0.0109 & 0 & 1.0510 & 0.3209 \\ -7.4092 & -0.0008 & -0.0425 & 2.2539 \\ 0.8377 & 0.0070 & -0.0236 & -0.5044 \end{bmatrix}, B = \begin{bmatrix} 0 & 0 \\ 0 & 0 \\ -3.1296 & 0.3881 \\ 0 & -0.1354 \end{bmatrix}$$

$$C = \begin{bmatrix} 0.9986 & 0 & -0.0058 & 0.0528 \\ -0.0516 & 0 & 0.1309 & 0.9901 \end{bmatrix} \quad (41)$$

and $\Psi(t) = 0.1 [0 \quad \cos t \quad 0 \quad \sin t]^T$.

Since (A, C) is observable, design

$$L = \begin{bmatrix} 1.4049 & 0.7594 \\ 3.7481 & 19.2627 \\ -7.2881 & 1.0807 \\ 0.8892 & 0.8237 \end{bmatrix} \quad (42)$$

such that the eigenvalues of $A - LC$ are $[-1 \quad -1 \quad -0.5 \quad -0.5]^T$. Then choosing $Q_1 = 2I_4$, the solution of the Lyapunov equation (3) is

$$P_1 = \begin{bmatrix} 0.1084 & -0.0550 & 0.0523 & 0.6934 \\ -0.0550 & 0.0369 & -0.0286 & -0.3821 \\ 0.0523 & -0.0286 & 0.0570 & 0.3878 \\ 0.6934 & -0.3821 & 0.3878 & 4.8671 \end{bmatrix} \quad (43)$$

Since (A_{11}, A_{12}) is controllable, design

$$K = \begin{bmatrix} 1.1990 & 1.7753 \\ -3.9614 & 0.4179 \end{bmatrix} \quad (44)$$

Choosing $Q_2 = I_2$, the solution of the Lyapunov equation (4) is

$$P_2 = \begin{bmatrix} 0.1250 & 0 \\ 0 & 0.2500 \end{bmatrix} \quad (45)$$

The further transformation $z = Tx$ is given by

$$T = \begin{bmatrix} 1 & 0 & 0 & 0 \\ 0 & 1 & 0 & 0 \\ 1.1990 & 1.7753 & 1 & 0 \\ -3.9614 & 0.4179 & 0 & 1 \end{bmatrix} \quad (46)$$

The main parameters of the sliding function (18) are given as:

$$S = \begin{bmatrix} 1.1990 & 1.7753 & 1.00 & 0 \\ -3.9614 & 0.4179 & 0 & 1.00 \end{bmatrix}, S_1 = \begin{bmatrix} 0.1 & 0.2 \\ -0.1 & 0.2 \end{bmatrix} \quad (47)$$

$$\bar{A}_{12}^{right} = \begin{bmatrix} 0.3054 & 0.8673 \\ -1.0004 & 0.2756 \end{bmatrix}$$

Based on the above analysis, Assumptions 1-4 and Lemma 1 are all satisfied with $\rho_1 = 0.1, \rho_2 = 5$.

Meanwhile, by direct computation, it follows that $\rho_3 = 2, \rho_4 = 2$ and

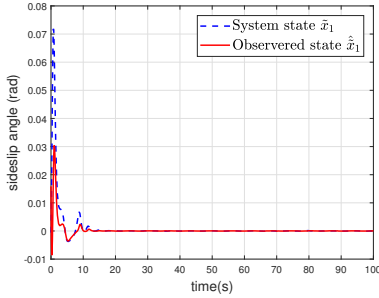
$$M = \begin{bmatrix} 1 & -0.5620 & -1.1491 \\ -0.5620 & 1 & 0 \\ -1.1491 & 0 & 10 \end{bmatrix} \quad (48)$$

Thus the conditions of Theorem 1 are satisfied. The parameters of the control (34) are given as:

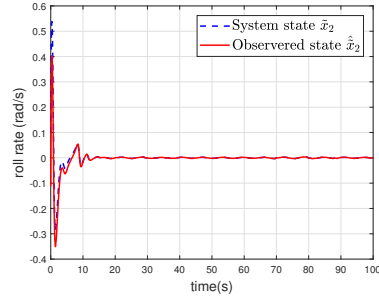
$$\mu = 0.5, \chi_1 = 0.6, \chi_2 = 0.4 \quad (49)$$

which guarantee that Theorem 2 holds.

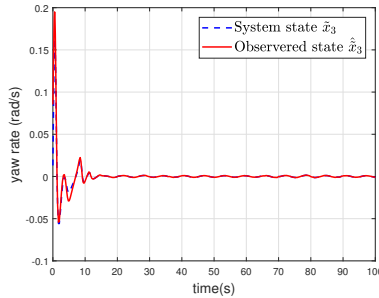
For simulation purposes, $\sigma / (\|\sigma\| + \beta)$ is used to replace $\sigma / \|\sigma\|$ to reduce the chattering with $\beta = 0.01$.



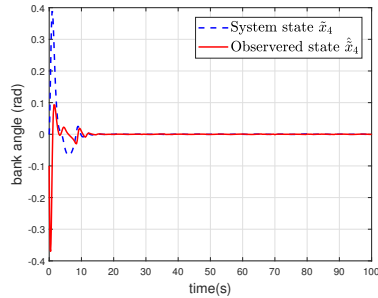
(a) The response of the sideslip angle and its estimate



(b) The response of the roll rate and its estimate



(c) The response of the yaw rate and its estimate



(d) The response of bank angle and its estimate

Figure 1. The time responses of the system states and their estimates

The time responses of the system states and their estimates are shown in Fig.1. The time responses of the sliding functions σ and the system control signal u are shown in Fig.2. The parameter estimate is shown in Fig.3. From Figs.1-3, it can be seen that the proposed joint state-parameter adaptive observer can estimate the system states and the uncertain parameter effectively. The HIRM aircraft is stabilized despite the presence of unmatched disturbances with nonzero steady-state value. The simulation results show that the proposed disturbance observer-based dynamic OFSMC is effective.

To further test the proposed disturbance observer-based dynamic OFSMC, the results obtained in Yan et al., (2004) will be compared with the results in this paper. The

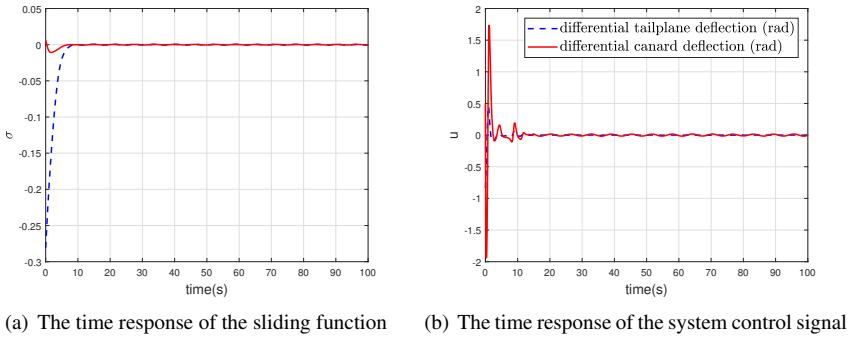


Figure 2. The time responses of the sliding function and system control signal

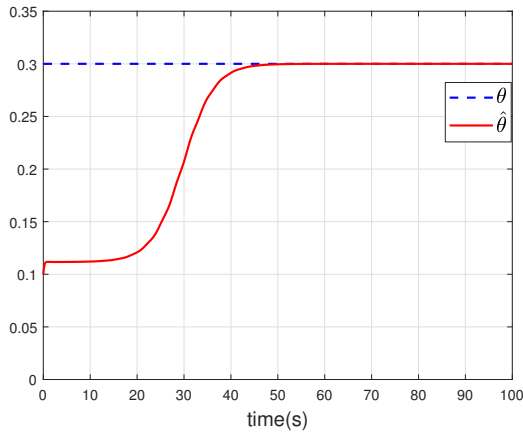
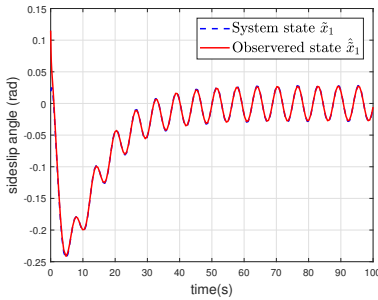


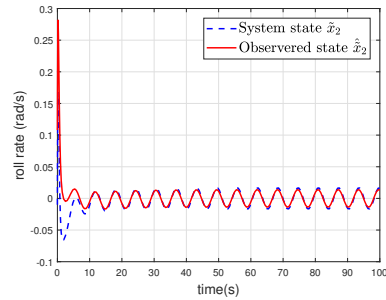
Figure 3. Estimate of the parameter and its actual value

selection of the control parameters is the same as those in Yan et al., (2004). The time responses of the system states and their estimates using the dynamic OFSMC proposed in Yan et al., (2004) are shown in Fig.4. The time responses of the sliding functions σ and the system control signal u are shown in Fig.5. Compared with Figs.1-5, the proposed dynamic OFSMC has a faster response speed and better system performance due to its direct consideration of the unmatched disturbances with nonzero steady-state value.

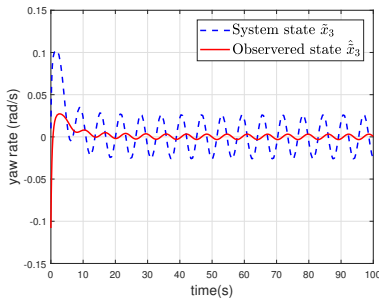
It should be noted that μ , χ_1 and χ_2 are important factors to ensure that the considered system can reach the sliding surface in finite time and the method of determining χ_1 and χ_2 has been given in Zhang (2002); Zhang and Clavel (2001). In addition, the state observer gain L in (3) and the forgetting factor ρ_2 in (10) determine the convergence rate of the state observation error and parameter estimation error respectively. The following



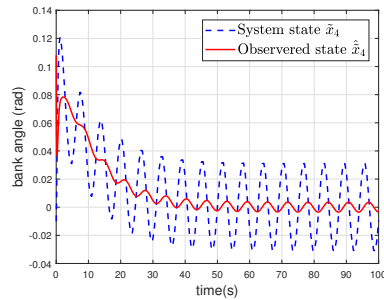
(a) The response of the sideslip angle and its estimate



(b) The response of the roll rate and its estimate



(c) The response of the yaw rate and its estimate



(d) The response of bank angle and its estimate

Figure 4. The time responses of the system states and their estimates using the dynamics OFSMC proposed in Yan et al., (2004)

simulation results will further show the influence of μ on the time for the system to reach the sliding surface and the influence of L and ρ_2 on system performance. Without loss of generality, only the time response of the system state \hat{x}_1 , the sliding function σ , the system control signal u and the estimate of the parameter are given. All simulation times are selected as 100s.

i. Keeping the other parameters in (42)-(49) unchanged, the values of μ are chosen as 0.005, 0.05, 0.5, 5, 50 respectively. The time for the system to reach the sliding surface and the maximum of the norm of u are shown in Table 1. The time response of the system under the condition $\mu = 50$ is shown in Fig.6. The time response of the system under the condition $\mu = 0.005$ is shown in Fig.7. By comparing Figs.1-3 with Figs.6-7 and Table 1, with the increase of μ , the time for the system to reach the sliding surface will be shorter while the system input will gradually increase. In applications, the trial and error method can be used to find an appropriate μ .

ii. Keeping the other parameters in (42)-(49) unchanged, the value of L only is changed. Choose $L = L_1, L_2, L_3, L_4, L_5$ by pole placement so that the

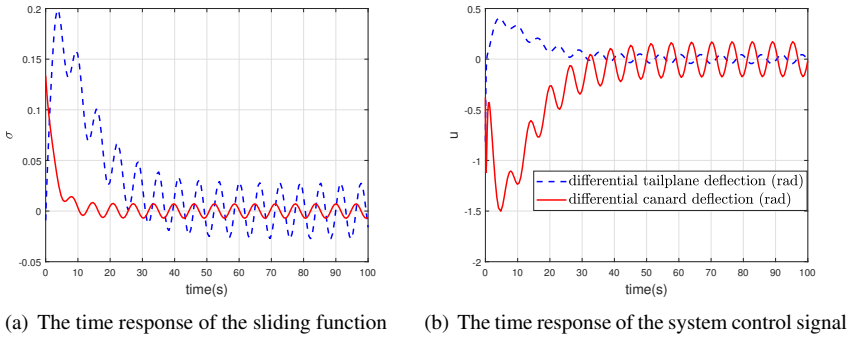


Figure 5. The time responses of the sliding function and system control signal using the dynamics OFSMC proposed in Yan et al., (2004).

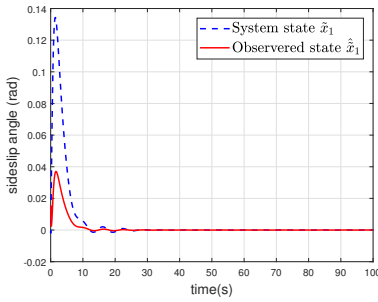
Table 1. The test for μ

μ	The time for the system to reach the sliding surface (s)	$\ u\ _{\max}$
0.005	14.3	1.52
0.05	13.1	1.77
0.5	10.2	1.95
5	2.5	2.86
50	0.2	15.08

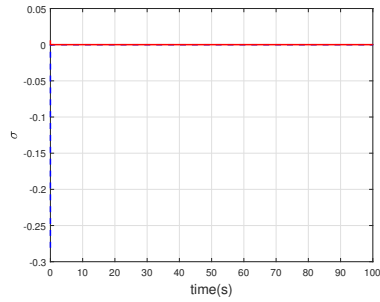
Table 2. The test for L

L	J_1	J_2	J_3	$\ u\ _{\max}$
L_1	7397.1260	5.5873	6.9203	2
L_2	5875.9461	4.3763	4.8650	1.98
L_3	17.5799	1.3640	2.4049	1.95
L_4	41.2148	7.6724	3.1824	568
L_5	1173.1036	16.7710	6.0929	1438

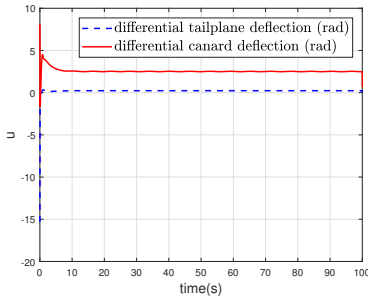
eigenvalues of $A - LC$ are 0.01Ω , 0.1Ω , Ω , 10Ω , 100Ω respectively, with $\Omega = \begin{bmatrix} -1 & -1 & -0.5 & -0.5 \end{bmatrix}^T$. The integral time multiplied square error (ITSE) indexes $J_1 = \int_0^\infty t\tilde{\theta}^2 dt$, $J_2 = \int_0^\infty te^T edt$ and $J_3 = \int_0^\infty t\tilde{x}^T \tilde{x} dt$ are introduced to analyze the system performance. The ITSE results and the maximum of the norm of u are shown in Table 2. The time response of the system under the condition $L = L_1$ is shown in Fig.8. The time response of the system under the condition $L = L_5$ is shown in Fig.9. By comparing Figs.1-3 with Figs.8-9 and Table 2, with the decrease in the eigenvalues of $A - LC$, the convergence time of the state observation error reduces, but the system performance will first improve and then worsen, especially when the eigenvalue is very small, the system input will become very large.



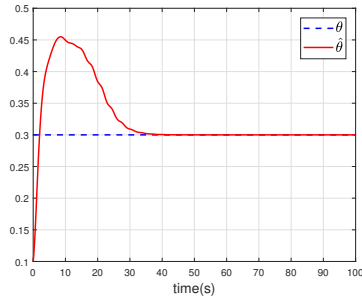
(a) The response of the sideslip angle and its estimate



(b) The time response of the sliding function



(c) The time response of the system control signal



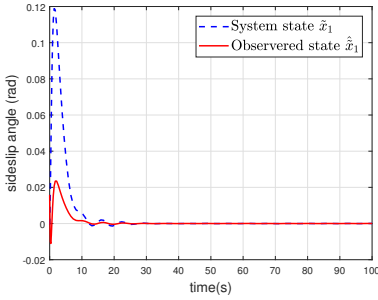
(d) Estimate of the parameter and its actual value

Figure 6. The time response of the system under the condition $\mu = 50$

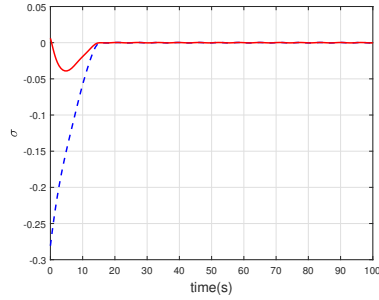
iii. Keeping the other parameters in (42)-(49) unchanged, the values of ρ_2 are chosen as 0.05, 0.5, 5, 50, 500 respectively. The ITSE results and the maximum of the norm of u are shown in Table 3. The time response of the system under the condition $\rho_2 = 0.05$ is shown in Fig.10. The time response of the system under the condition $\rho_2 = 500$ is shown in Fig.11. By comparing Figs.1-3 with Figs.10-11 and Table 3, with the increase of ρ_2 , the convergence time of the parameter estimation error will be shorter, but the system performance will initially improve and then worsen, especially when ρ_2 is very large, the system input will become very large. It should be noted that if ρ_2 is too small, the parameter estimate will not converge to the actual value.

Conclusion

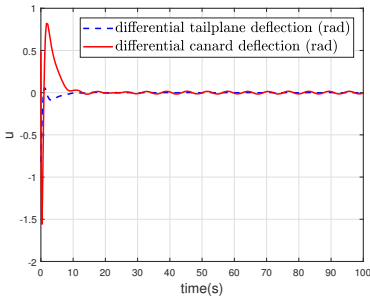
A dynamical OFSMC method has been proposed for a class of uncertain linear non-minimum phase systems. The unmatched disturbances can be allowed to have nonzero steady-state value. A sliding mode control has been designed to ensure that the system states reach the designed sliding surface in finite time. Simulation test results are given



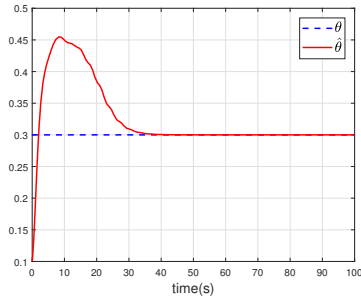
(a) The response of the sideslip angle and its estimate



(b) The time response of the sliding function



(c) The time response of the system control signal



(d) Estimate of the parameter and its actual value

Figure 7. The time response of the system under the condition $\mu = 0.005$

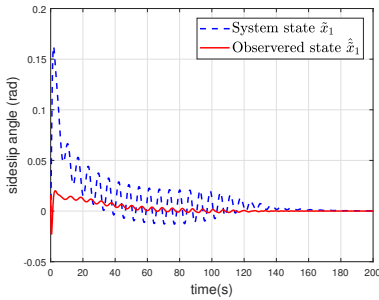
Table 3. The test for ρ_2

ρ_2	J_1	J_2	J_3	$\ u\ _{\max}$
0.05	26.7828	9.5900	11.8311	1.6082
0.5	20.3254	8.5410	11.2451	1.7392
5	17.5799	1.3640	2.4049	1.95
50	110.6306	15.5977	20.2541	14.3216
500	3463.8769	21.1690	27.5700	136.2214

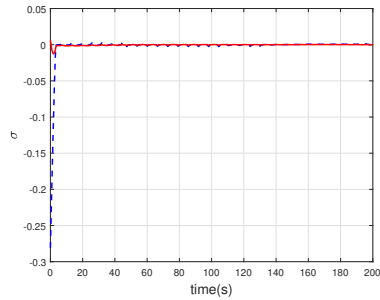
to show the effectiveness of the proposed control scheme. Future work will focus on the application of the proposed method to interconnected systems.

Acknowledgements

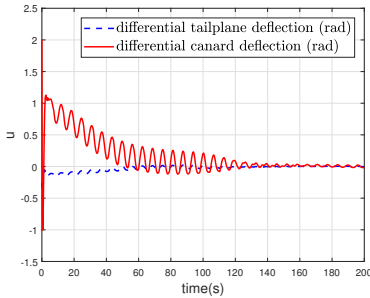
This work is partially supported by the National Nature Science Foundation of China under grant nos 61973315, 61473312.



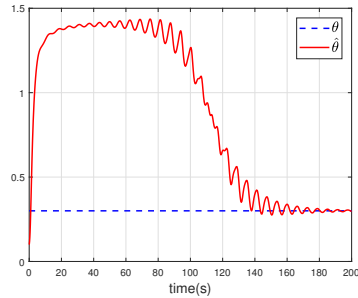
(a) The response of the sideslip angle and its estimate



(b) The time response of the sliding function



(c) The time response of the system control signal

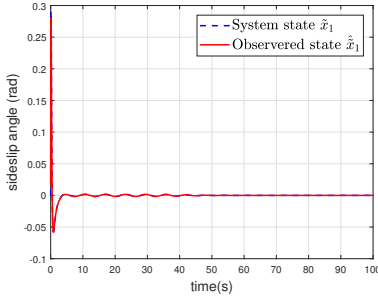


(d) Estimate of the parameter and its actual value

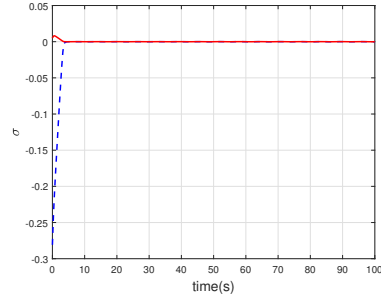
Figure 8. The time response of the system under the condition $L = L_1$

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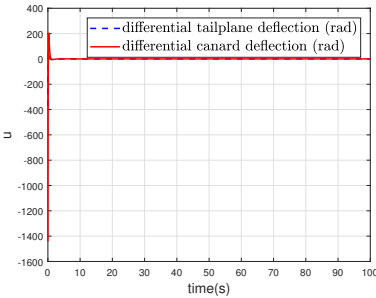
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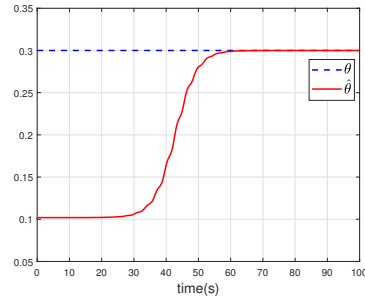
(a) The response of the sideslip angle and its estimate



(b) The time response of the sliding function



(c) The time response of the system control signal



(d) Estimate of the parameter and its actual value

Figure 9. The time response of the system under the condition $L = L_5$.

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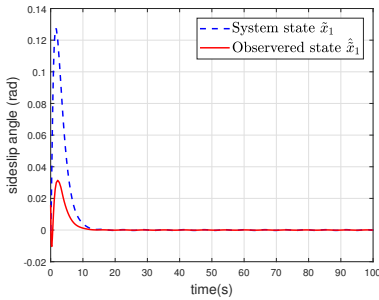
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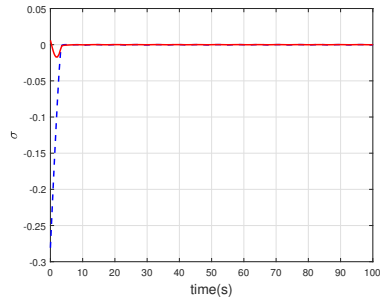
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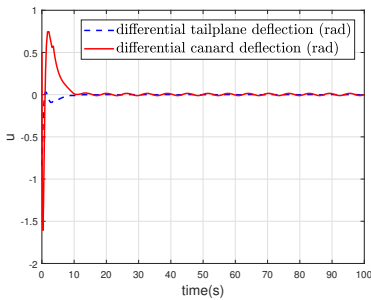
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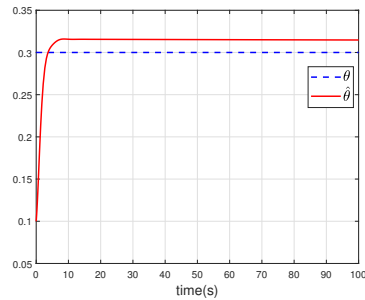
(a) The response of the sideslip angle and its estimate



(b) The time response of the sliding function



(c) The time response of the system control signal



(d) Estimate of the parameter and its actual value

Figure 10. The time response of the system under the condition $\rho_2 = 0.05$

1983-1992.

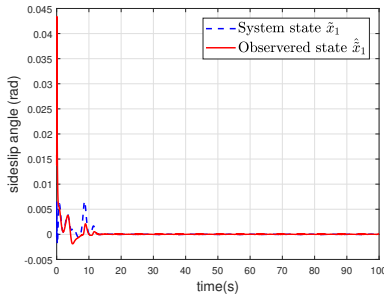
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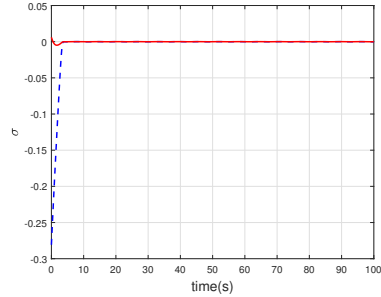
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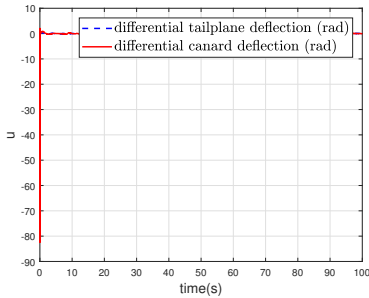
Kureemun, R., 2002, μ -Analysis tools for the flight clearance of highly augmented aircraft, PhD Thesis: Department of Engineering, University of Leicester, UK.



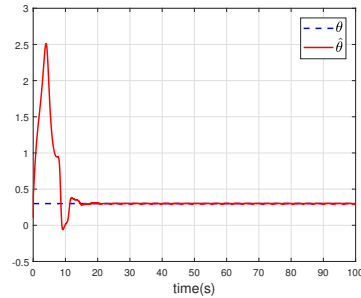
(a) The response of the sideslip angle and its estimate



(b) The time response of the sliding function



(c) The time response of the system control signal



(d) Estimate of the parameter and its actual value

Figure 11. The time response of the system under the condition $\rho_2 = 500$

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