

1     **Study on the performance of two water-side free cooling methods in a**  
2                     **semiconductor manufacturing factory**

3                     Lizhi Jia<sup>a,b</sup>, Junjie Liu<sup>a,\*</sup>, Shen Wei<sup>b</sup>, Jing Xu<sup>c</sup>

4             <sup>a</sup>Tianjin Key Laboratory of Indoor Air Environmental Quality Control, School of  
5     Environmental Science and Engineering, Tianjin University, Tianjin 300072, P.R.  
6                     China

7             <sup>b</sup>The Bartlett School of Construction and Project Management, University College  
8                     London (UCL), London, WC1E 7HB, UK

9             <sup>c</sup>Tianjin Tranair Technology Co.,Ltd. Tianjin 300392, P.R. China

10    **Abstract:** Buildings account for a large portion of global energy consumption and CO<sub>2</sub>  
11    emissions. Therefore, reducing the energy demands of buildings has become a global  
12    topic for sustainable development. A central cooling system accounts for a significant  
13    part of a building's energy consumption. This study proposes a tap water-based free  
14    cooling system that recovers heat from spaces for tap water preheating in winter. The  
15    tap water-based free cooling and cooling tower free cooling systems integrated with a  
16    water-cooled central cooling system were applied in a semiconductor manufacturing  
17    factory. Based on the whole-year operation data, the performance of the tap water-  
18    based free cooling and cooling tower free cooling systems were compared from  
19    thermodynamic, energy, environmental, and economic perspectives. The results  
20    showed that free cooling was the dominant cooling method in winter in Tianjin.  
21    Compared with the cooling tower free cooling system, the tap water-based free cooling  
22    system could provide slightly higher chilled water temperature resulting in 10% shorter  
23    operating time. The energy analysis revealed that the coefficient of performance (COP)  
24    of the tap water-based free cooling system was approximately 7.4-fold and 2.2-fold  
25    higher than that of the mechanical cooling and cooling tower free cooling systems,  
26    respectively. Using the two free cooling methods reduced electricity consumption by  
27    6,044 MWh and reached an annual energy-saving rate of 15.1%. Furthermore, the tap  
28    water-based free cooling system saved  $1.48 \times 10^5$  kg of natural gas for tap water  
29    preheating. Energy reductions attributed to two free cooling methods reduced CO<sub>2</sub>  
30    emissions by 6,236 tons. The tap water-based free cooling is more environmentally

31 friendly with a 4.4-fold greater CO<sub>2</sub> emission reduction rate than the cooling tower free  
 32 cooling method. From the economic perspective, with a short payback period (1.4  
 33 years), the tap water-based free cooling system is an attractive solution for improving  
 34 the energy efficiency of central cooling systems.

35 **Keywords:** Building energy, Energy-efficient, CO<sub>2</sub> emission reduction, Free cooling,  
 36 Tap water-based free cooling.

## 37 1. Introduction

Nomenclature			
$c_p$	Specific heat capacity, kJ/(kg·°C)	$\Delta OC$	Operation cost savings, RMB
$C_{pur}$	Purchase cost, RMB	$\mu$	CO <sub>2</sub> emission factor, kg/kWh
$COP$	Coefficients of performance	$\rho$	Density, kg/m <sup>3</sup>
$\overline{COP}$	Average COP	$\tau$	The billing period, hours
$ES$	Electricity savings, kWh	<i>Subscripts</i>	
$H_{NG}$	Heating load reduction, kWh	$CH$	Chillers
$LHV$	Lower heating value, kJ/kg	$CP$	Cooling pumps
$m$	Mass flow rate, kg/s	$CT$	Cooling tower
$M$	Mass of CO <sub>2</sub> reduction, tons	$CTFC$	Cooling tower free cooling
$n$	Total number of sampling data	$Ele$	Electricity
$p_{Ele}$	Price of electricity, RMB/kWh	$Exp$	Heat exchanger pumps
$p_{NG}$	Price of natural gas, RMB/m <sup>3</sup>	$FC$	Free cooling
$PBP$	Payback period, year	$i$	The number of sampling data
$q$	Cooling supply, kW	$MC$	Mechanical cooling
$Q$	Cooling supply in a period, kWh	$NG$	Natural gas
$r$	Installation factor	$PriP$	Primary pumps
$R_{CO_2}$	CO <sub>2</sub> reduction rate, ton/MWh	$re$	Return chilled water
$S$	Standard deviation	$su$	Supply chilled water
$SR$	Energy Saving rate, %	$t$	Time
$T$	Temperature, °C	$tap$	Tap water
$\bar{T}$	Moving average temperature, °C	$TWFC$	Tap water-based free cooling
$W$	Power consumption, kW	$w$	Chilled Water
<i>Greek symbols</i>		$wet$	Wet-bulb
$\Delta IC$	Additional initial cost, RMB		

38 Buildings account for nearly 40% of global energy consumption and CO<sub>2</sub> emissions [1,  
 39 2]. Therefore, reducing the energy demands of and CO<sub>2</sub> emissions from buildings has  
 40 become a global topic for sustainable development [3, 4]. High-tech manufacturing  
 41 factories are among the most energy-intensive buildings, which consume 30-50-fold  
 42 more energy than typical commercial buildings [5]. In these factories, 40%-50% of

43 energy is consumed by process equipment, and the remaining energy is consumed by  
44 the facility system [6, 7]. Due to the rigorous control requirements for the indoor  
45 environment, the central cooling system is a dominant energy consumer in a facility  
46 system, accounting for 20%-30% of the total energy consumed [7-9]. Thus, central  
47 cooling system energy savings are crucial for reducing the energy consumption of  
48 these factories.

49 Free cooling is an energy-efficient solution to reduce the cooling system energy  
50 consumption using natural cooling sources in suitable climate zones [10]. Free cooling  
51 systems can be grouped into three categories: air-side free cooling, water-side free  
52 cooling, and heat pipe free cooling [11, 12]. Air-side free cooling systems use the  
53 outdoor cold air for space cooling by drawing the outdoor air inside directly or indirectly  
54 via economizers. Direct air-side free cooling is simple to use, but it may introduce  
55 humidity disturbances, particulates, and gas contaminants inside [13, 14]. Indirect air-  
56 side free cooling can overcome those drawbacks, but it is less efficient and incurs  
57 higher initial and operational costs. Because of these concerns, water-side free cooling  
58 is preferred in some applications, such as for data centers and cleanrooms [15]. Water-  
59 side free cooling utilizes natural cold sources through a cooling water infrastructure,  
60 which allows the free cooling process to be introduced without compromising the  
61 internal environment [11, 16]. Heat pipe free cooling is a special kind of economizer  
62 with the ability to transfer heat with small temperature differences. However, this  
63 technology is still in the exploratory stage and has not been widely adopted due to its  
64 unfamiliarity and reliability concerns [11]. In the literature, studies on free cooling  
65 technologies mainly focus on data centers because of the large amount of heat  
66 generated all year-round. For electronic cleanrooms, cooling is required 24 hours per  
67 day and 365 days per year to remove the heat generated by the process equipment  
68 [17]. Due to rigorous indoor environment control requirements, water-side free cooling  
69 is a reasonable way to reduce the energy used by the central cooling system. Water-  
70 side free cooling can be further classified into direct water-side free cooling, dry cooler,  
71 and cooling towers (wet cooler) systems. Direct water-side free cooling systems cool  
72 the indoor air by directly pumping natural cold water into cooling coils in air-conditioned

73 zones. Clidas et al. [18] proposed a seawater-based free cooling system and  
74 designed a closed-loop to transfer heat from the indoor environment to seawater. Li et  
75 al. [19] employed lake water to cool the data center through a heat exchanger. However,  
76 these water-side free cooling methods are limited by the location of the buildings.  
77 James and Rubenstein [20] invented geothermal free cooling in which heat  
78 exchangers buried in the earth directly to remove heat generated by computer services.  
79 Similarly, a water/soil heat exchanger integrated with radiant floor coils was utilized for  
80 free space cooling [21]. However, the high initial investment and fouling concerns have  
81 hindered the broad application of geothermal free cooling. A dry cooler free cooling  
82 system utilizes low-temperature outdoor air to cool chilled water, and the dry cooler is  
83 usually integrated into air-cooled chillers [22]. However, the cooling capacity of this  
84 free cooling method is limited by the outdoor dry-bulb temperature. Cooling tower free  
85 cooling (CTFC) systems are the most widely used because they could produce colder  
86 chilled water by evaporative cooling. The CTFC is an energy-efficient solution, and the  
87 energy savings are higher in cooler climate zones [23]. In electronic cleanrooms,  
88 cooling demands are significant all year round, making free cooling methods sensible  
89 for producing chilled water in cold seasons [24].

90 Combined space cooling with water heating is another method to improve the energy  
91 efficiency of the central cooling system. There are numerous studies focused on  
92 recovering heat from space cooling systems to heat/preheat water. The space cooling  
93 and water heating systems could be combined through a heat exchanger directly or  
94 thermal energy storage indirectly [25]. Ji et al. [26, 27] investigated the energy  
95 performance of a split-type air-conditioner integrated with a water heater in summer in  
96 a subtropical climate zone, and the results have shown that the condensing heat  
97 recovery could considerably improve the system energy performance. Jiang et al. [28]  
98 studied the dynamic characteristic of a combined system that reuses the condensing  
99 heat for domestic hot water heating. This combined system has shown great economic  
100 and environmental value, with 38.6% higher energy efficiency. Yi et al. [29] employed  
101 a helical heat exchanger to transfer condensing heat from the water-cooled air-  
102 conditioner to the domestic hot water system, and the energy efficiency of the system

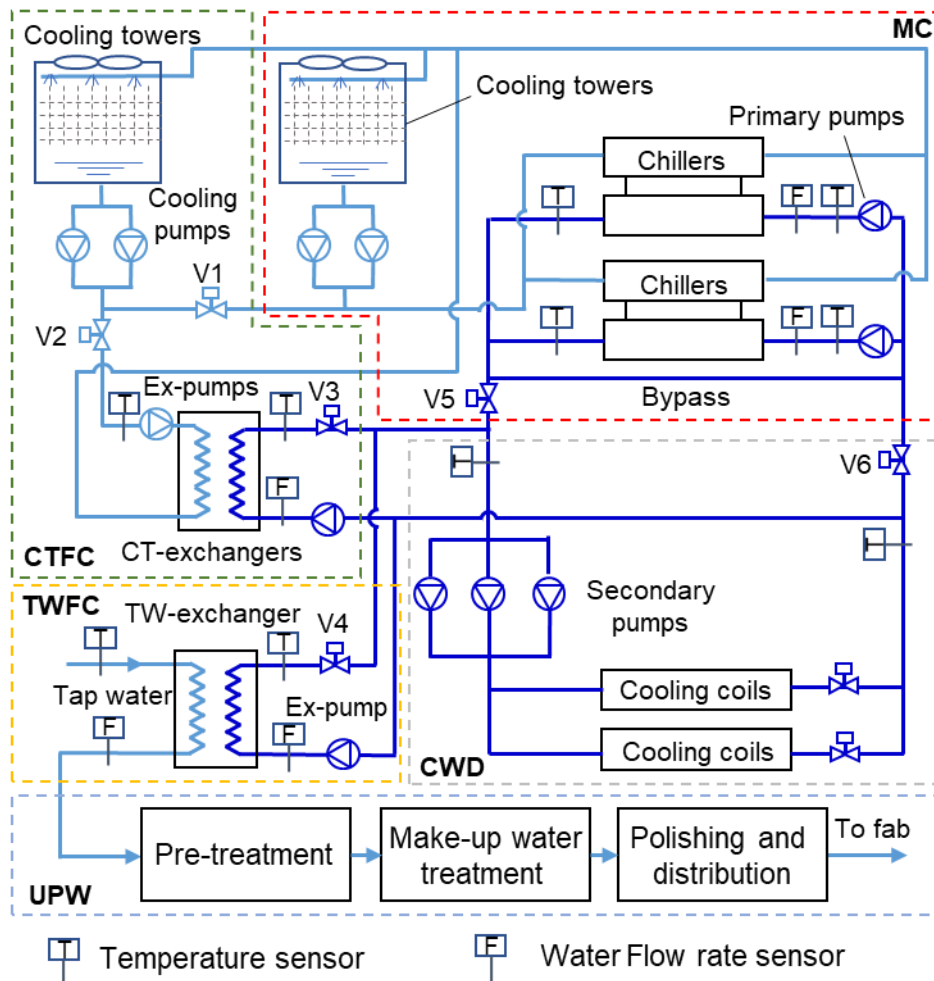
103 was increased by 12.3%. Chen and Lee [30] conducted questionnaire surveys to  
104 obtain the simultaneous consumption characteristic of space cooling and water heating  
105 in Hong Kong, and the potential energy savings of the combined system was 50% for  
106 typical public rental housing. The success of the combined system depended heavily  
107 on the matching degree of space cooling and water heating demands. To solve the  
108 mismatch of the demands, thermal energy storage with and without phase change  
109 material was employed [31-33]. In summary, the typical system that combined space  
110 cooling with water heating utilized the condensing heat from air-conditioners to  
111 heat/preheat domestic hot water. These studies were mainly focused on domestic air-  
112 conditioners and hot water systems in tropical and subtropical zones. The combined  
113 system could only operate in summer with the simultaneous demand for space cooling  
114 and water heating.

115 For semiconductor manufacturing factories, the cleanrooms are required to be cooled  
116 all year-round, and the ultrapure water is needed to be heated in winter. Therefore, a  
117 tap water-based free cooling method was proposed to combine space cooling and  
118 water preheating, and this method was designed to be used in winter. The heat  
119 generated by process equipment was directly transferred to the tap water through a  
120 heat exchanger rather than energy-consuming air-conditioners. In this system, a small  
121 amount of energy was consumed by pumps to overcome the flow resistance, and thus  
122 it was named as tap water-based free cooling method. The tap water-based free  
123 cooling system and cooling tower free cooling system were integrated into a water-  
124 cooled central cooling system in a semiconductor manufacturing factory located in  
125 Tianjin. The performance of two free cooling methods was evaluated and compared  
126 from different perspectives using whole-year operating data. In Section 2, the  
127 principles and the operation strategy of the different cooling methods are described,  
128 and the performance metrics are introduced. In Section 3, the performance of tap  
129 water-based free cooling and the cooling tower free cooling systems are compared  
130 from thermodynamic, energy, environmental, and economic perspectives. The  
131 characteristics of the tap water-based free cooling system and its application prospects  
132 are discussed in Section 4.

133 **2. Methodology**

134 **2.1 System description**

135 This study focuses on a central cooling system that integrated free cooling methods  
136 with the mechanical cooling system. This central cooling system with multiple cooling  
137 sources was applied in a semiconductor manufacturing factory in Tianjin. The  
138 manufacturing of semiconductors requires cleanrooms to meet the environmental  
139 requirement of production. In this case, the cleanrooms are required to achieve the  
140 cleanness of ISO Class - 5 to Class - 7 for different areas. The main indoor design  
141 temperature and humidity are  $22\pm 1$  °C and  $45\pm 5\%$ . The system cooling load mainly  
142 includes heat generated by the process equipment and heat bring by fresh air. As the  
143 production line works 24 hours per day throughout the year, the cleanrooms require  
144 cooling all year-round. In this system, dry cooling coils (DDC) with medium-  
145 temperature chilled water were employed to remove the sensible cooling load, and this  
146 part of the cooling load is relatively stable. Make-up air units are used to process the  
147 fresh air, and their cooling demand varies with the outdoor weather conditions. This  
148 study focused on free cooling methods to improve the energy efficiency of the central  
149 cooling system. In winter, cooling towers and cold tap water can be used to reduce the  
150 energy consumption of chillers and improve the energy efficiency of the central cooling  
151 system. Figure 1 shows the central cooling system of the semiconductor manufacturing  
152 factory in this study. In this system, three different cooling sources, chillers, cooling  
153 towers, and tap water, share the chilled water distribution (CWD) system. In the CWD  
154 loop, six variable speed secondary pumps are designed to transport medium-  
155 temperature chilled water to terminal cooling coils and three pumps are used for low-  
156 temperature coolant distribution. With multiple cooling sources, the system can be  
157 divided into three subsystems, mechanical cooling, cooling tower free cooling, and tap  
158 water-based free cooling subsystems.



159

160 Figure 1: Diagram of the central cooling system and the ultrapure water system

161 **2.1.1 Mechanical cooling**

162 The mechanical cooling system lowers the temperature of chilled water using various  
 163 refrigerating systems. For large buildings, the water-cooled chillers are generally used  
 164 to transfer heat from the chilled water loop to the cooling water loop. Then, cooling  
 165 towers reject heat from cooling water to ambient air by water evaporation. The  
 166 mechanical cooling system is shown in the red box in Figure 1. In this system, nine  
 167 medium-temperature centrifugal chillers with dedicated primary pumps are employed  
 168 to remove the sensible cooling load and precool the fresh air, and three low-  
 169 temperature centrifugal chillers with dedicated primary pumps are used for fresh air  
 170 dehumidification in summer. To exhaust heat from chillers, nine cooling towers and  
 171 nine cooling pumps are configured in the cooling water loop. The specifications of the  
 172 major equipment used in the central cooling system are listed in Table 1.

173 Table 1. Specifications of the major equipment of the central cooling system

Equipment	Number	Power (kW)	Capacity (kW)	Flow rate (m <sup>3</sup> /h)	Head (kPa)	Area (m <sup>2</sup> )
Chiller	9	893	4,572	-	-	-
	3	619	2,286	-	-	-
Cooling tower	8	110	8,272	-	-	-
	1	150	11,030	-	-	-
Primary pump	9	19	-	440	119.5	-
	3	22	-	360	164.3	-
Secondary pump	6	93	-	760	289.7	-
	3	30	-	360	194.2	-
Cooling pump	8	93	-	1,425	194.2	-
	1	250	-	1,900	274.6	-
Ex-pump	10	55	-	650	294.2	-
	1	30	-	420	147.0	-
CT-exchanger	5	-	-	-	-	325
TW-exchanger	1	-	-	-	-	304

174 **2.1.2 Cooling tower free cooling**

175 Cooling tower free cooling is a widely used energy-saving technology in water-cooled  
 176 central cooling systems. In a cooling tower free cooling system, cooling towers produce  
 177 cold water and then cool the chilled water through heat exchangers. The green box in  
 178 Figure 1 shows a schematic of a typical cooling tower free cooling system. In this case,  
 179 four cooling towers and five plate heat exchangers (named CT-exchangers) are  
 180 employed to produce chilled water in winter. To overcome the flow resistance of heat  
 181 exchangers, pumps (named Ex-pumps) are employed on both sides of the heat  
 182 exchangers.

183 **2.1.3 Tap water-based free cooling**

184 Additional to being energy-intensive, high-tech manufacturing is a water-intensive  
 185 industry because of the large amount of water involved in the rinsing and cleaning  
 186 procedures of the production lines [34-36]. In these procedures, ultrapure water is  
 187 generally used, and ultrapure water use accounts for approximately 70% of the total  
 188 factory water use [37]. To produce ultrapure water, a series of processes, including  
 189 pretreatment, make-up water treatment, and polishing, are conducted on tap water



190 (raw water), as shown in the blue box in Figure 1 [38]. In the pretreatment process, tap  
191 water is usually heated over 20 °C to improve the productivity of the ultrapure water  
192 system [37, 39]. In some special cases, the ultrapure water is required to be heated to  
193 70 °C [39]. Considering the simultaneous demand of heating and cooling, a tap water-  
194 based free cooling system was proposed to reduce the energy consumption of the  
195 factory. The tap water-based free cooling system consists of a heat exchanger that  
196 transfers heat from the chilled water loop (cooling side) to the cold tap water (heating  
197 side), as shown in the orange box in Figure 1. This approach reduces both the energy  
198 consumption for chilled water cooling and tap water heating. In this case, a plate heat  
199 exchanger is employed for tap water-based free cooling (TW-exchanger), and an Ex-  
200 pump is employed to overcome the resistance of the exchanger on the cooling side.  
201 The specifications of the major equipment are listed in Table 1.

## 202 **2.2 System operation strategy**

203 The operation of a central cooling system with multiple cooling sources is complicated.  
204 As the cooling load in winter is mainly sensible heat generated by the process  
205 equipment, the two free cooling methods can efficiently generate medium-temperature  
206 chilled water. The operation strategy of different cooling sources is shown in Table 2.  
207 The free cooling system could be enabled when the downstream water temperature of  
208 economizers is lower than the chilled water supply temperature setpoint (12 °C in  
209 winter). For the cooling tower free cooling system, the downstream temperature of the  
210 economizer can be estimated by the sum of the outdoor wet-bulb temperature and the  
211 approach temperatures of the cooling tower (5.5 °C) and the heat exchanger (2.0 °C).  
212 Thus, the critical point of the system switching is the outdoor air wet-bulb temperature  
213 of 4.5 °C. To prevent short cycling CTFC system, a deadband of 1.0 °C was employed  
214 in the real application. When the outdoor wet-bulb temperature is low enough, the  
215 cooling tower free cooling system starts with valves V2 and V3 open and V1 closed  
216 (Figure 1). For the tap water-based free cooling system, the inlet tap water temperature  
217 should be lower than the chilled water temperature setpoint minus the approach of the  
218 heat exchanger. In this case, when the tap water temperature is lower than 10 °C, the

219 tap water-based free cooling system begins with valve V4 open. The two free cooling  
 220 systems can operate simultaneously when both operation conditions are met. When  
 221 the cooling capacity of the free cooling systems is insufficient, chilled water from  
 222 chillers is mixed with that from economizers before entering the secondary pumps to  
 223 satisfy the cooling demand and maintain the chilled water supply temperature. In this  
 224 scenario, valves V2 to V6 are opened, and V1 is closed. When both free cooling  
 225 methods are disabled, the system operates in a fully mechanical cooling mode with  
 226 valves V1, V5, and V6 open.

227 Table 2. Operating strategy for the central cooling system

Cooling sources	Operating conditions
Cooling tower free cooling	$T_{wet} < 4.5\text{ °C}$
Tap water-based free cooling	$T_{tap} < 10\text{ °C}$
Fully mechanical cooling	$T_{wet} \geq 4.5\text{ °C} \& T_{tap} \geq 10\text{ °C}$

## 228 2.3 Data collection and performance metrics

### 229 2.3.1 Data collection

230 In this study, two free cooling methods are integrated into the central cooling system  
 231 to reduce the system energy consumption. To evaluate the performance of the system,  
 232 water temperature, water flow rate, and the electricity consumption of different cooling  
 233 systems were monitored, which span over one year (2018). The location of the major  
 234 sensors is shown in Figure 1. The specification of sensors and their accuracy were  
 235 listed in

236 Table 3.

237 Table 3. The specification of sensors

Sensors	Range	Accuracy
Water Temperature	-20 - 80 °C	±0.3 °C
Water flow rate	0.1 - 15 m/s	±1%
Electric meter*	0 - 100 V, 1.5 - 6 A	±0.5%

238 \* Transformers are required for equipment with high input voltage and current.

### 239 2.3.2 Performance metrics

240 Based on the measured data, the analysis from the perspectives of energy,  
241 environment, and economic were taken to evaluate the two free cooling systems'  
242 performance. The performance metrics are introduced below.

#### 243 1. Features of the cooling sources

244 The water supply temperature, operating hours, and cooling capacity of two free  
245 cooling systems were evaluated and compared.

##### 246 1) Water supply temperature

247 The water supply temperature and its stability impacted the use of the different free  
248 cooling methods in terms of volume and operating hours. The fluctuation of the water  
249 temperature can be described by the standard deviation of the data. As the variation  
250 of the supply water temperature shows a clear trend, the mean temperature is replaced  
251 by the moving average temperature to calculate the standard deviation, as defined in  
252 Equation (1),

$$253 \quad S = \sqrt{\sum_{i=1}^n \frac{(T_i - \bar{T})^2}{n-1}} \quad (1)$$

##### 254 2) Cooling supply

255 The cooling supply of different cooling sources can be estimated based on Equation  
256 (2),

$$257 \quad q = c_p m_w (T_{re} - T_{su}) \quad (2)$$

#### 258 2. Energy performance

259 The coefficients of performance (COPs), energy savings, and energy-saving rate were  
260 used to evaluate the energy performance of different cooling systems.

##### 261 1) COP

262 The COPs of different cooling methods are evaluated based on the cooling supply and  
263 the total power consumption of associated equipment according to Equations (3) to (5).

264 Since these three cooling systems share the chilled water distribution system, the  
265 energy consumption of secondary pumps is excluded.

266 For the mechanical cooling system, the power consumption of chillers, primary pumps  
267 cooling pumps, and cooling towers are included, as defined in Equation (3),

268 
$$COP_{MC} = \frac{q_{MC}}{W_{CH} + W_{PriP} + W_{CT} + W_{CP}} \quad (3)$$

269 For the cooling tower free cooling system, cooling towers, cooling pumps, and heat  
 270 exchanger pumps are the major energy-consuming equipment.

271 
$$COP_{CTFC} = \frac{q_{CTFC}}{W_{CT} + W_{CP} + W_{Exp}} \quad (4)$$

272 For the tap water-based free cooling system, the heat exchanger pump is the only  
 273 energy-consuming component.

274 
$$COP_{TWFC} = \frac{q_{TWFC}}{W_{Exp}} \quad (5)$$

275 2) Energy savings

276 Compared with the mechanical cooling method, the electricity savings of free cooling  
 277 methods in a billing period can be estimated according to Equation (6),

278 
$$ES = \sum_{t=0}^{\tau} \left( \frac{q_{FC,t}}{COP_{MC}} - W_{FC,t} \right) \quad (6)$$

279 For the tap water-based free cooling method, the heat absorbed from the chilled water  
 280 is used to preheat the tap water in the ultrapure water system. Thus, in addition to the  
 281 electricity savings on cooling side, the tap water-based free cooling system also  
 282 reduces natural gas (NG) consumption. The reduction in natural gas consumption is  
 283 the ratio of heat load reduction to the lower heating value of the natural gas (42700  
 284 kJ/kg) [40], as expressed in Equation (7). The natural gas savings can be converted to  
 285 equivalent electricity savings according to Equation (8).

286 
$$m_{NG,t} = \frac{q_{TWFC}}{LHV} \quad (7)$$

287 
$$ES_{NG} = \frac{p_{NG} \times \sum_{t=0}^{\tau} m_{NG,t}}{\rho_{NG} \times p_{Ele}} \quad (8)$$

288 3) Energy-saving rate

289 Taking the mechanical cooling method as the baseline, the energy-saving rate of the  
 290 different free cooling methods can be estimated according to Equation (9),

291 
$$SR = \frac{ES}{\sum_{t=0}^{\tau} \frac{q_{FC,t}}{COP_{MC}}} \times 100\% \quad (9)$$

292 **3. CO<sub>2</sub> emission reduction**

293 The reduction of energy use helps to reduce CO<sub>2</sub> emissions. The CO<sub>2</sub> emission  
 294 reduction using different free cooling methods can be estimated from the electricity

295 savings and the natural gas heating load reductions by multiplying the CO<sub>2</sub> emission  
 296 factors, as expressed in Equation (10). The CO<sub>2</sub> emission factor of electricity  
 297 consumption is 0.968 kg/kWh, and the CO<sub>2</sub> emission factor of natural gas is 0.220  
 298 kg/kWh [40].

$$299 \quad M_{CO_2} = \mu_{CO_2, Ele} ES + \mu_{CO_2, NG} H_{NG} \quad (10)$$

300 Since the design capacities of two free cooling systems differ greatly, the CO<sub>2</sub> emission  
 301 reduction levels are not comparable. Thus, the CO<sub>2</sub> emission reduction per cooling  
 302 capacity is proposed and named as the CO<sub>2</sub> emission reduction rate, as defined in  
 303 Equation (11),

$$304 \quad R_{CO_2} = \frac{M_{CO_2}}{Q} \quad (11)$$

#### 305 **4. Economic analysis**

306 The simple payback period (PBP) is the amount of time it takes the consumer to  
 307 recover the cost to purchase and install additional equipment as a result of the reduced  
 308 operating costs. The PBP is the ratio of the additional initial cost to the annual operating  
 309 cost savings, as defined in Equation (12). The initial cost can be estimated by  
 310 multiplying the purchase cost with an installation factor, as defined in Equation (13).  
 311 The operational cost savings consist of the savings from electricity reductions and  
 312 natural gas savings as defined in Equation (14).

$$313 \quad PBP = \frac{\Delta IC}{\Delta OC} \quad (12)$$

$$314 \quad \Delta IC = C_{pur} \times r \quad (13)$$

$$315 \quad \Delta OC = ES \times p_{Ele} + \frac{p_{NG}}{\rho_{NG}} \times \sum_{t=0}^{\tau} m_{NG,t} \quad (14)$$

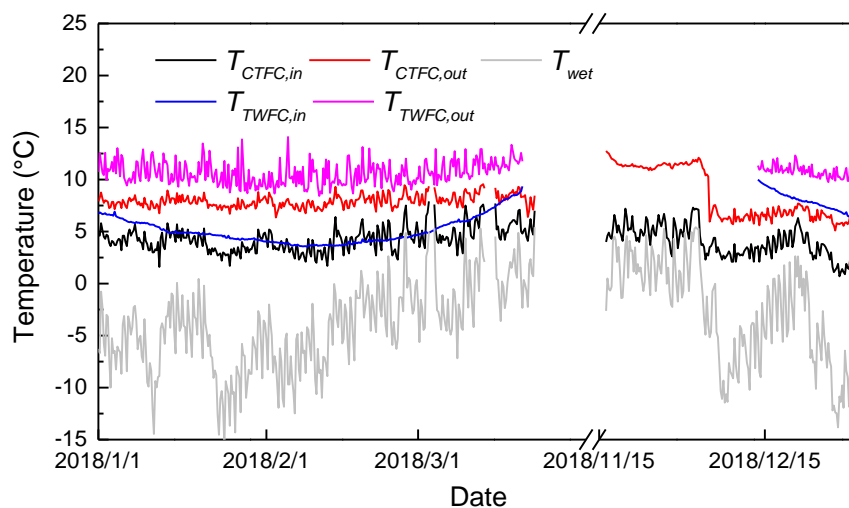
### 316 **3. Results and analysis**

#### 317 **3.1 Features of the cooling sources**

##### 318 1) Water supply temperature

319 The inlet and outlet water temperatures of heat exchangers in different free cooling  
 320 loops are shown in Figure 2. The varying outdoor wet-bulb temperature led to an  
 321 unstable supply water temperature of cooling towers (standard deviation of 0.7 °C).  
 322 The outlet water temperature of the cooling tower free cooling system was relatively

323 stable with a standard deviation of 0.4 °C. The variation of inlet tap water temperature  
 324 was relatively stable (standard deviation of 0.1 °C) since the tap water loops buried in  
 325 the soil had less impact from short-term outdoor temperature variations. The lowest  
 326 tap water inlet temperature occurred in February. However, the outlet water  
 327 temperature of the tap water-based free cooling system fluctuated significantly  
 328 (standard deviation of 0.9 °C). It is because the tap water flow rate fluctuates  
 329 significantly with the user's demand for ultrapure water. Compared With the cooling  
 330 tower free cooling, the tap water-based free cooling system outlet temperature was  
 331 approximately 2.0 °C higher, resulting in a smaller cooling capacity and fewer operating  
 332 hours.

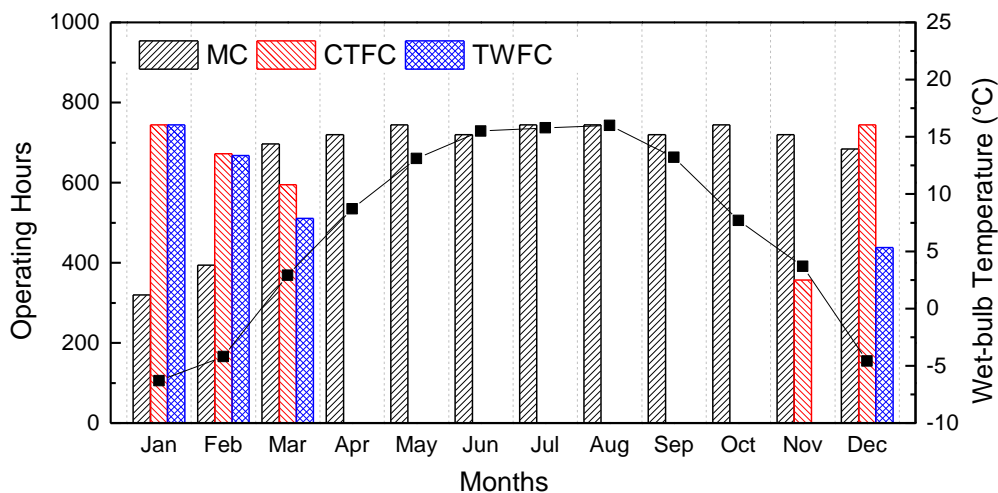


333  
 334 Figure 2: The inlet and outlet water temperature of the two free cooling systems

335 2) Operating hours

336 The operating hours of different cooling systems in each month are shown in Figure 3.  
 337 The cooling tower free cooling system started to work in mid-November and ended in  
 338 late March. There were 3,248 operating hours of cooling tower free cooling, accounting  
 339 for 37% of a year. The tap water-based free cooling system started later and ended  
 340 slightly earlier than the cooling tower free cooling system due to the slightly higher  
 341 water supply temperature. There were 2,361 hours, nearly 27% of a year, that the tap  
 342 water could be employed as a cooling source. The cooling capacity and the chilled  
 343 water supply temperature of the free cooling system vary with outdoor conditions and  
 344 tap water flow rate. When the cooling capacity of free cooling systems was insufficient,

345 chillers were employed to provide the remaining cooling demand and maintain the  
 346 chilled water supply temperature. Due to the great cooling demand and strict  
 347 temperature control of end-users, there were only 57% hours in January and 41%  
 348 hours in February when all chillers were turned off. In summer, the mechanical cooling  
 349 system comprised of medium-temperature chillers and low-temperature chillers was in  
 350 operation all the time.



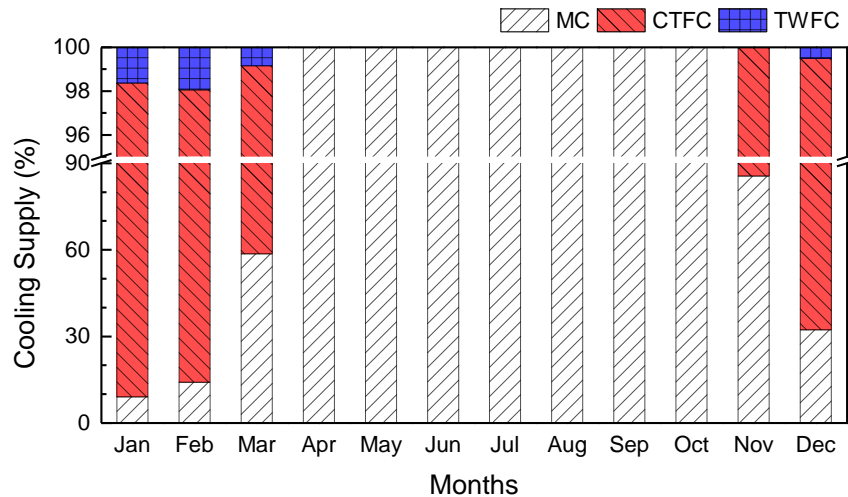
351

Figure 3: Operating hours of different cooling systems

352

### 353 3) Cooling supply

354 The cooling load distribution of different cooling systems is shown in Figure 4. In  
 355 January, February, and December, free cooling was the dominant cooling method  
 356 which provided 91%, 86%, and 68% of the total cooling demand, respectively. The  
 357 proportion of tap water-based free cooling was very small, approximately 2%, limited  
 358 by its small design capacity. In November and March, both mechanical cooling and  
 359 free cooling were important to satisfy the cooling demand of users. In other months,  
 360 the mechanical cooling system removed all the sensible and latent cooling loads.



361

362

Figure 4: Cooling load distribution of the central cooling system

### 363 3.2 Energy performance

#### 364 1) COPs

365 The COPs of the different cooling methods are shown in Figure 5. In winter, the COP  
 366 of the mechanical cooling system achieved an average value of 6.3. In other months,  
 367 the COP for mechanical cooling decreased with the increase of outdoor wet-bulb  
 368 temperature and reached the lowest value of 4.3 in July. The COPs of the two free  
 369 cooling methods were significantly greater than those of the mechanical cooling  
 370 method, especially the tap water-based free cooling. For cooling tower free cooling,  
 371 the COP of the system ranged from 12.7 to 22.8 and reached its maximum COP in  
 372 January since the lower outdoor wet-bulb temperature helped to reduce the energy  
 373 consumption of the cooling towers. The COP of the tap water-based free cooling  
 374 system ranged from 26.5 to 62.4, and its maximum COP occurred in February,  
 375 corresponding to the lowest tap water temperature. In comparison, the COP of tap  
 376 water-based free cooling was approximately 7.4-fold higher than the mechanical  
 377 cooling system and 2.2-fold higher than the cooling tower free cooling system. It is  
 378 because that the cold tap water is a free natural cooling source, and no energy-  
 379 consuming equipment is required for cold water production. However, cooling towers  
 380 and/or chillers were required for the cooling tower free cooling and mechanical cooling  
 381 systems to produce chilled water.



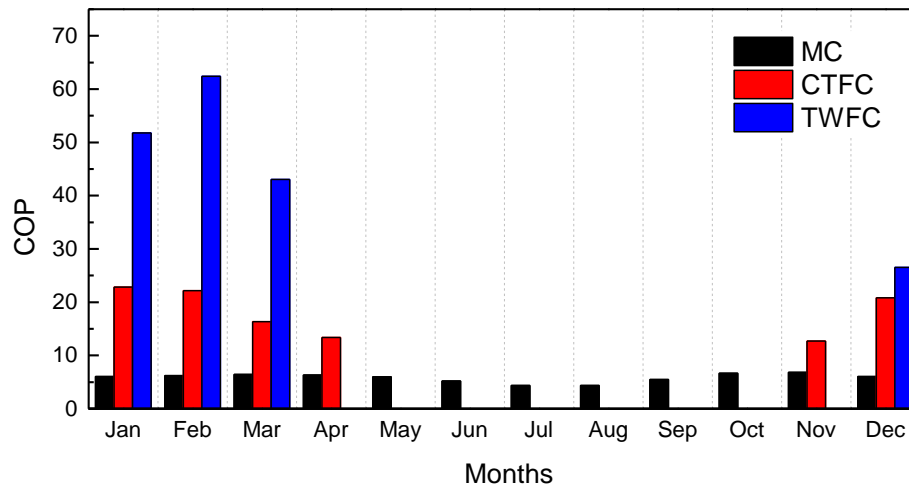


Figure 5: The COPs of different cooling systems

382

383

### 384 3) Energy savings

385 According to Equations (6) to (8), the energy savings of two free cooling methods were  
 386 evaluated, as shown in Figure 6. Compared with mechanical cooling, the electricity

387 savings of the cooling tower free cooling system ranged from 139.4 MWh to 1860 MWh

388 in winter months, as shown in Figure 6 (a), and the energy-saving rates ranged from

389 49.9% to 72.3%. Additional to the electricity savings, tap water-based free cooling

390 could reduce the natural gas consumption for tap water preheating. The natural gas

391 savings could be converted to equivalent electricity savings according to Equation (8).

392 The total electricity savings of the tap water-based free cooling system were ranged

393 from 113.5 MWh to 416.5 MWh energy, as shown in Figure 6 (b). The energy savings

394 on the heating side (natural gas reduction) was much larger than they were on the

395 cooling side since the energy efficiency of the cooling system was normally higher than

396 that of the direct heating system. The monthly electricity-saving rate on the cooling

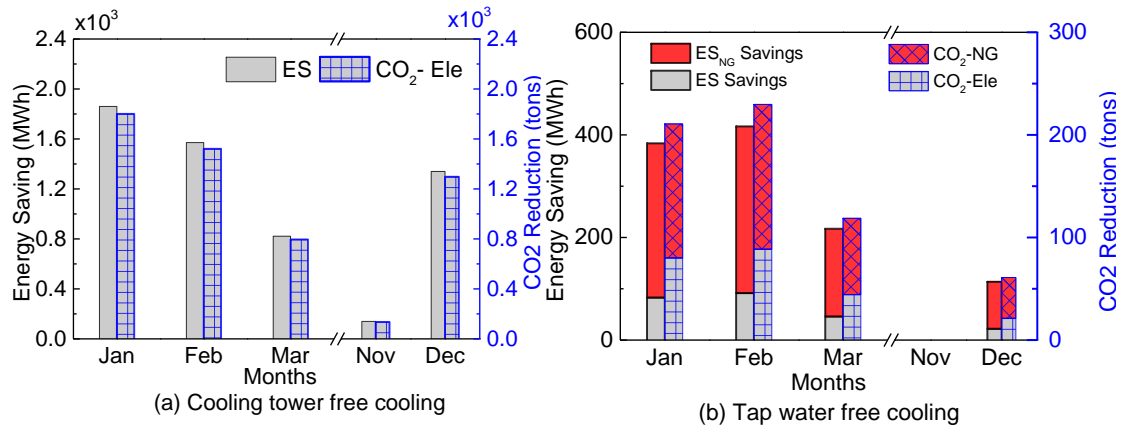
397 side ranged from 76.3% to 89.9%. Compared with cooling tower free cooling, the tap

398 water-based free cooling method showed a higher energy-saving rate. In summary,

399 the total electricity savings of two free cooling methods were 6,044 MWh, and the

400 corresponding annual energy-saving rate was 15.1% (cooling side only). The annual

401 total natural gas savings were  $1.48 \times 10^5$  kg.



402 Figure 6: Energy savings and CO<sub>2</sub> reduction of two free cooling methods

403 **3.3 CO<sub>2</sub> emission reduction**

404 Figure 6 also shows the CO<sub>2</sub> emission reductions of two free cooling methods. The  
 405 annual CO<sub>2</sub> emission reductions were 5,616 tons for the cooling tower free cooling  
 406 system and 620 tons for the tap water-based free cooling system. The CO<sub>2</sub> emission  
 407 reduction from the cooling tower free cooling system achieved its peak value in  
 408 January along with its highest energy savings. For tap water-based free cooling, the  
 409 CO<sub>2</sub> emission reduction reached its peak value in February, which also aligned with its  
 410 energy savings. The CO<sub>2</sub> emission reductions from natural gas savings were greater  
 411 than those from the electricity savings, as shown in Figure 6 (b). With a large cooling  
 412 capacity, the cooling tower free cooling system reduced a greater amount of CO<sub>2</sub>  
 413 emissions than the tap water-based free cooling system. However, the CO<sub>2</sub> emission  
 414 reduction rate of tap water-based free cooling system (0.44 ton/MWh) was 4.4-fold  
 415 higher than that of the cooling tower free cooling (0.10 ton/MWh), indicating more  
 416 environmentally friendly.

417 **3.4 Economic analysis**

418 The payback periods of both free cooling methods were analyzed according to  
 419 Equations (12) to (14). As analyzed in Section 3.2, both free cooling methods achieved  
 420 significant energy savings. However, additional heat exchangers and pumps are  
 421 required, thus increasing initial costs. The initial costs were estimated by multiplying  
 422 the purchase cost of equipment by an installation factor to cover all piping, sensors,

423 electrical installations, foundations, and enclosures, and the installation factor of 3 is  
 424 used in this study according to a similar economic analysis of the free cooling system  
 425 in [41]. This results in the installed cost of  $7.14 \times 10^6$  RMB for the cooling tower free  
 426 cooling system and  $9.45 \times 10^5$  for tap water-based free cooling system, as listed in Table  
 427 4. The initial cost of cooling towers was excluded since cooling towers were also  
 428 required in the mechanical cooling system in summer. The operation cost savings was  
 429 the energy cost savings minus maintenance and repair costs of the system. However,  
 430 the major maintenance tasks for heat exchangers and pumps are mainly performed at  
 431 relatively low frequency, such as quarterly, semiannually, or annually, according to  
 432 ASHRAE standard 180-2018 [42]. In addition, the free cooling systems only operate in  
 433 winter, which would help to reduce the maintenance and repair cost further. Thus, the  
 434 maintenance and repair costs were neglected in estimating operation cost, as defined  
 435 in Equation (14). The operation cost savings are  $3.54 \times 10^6$  RMB and  $6.90 \times 10^5$  RMB for  
 436 the cooling tower free cooling and the tap water-based free cooling system,  
 437 respectively. The payback period of tap water-based free cooling was 1.4 years, which  
 438 was slightly shorter than the cooling tower free cooling system (2 years) due to higher  
 439 COP on the cooling side and natural gas reductions on the heating side. A shorter  
 440 payback period makes the tap water-based free cooling method an attractive solution  
 441 for improving the energy efficiency of central cooling systems.

442 Table 4. Parameters used in the economic analysis

Items	CTFC	TWFC
Heat exchangers (RMB)	$1.25 \times 10^6$	$2.5 \times 10^5$
Pumps (RMB)	$1.13 \times 10^6$	$6.5 \times 10^4$
Installed Cost (RMB)	$7.14 \times 10^6$	$9.45 \times 10^5$
Electricity Saving (MWh)	5802	242
Natural gas savings ( $\text{m}^3$ )	-	$2.06 \times 10^5$
Electricity price (RMB/kWh)	0.61	0.61
Natural gas (RMB/ $\text{m}^3$ )	-	2.63
Operation Cost Saving (RMB)	$3.54 \times 10^6$	$6.90 \times 10^5$
PBP (years)	2.0	1.4

#### 443 4. Discussion

444 In this study, the potential of a tap water-based free cooling method was explored, and

445 its performance was compared with the widely used cooling tower free cooling system.  
446 According to the analysis above, the tap water-based free cooling shows good  
447 performance with higher energy efficiency, greater CO<sub>2</sub> reduction rate, and shorter  
448 payback period. These features make the tap water-based free cooling system an  
449 attractive solution for reducing energy consumption of the cooling system. However,  
450 the variation of tap water flow rate leads to fluctuations in the supply water temperature,  
451 complicating system control and operation. The fluctuation in the supply temperature  
452 can be mitigated by controlling the flow rate through the exchangers, such as a three-  
453 way control valve. Furthermore, the cooling capacity of the tap water-based free  
454 cooling system may be insufficient due to the limited tap water flow rate, necessitating  
455 the use of other cooling sources to compensate for the remaining cooling demand.  
456 There is considerable water consumption in industrial and domestic uses. For example,  
457 China's industrial water consumption was 126.16 billion m<sup>3</sup> and 85.99 billion m<sup>3</sup> for  
458 domestic uses in 2019, accounting for 21.0% and 14.3% of the national water  
459 consumption, respectively [43]. When the city water network in a district or larger scale  
460 is used as the natural cooling source, the cooling capacity will be greater and more  
461 stable. The proposed free cooling system also could reduce the heat demand of  
462 downstream users, and thus, considerable energy-saving potentials exist.

## 463 **5. Conclusions**

464 This study proposed a tap water-based free cooling system and explored its energy-  
465 saving potential. The proposed tap water-based free cooling system integrated with a  
466 cooling tower free cooling and mechanical cooling system was applied in a  
467 semiconductor manufacturing factory in Tianjin. The performance of the tap water-  
468 based free cooling system was compared with the widely used cooling tower free  
469 cooling from thermodynamic, energy, environmental, and economic perspectives. The  
470 following conclusions can be founded.

- 471 1) Compared with the cooling tower free cooling, the inlet temperature of the tap  
472 water-based free cooling system was more stable with a smaller standard deviation  
473 of 0.1 °C. However, its outlet water temperature fluctuated significantly with a

474 standard deviation of 0.9 °C due to the tap water flow rate variation. The operating  
475 hours of the tap water-based free cooling system accounted for 27% of a year,  
476 which was 10% shorter than the cooling tower free cooling system due to slightly  
477 higher outlet water temperature.

478 2) From the energy analysis, both free cooling systems showed significantly higher  
479 energy efficiency than the mechanical cooling system, especially the tap water-  
480 based free cooling system. The COP of the tap water-based free cooling system  
481 was approximately 7.4-fold higher than that of the mechanical cooling system and  
482 2.2-fold higher than that of the cooling tower free cooling system. Using these two  
483 free cooling methods together saved 6,044 MWh electricity for cooling and  
484 achieved an annual energy-saving rate of 15.1%. In addition, the tap water-based  
485 free cooling system saved  $1.48 \times 10^5$  kg of natural gas for ultrapure water preheating.

486 3) Energy reductions from the two free cooling systems reduced 6,236 tons of CO<sub>2</sub>  
487 emissions in a year. The CO<sub>2</sub> emission reduction rate of the tap water-based free  
488 cooling system was 4.4-fold higher than that of the cooling tower free cooling  
489 system, indicating that the tap water-based free cooling method was more  
490 environmentally friendly.

491 4) From the economic analysis, the tap water-based free cooling showed a shorter  
492 payback period, which makes it an attractive solution for improving the central  
493 cooling system efficiency.

## 494 **Acknowledgements**

495 This work was supported by the China National Key Research and Development  
496 Program [Grant No. 2018YFC0705203]; the Tianjin Science and Technology  
497 Commission [Grant No.18ZXQSF00070]; and the China Scholarship Council [No.  
498 201806250235].

## 499 **References**

- 500 [1] Yang, L., Yan, H. and Lam, J.C. Thermal comfort and building energy consumption  
501 implications – A review. Applied Energy, 2014. 115: p. 164-173.  
502 [2] 2019 global status report for buildings and construction: Towards a zero-emission, efficient

- 503 and resilient buildings and construction sector. 2019, Global Alliance for Buildings and  
504 Construction, International Energy Agency and the United Nations Environment  
505 Programme.
- 506 [3] Li, D.H.W., Yang, L. and Lam, J.C. Zero energy buildings and sustainable development  
507 implications – A review. *Energy*, 2013. 54: p. 1-10.
- 508 [4] Clift, R. Climate change and energy policy: The importance of sustainability arguments.  
509 *Energy*, 2007. 32(4): p. 262-268.
- 510 [5] Kircher, K., et al. Cleanroom energy efficiency strategies: Modeling and simulation. *Energy*  
511 and Buildings, 2010. 42(3): p. 282-289.
- 512 [6] Chang, C., et al. Various Energy-Saving Approaches to a TFT-LCD Panel Fab.  
513 *Sustainability*, 2016. 8(9): p. 907-916.
- 514 [7] Chien, C., Peng, J. and Yu, H. Building energy saving performance indices for cleaner  
515 semiconductor manufacturing and an empirical study. *Computers & Industrial Engineering*,  
516 2016. 99: p. 448-457.
- 517 [8] Hu, S.C. and Chuah, Y.K. Power consumption of semiconductor fabs in Taiwan. *Energy*,  
518 2003. 28: p. 895–907.
- 519 [9] Lee, S., et al. Application of an energy management system in combination with FMCS to  
520 high energy consuming IT industries of Taiwan. *Energy Conversion and Management*,  
521 2011. 52(8-9): p. 3060-3070.
- 522 [10] IES, ANSI and ASHRAE. Energy standard for buildings except low-rise residential buildings,  
523 in Standard 90.1-2019. 2019.
- 524 [11] Zhang, H., et al. Free cooling of data centers: A review. *Renewable and Sustainable*  
525 *Energy Reviews*, 2014. 35: p. 171-182.
- 526 [12] Daraghmeh, H.M. and Wang, C. A review of current status of free cooling in datacenters.  
527 *Applied Thermal Engineering*, 2017. 114: p. 1224-1239.
- 528 [13] Dai, J., Das, D. and Pecht, M. A multiple stage approach to mitigate the risks of  
529 telecommunication equipment under free air cooling conditions. *Energy Conversion and*  
530 *Management*, 2012. 64: p. 424-432.
- 531 [14] Dai, J., Das, D. and Pecht, M. Prognostics-based risk mitigation for telecom equipment  
532 under free air cooling conditions. *Applied Energy*, 2012. 99: p. 423-429.
- 533 [15] Taylor, S.T. How to design & control waterside economizers. *ASHRAE Journal*, 2014. 56(6):  
534 p. 30-36.
- 535 [16] Habibi Khalaj, A. and Halgamuge, S.K. A review on efficient thermal management of air-  
536 and liquid-cooled data centers: From chip to the cooling system. *Applied Energy*, 2017.  
537 205: p. 1165-1188.
- 538 [17] Schneider, R.K. Designing cleanroom HVAC system. *ASHRAE Journal*, 2001(43): p. 39-  
539 46.
- 540 [18] Clidaras, J., Stiver, D.W. and Hamburgren, W. Water-based data center. 2007.
- 541 [19] Ling, L., et al. Energy saving analysis of the cooling plant using lake water source base on  
542 the optimized control strategy with set points change. *Applied Thermal Engineering*, 2018.  
543 130: p. 1440-1449.
- 544 [20] Sean M. James, O. and Rubenstein, B.A. Renewable energy-based datacenter cooling.  
545 2013.
- 546 [21] Benzaama, M.H., et al. Experimental and numerical analysis of the energy performance of

547 a water/soil exchanger coupled to a cooling floor for North Africa. *Geothermics*, 2019. 80:  
548 p. 8-19.

549 [22] Niemann, J., Bean, J. and Avelar, V. Economizer Modes of Data Center Cooling Systems  
550 White Paper 132. 2011, Schneider, Inc.

551 [23] Agrawal, A., Khichar, M. and Jain, S. Transient simulation of wet cooling strategies for a  
552 data center in worldwide climate zones. *Energy and Buildings*, 2016. 127: p. 352-359.

553 [24] Chu, L., et al. High performance cleanrooms - A design guidelines sourcebook. 2006,  
554 Pacific Gas And Electric Company.

555 [25] She, X., et al. Energy-efficient and -economic technologies for air conditioning with vapor  
556 compression refrigeration: A comprehensive review. *Applied Energy*, 2018. 232: p. 157-  
557 186.

558 [26] Ji, J., et al. Domestic air-conditioner and integrated water heater for subtropical climate.  
559 *Applied thermal engineering*, 2003. 23(5): p. 581-592.

560 [27] Ji, J., et al. Performance of multi-functional domestic heat-pump system. *Applied Energy*,  
561 2005. 80(3): p. 307-326.

562 [28] JIANG, H., et al. An experimental study on a modified air conditioner with a domestic hot  
563 water supply (ACDHWS). *Energy*, 2006. 31(12): p. 1789-1803.

564 [29] Xiaowen, Y. and Lee, W.L. The use of helical heat exchanger for heat recovery domestic  
565 water-cooled air-conditioners. *Energy Conversion and Management*, 2009. 50(2): p. 240-  
566 246.

567 [30] Chen, H. and Lee, W.L. Combined space cooling and water heating system for Hong Kong  
568 residences. *Energy and Buildings*, 2010. 42(2): p. 243-250.

569 [31] Jia, J. and Lee, W.L. Applying storage-enhanced heat recovery room air-conditioner  
570 (SEHRAC) for domestic water heating in residential buildings in Hong Kong. *Energy and*  
571 *Buildings*, 2014. 78: p. 132-142.

572 [32] Jia, J. and Lee, W.L. Experimental investigations on using phase change material for  
573 performance improvement of storage-enhanced heat recovery room air-conditioner.  
574 *Energy*, 2015. 93: p. 1394-1403.

575 [33] Zhang, X., et al. Experimental research on condensing heat recovery using phase change  
576 material. *Applied Thermal Engineering*, 2011. 31(17-18): p. 3736-3740.

577 [34] Den, W., Chen, C. and Luo, Y. Revisiting the water-use efficiency performance for  
578 microelectronics manufacturing facilities: Using Taiwan's Science Parks as a case study.  
579 *Water-Energy Nexus*, 2018. 1(2): p. 116-133.

580 [35] Villard, A., Lelah, A. and Brissaud, D. Drawing a chip environmental profile: environmental  
581 indicators for the semiconductor industry. *Journal of Cleaner Production*, 2015. 86: p. 98-  
582 109.

583 [36] Hu, S., et al. Power consumption benchmark for a semiconductor cleanroom facility system.  
584 *Energy and Buildings*, 2008. 40(9): p. 1765-1770.

585 [37] Code for design of pure water system of electronic industry, in GB50685-2011. 2011, China  
586 Planning Press: Beijing.

587 [38] Kozicki, M. *Cleanrooms facilities and practices*. 1991: Springer Netherlands.

588 [39] ASHRAE. *Design guide for cleanrooms fundamentals, systems, and performance*. 2017:  
589 W. Stephen Comstock.

590 [40] Deymi-Dashtebayaz, M., Valipour Namanlo, S. and Arabkoohsar, A. Simultaneous use of

591 air-side and water-side economizers with the air source heat pump in a data center for  
592 cooling and heating production. Applied Thermal Engineering, 2019. 161: p. 114133.  
593 [41] Jaramillo, R., et al. Simulation assessment of free-cooling technology for a large campus.  
594 ASHRAE Transactions, 2015. 121: p. 471-486.  
595 [42] ASHRAE. Standard Practice for Inspection and Maintenance of Commercial Building  
596 HVAC Systems, in 180-2018. 2018, ASHRAE and ACCA.  
597 [43] China, N.B.O.S. China Statistical Yearbook 2019. 2019, China Statistics Press.  
598