

# Modelling of laminar diffusion flames with biodiesel blends and soot formation

Anxiong Liu<sup>a,b</sup>, Zhan Gao<sup>c,\*</sup>, Stelios Rigopoulos<sup>b</sup>, Kai H. Luo<sup>a,\*</sup>, Lei Zhu<sup>c</sup>

<sup>a</sup>Department of Mechanical Engineering, University College London, London WC1E 7JE, United Kingdom

<sup>b</sup>Department of Mechanical Engineering, Imperial College London, Exhibition Road, London SW7 2AZ, UK

<sup>c</sup>Key Laboratory for Power Machinery and Engineering of M.O.E, Shanghai Jiao Tong University, Shanghai, China

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## Abstract

Modelling of soot formation and oxidation of biodiesel fuels is a challenge, due to the complexity of the chemical reactions and soot formation pathways. In this paper, the discretised population balance approach and a PAH-HACA based detailed soot model have been coupled with an in-house CFD code to simulate a laminar diffusion flame with blends of different components of oxygenated biodiesel fuel, and employed to predict combustion performance and soot formation. The simulation aims to reproduce a target experiment which investigated the effects of dibutyl ether (DBE) addition to the biodiesel surrogate (methyl decanoate, MD) by increasing the the mole fraction of DBE from 0 to 40%. A combined and reduced MD-DBE-PAH mechanism developed from three sub-mechanism branches has been employed in the simulation. The simulation results show that temperature rises as the DBE percentage increases to 40%. The swallow-tail shape of the soot occurrence zone and the quantity of soot production are correctly predicted. Regarding the effect of soot suppression, the model has basically captured the reducing tendency of soot formation in the measurements as the DBE addition increases from 0% to 40%. Concentrations of PAHs and C3 species contributing to the formation of aromatic rings are slightly reduced due to the addition of DBE, which is a leading cause of soot suppression. However, on the whole, the numerical solution featured much smaller differences than those observed in the experiment among laminar flames with different MD/DBE ratios, because the combined MD-DBE-PAH mechanism only present a slight difference of concentrations of soot precursor species.

*Keywords:* biodiesel, methyl decanoate, dibutyl ether, soot suppression, detailed soot model, sectional population balance model

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## 1. Introduction

Biodiesel fuel is regarded as renewable energy source in the pathway of going carbon neutral. Vegetation plants, the source of biodiesel, capture the same amount of CO<sub>2</sub> through photosynthesis when growing as is released when biomass is burned. Biodiesel fuel can be blended with petrodiesel in any proportions so that it is compatible with existing diesel engines and distributed combustion infrastructure. The dependence on fossil fuel is reduced as biodiesel fuel can be grown, produced and distributed. The utilisation of biodiesel can also reduce the emissions of total particulate matters, carbon monoxide and unburned hydrocarbons, compared to conventional petro/diesel fuel. Increasingly stringent environmental regulations relating to such emissions, therefore, have prompted a number of study on PM emissions from the combustion of biodiesel and its blends in diesel engines in well-controlled lab-scale flames and reactors [1–8] and diesel engines [9–15].

Most of experimental studies have demonstrated that the addition of biodiesel content has notably decreased PM emissions, compared to long-chain hydrocarbon fuels. For example, Kholghy et al. [4] report that soot formation of a biodiesel surrogate (50%/50% molar ratio of methyl-octanoate/n-decane blends) is lower than n-decane, 1-decene and 5-decene in laminar coflow diffusion flames. Muelas et al. [6] examined in the droplet combustion that a UCO biodiesel (chemical formula C<sub>17.32</sub>H<sub>32.41</sub>O<sub>1.9</sub>) and its blends display a much lower propensity to soot formation than

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\*Corresponding authors

Email addresses: gaozhan@sjtu.edu.cn (Zhan Gao), k.luo@ucl.ac.uk (Kai H. Luo)

neat hexadecane ( $C_{16}H_{34}$ ), a major composition of conventional diesel. Iannuzzi et al. [16] and Abboud et al. [3] report that the reduction of soot emissions strongly correlates to the fuel's oxygen content if oxygenated fuel is used. Desjardins et al. [17] has proved the reduction of sooting propensity depends strongly on the oxygenated functional groups: ether, alcohol and ester are in a descending order for soot reduction on the base fuel of pentane ( $C_5H_{12}$ ). Knothe et al. [18] and Merchan-merchan et al. [2] have concluded that the amount of aromatic compounds are much smaller in biodiesel fuel than in regular diesel, and therefore biodiesel fuel can suppress the soot propensity. However, the existence of unsaturated bonds in the biodiesel molecules adversely enhance the soot formation. For example, Jiang et al. [19] shows the MC ( $C_5H_8O_2$ , an oxygenated unsaturated ester) blends form about double the amount of soot production than pure  $C_2H_4$  in a laminar coflow flame. Kholghy et al. [4] specifically reports centrally located unsaturation (C=C bond) significantly increases soot formation. McEnally and Pfefferle [20] discussed that doping oxygenateds, e.g., ethanol into ethylene causes a synergistic effect for the promotion of  $C_3$  chemistry.

However, only a very small number of modelling studies have been performed for soot formation with biodiesel fuel. Ni et al. [21] employs an empirical non-precursor model to estimate the soot volume fraction with the filter smoke number (FSN), based on the general purpose CFD software AVL-Fire for diesel engine tests. Arad et al. [22] employed an empirical non-precursor model for the soot nucleation process. Some numerical studies have employed semi-empirical precursor models, correlating precursor species with soot formation processes. For example, An et al. [23] and E et al. [24] assume that the soot nucleation process is a one-step transition from  $C_6H_2$  (a long-chain acetylene) and  $A_2R_5$  (acenaphthylene).  $C_2H_2$  (acetylene) is assumed as the precursors for the soot nucleation process [10, 25–28] or surface growth [22, 29, 30], modelled by a chemical reaction with an Arrhenius expression. Some studies also simplify gaseous chemical mechanisms to estimate soot precursor species. For instance, Wang et al. [10] models the formation of soot precursor species ( $C_2H_2$ ) using a one-step pyrolysis reaction of fuel. Arad et al. [22] employs twelve reactions to model the pyrolysis of a blend of aromatic fuel, aliphatic fuel and ester fuel. In terms of numerical approaches, the two-equation transport approach has been mainly used to solve for two primary variables of soot: the number density / mass fraction of soot [26–28, 31] or the nuclei concentration / mass fraction of soot [22]. In general, soot formation modelling for the combustion of biodiesel fuel (including diesel [32]), is still at the stage of applying empirical/semi-empirical soot kinetics and affordable numerical methods, in the context of complex chemistry and turbulent flows in practical diesel engines.

On the other hand, soot models for the benchmark flames of the International Sooting Flames (ISF) workshop are very detailed and comprehensive, although it is still difficult to obtain quantitative predictions, particularly for turbulent flames. These benchmark flames, which are of concern to the majority of researchers in the soot formation modelling community, involve primarily lightweight hydrocarbon fuel: *i.e.*  $CH_4$  [33],  $C_2H_4$  [34],  $C_2H_6$  [35] in lab-scale conditions, *i.e.* premixed flames, coflow flames and pressurised swirled flames. The soot models applied in benchmark flames, generally include detailed gaseous chemical mechanisms with PAHs pathways, detailed soot processes and aerosol dynamics. Soot processes usually include nucleation (the dimerisation of PAHs [36, 37]), surface growth (HACA mechanism [38] or ARS aromatic site model [39], adsorption of PAHs), multi-regime coagulation/agglomeration [40] and sintering [41], and also the morphology of soot particles. Numerical approaches, including methods of moment, discretisation methods, and Monte Carlo methods, have been implemented to solve properties of soot particles, *i.e.* particle size distribution, the morphology of soot aggregates.

Several challenges need to be identified and addressed in order to close the gap of soot formation models between biodiesel flames and the ISF benchmark flames. First of all, biodiesel fuel is sourced from a variety of vegetation plants and is diverse in terms of chemical components. Researchers have investigated the soot emissions of soybean biodiesel [42], rapeseed biodiesel [43], cooking oil [44], palm oil and rice bran [45]. Generally, the main components of biodiesel are saturated and unsaturated methyl esters with long alkyl chains. A large number of these heavy-molecular components have to be specified by a few species of similar molecular structures. For example, the soybean biodiesel can be modelled as two components - methyl butanoate ( $C_5H_{10}O_2$ ) and n-heptane ( $C_7H_{16}$ ) by Brakora et al. [46], or as three components - methyl decanoate ( $C_{11}H_{20}O_2$ ), 1,4-hexadiene ( $C_6H_{10}$ ) and n-dodecane ( $C_{12}H_{26}$ ) by Yu et al. [47], or as four components - methyl decanoate, n-hexadecane ( $C_{16}H_{34}$ ), methyl trans-3-hexenoate ( $C_7H_{12}O_2$ ) and 1,4-hexadiene [48]. As a result, a large variety of biodiesel fuel and the uncertainty of chemical compositions lead to a difficulty in formulating benchmark flames for biodiesel combustion and emissions.

Second, researchers prefer to study the blending of different types of biodiesel fuel to optimise the combustion performance and emission reduction. Therefore, different sub-mechanisms of reaction kinetics and chemical/physical properties should be considered in order to form a detailed chemical mechanism with soot precursor species. Errors

occur when the connections of different chemical databases are not well coordinated, particularly when the chemical concentrations of components cannot be fully validated in the measurements. This viewpoint will be elaborated later in this article.

Third, detailed soot formation models with a series of parameters have been calibrated for ISF benchmark flames with lightweight hydrocarbon fuels. The more elaborate a soot model is, the more parameters should be calibrated, *i.e.* the collision efficiency of the PAH dimerisation for the nucleation process [36], the active site ratio in the HACA mechanism [37] or the ARS aromatic site model [49] for the surface growth rate, the particle sintering time [41] and the morphology description of soot aggregates [49, 50]. Some of the sub-models are based on soot precursor species (*i.e.*,  $C_2H_2$ , PAH), and very sensitive to the gaseous chemical mechanisms [50].

Prior to this study, researchers have explored co-flow diffusion flames, using the methane/ethylene base added with the biodiesel fuel [51–53], or using gasoline / diesel / long-chain hydrocarbons added with biodiesel fuel [45, 54, 55]. The added biodiesel fuel was basically a pure component of alcohols, esters or ethers. The aim of the article is to model the soot formation in flames with the blended fuel of different oxygenated surrogates using detailed soot kinetics. The target experiment in this study was conducted by Gao et al. [5]. In the experiment, dibutyl ether ( $C_8H_{18}O$ , DBE) was added to the base fuel of methyl decanoate ( $C_{11}H_{22}O_2$ , MD), a major component in many types of biodiesel fuel. When the mole fraction of DBE in the fuel stream rises from 0% to 40%, the soot formation is largely suppressed. The main task of this paper is to employ a recently developed conservative method for solving the population balance equation [56], coupled with an in-house CFD code (BOFFIN) [57] with a comprehensive chemical mechanism and transport properties, to predict soot formation in a laminar diffusion flame with biodiesel blends. A detailed soot model, of which the parameters are consistent with an application in one of the ISF benchmark flames, will be used in this simulation. The main challenge is to form a proper combined gas-phase chemistry including various necessary oxygenated surrogates and to propose a detailed soot kinetics to capture the soot suppression effect. Main error sources of the discrepancy between the simulations and target measurements will be offered.

## 2. Simulation and experiment setup

### 2.1. The Co-flow flame burner and simulation cases

In the experiment, the burner consisting of three concentric brass tubes is shown in Fig.1a. The central tube supplies the mixture of fuel: methyl decanoate (MD), dibutyl ether (DBE) and carrier gas  $N_2$ . The intermediate tube is used to supply co-flow air stream with the flow rate (8.43 L/min). The outer tube supplies  $N_2$  (16.2 L/min) as a shield to protect flames from the surrounding air. The inner diameters of three tubes are 10, 26 and 59 mm, respectively. The fuel and the burner were heated at 573 K to keep the fuel above the boiling point. Air and the protecting gas were preheated to 473 K. In the experiment, the carbon atom flow rates for all the fuel blends remain constant at 6 g/h. Compositions of the fuel stream in three flames are presented in Table 1.

Blended fuel	Components (mol %)		blend fuel [g/h]	Flow rate	
	MD	DBE		Carbon atom flow rates [g/h] / [mol/h]	carrier gas $N_2$ [L/h]
Pure MD	100	0	8.45		
70% MD	70	30	8.38	6.0 / 0.5	5.82 (473 K)
60% MD	60	40	8.35		

**Table 1:** The simulation cases: fuel components and mass flow rates

The experimental apparatus is completely described in [5]. Temperature profiles in flames were measured by the rapid thermocouple insertion technique and the soot volume fraction was performed by the 2D LII measurement, which were quantitatively calibrated against the high spatial resolution laser cavity extinction measurement [58].

### 2.2. Numerical methods for the reacting flow and soot formation

The population balance equation (PBE) is a rigorous way to solve the particle size distribution in aerosol dynamics. In the present article, the discretised PBE with a conservative finite volume scheme [56] considers the processes of

convection, thermophoresis, diffusion, nucleation, growth and coagulation, which is written as

$$\begin{aligned}
& \frac{\partial n(v; \mathbf{x}, t)}{\partial t} + \sum_{j=1}^3 \frac{\partial ((u_j + U_j^T)n(v; \mathbf{x}, t))}{\partial x_j} + \frac{\partial (G(v)n(v; \mathbf{x}, t))}{\partial v} \\
&= - \sum_{j=1}^3 \frac{\partial K_j(v; \mathbf{x}, t)}{\partial x_j} + \dot{B}(\mathbf{Y})\delta(v - v_0) \\
&+ \frac{1}{2} \int_0^v \beta(v - v', v')n(v - v', \mathbf{x}, t)n(v', \mathbf{x}, t) dv' - \int_0^\infty \beta(v, v')n(v, \mathbf{x}, t)n(v', \mathbf{x}, t) dv', \tag{1}
\end{aligned}$$

Here, the number density  $n(v; \mathbf{x}, t)$ , corresponds to the number of particles with volume in  $[v, v + dv]$  at the location  $x$  and time  $t$  per unit of physical space, referred as the particle size distribution. And  $G(v)$  is the combined surface growth and oxidation rate,  $\dot{B}$  represents the nucleation rate,  $v_0$  is the nuclei volume and  $\beta(v, v')$  denotes the coagulation kernel.  $K_j(v, \mathbf{x}, t)$ , moreover, represents the diffusive flux of number density along the  $j^{\text{th}}$  coordinate direction in physical space and  $\mathbf{U}^T$  represents the thermophoretic velocity. The soot kinetics will be described in Section 2.4.

The discretised PBE is coupled with the continuity, Navier-Stokes, energy and species transport equations to simulate combustion with soot formation. The transport equations of species and energy in the complete form are described as Eq. 2 and Eq. 3, respectively,

$$\frac{\partial \rho Y_k}{\partial t} + \frac{\partial}{\partial x_j} (\rho u_j Y_k) = \underbrace{\frac{\partial}{\partial x_j} \left[ \rho D_{km} \frac{\partial Y_k}{\partial x_j} \right]}_{\text{Ordinary diffusion}} + \dot{\omega}_k + \underbrace{\frac{\partial}{\partial x_j} \left[ \rho D_{km} Y_k \frac{\partial (\ln \bar{W})}{\partial x_j} \right]}_{\text{Molecular Weight}} + \underbrace{\frac{\partial}{\partial x_i} \left( D_k^T \frac{\partial (\ln T)}{\partial x_i} \right)}_{\text{Thermophoresis}} \tag{2}$$

where  $Y_k$  and  $X_k$  are the mass fraction and mole fraction of the species  $k$ , respectively.  $D_{km}$  is the molecular diffusion coefficient of species  $k$  in the multi-component mixture, while  $D_K^T$  is the thermal diffusion coefficient.  $W_k$  and  $\bar{W}$  are the molecular weight of species  $k$  and the local average molecular weight, respectively.

$$\begin{aligned}
& \frac{\partial \rho h}{\partial t} + \frac{\partial}{\partial x_j} (\rho u_j h) = \underbrace{\frac{\partial}{\partial x_j} \left( \rho D \frac{\partial h}{\partial x_j} \right) - \sum_k \frac{\partial}{\partial x_j} \left[ \rho (D - D_k) h_k \frac{\partial Y_k}{\partial x_j} \right]}_{\text{Ordinary differential diffusion}} \\
&+ \underbrace{\frac{\partial}{\partial x_j} \left[ \rho Y_k D_k h_k \frac{\partial (\ln W_k)}{\partial x_j} \right]}_{\text{Molecular Weight}} + \underbrace{\frac{\partial}{\partial x_j} \left[ \rho Y_k D_k^T h_k \frac{\partial (\ln T)}{\partial x_j} \right]}_{\text{Thermophoresis}} + \underbrace{\frac{\partial}{\partial x_j} \left( \sum_k \frac{RT}{W_k X_k} D_k^T \frac{\partial X_k}{\partial x_j} \right)}_{\text{Dufour effect}} + \dot{Q}_r \tag{3}
\end{aligned}$$

where  $h$  and  $h_k$  are the total enthalpy and specific enthalpy of species  $k$ , respectively.  $D$  is the heat transfer coefficient. All these thermodynamic and transport parameters are formulated in the same way as Chemkin Pro [59]. In order to account for the reduction of mixture enthalpy due to radiation  $\dot{Q}_r$  by gas phase molecules and soot, we include a radiation model based on the assumption of optical thinness [60]. Here, the temperature dependence of the Planck mean absorption coefficients is approximated by polynomials [61].

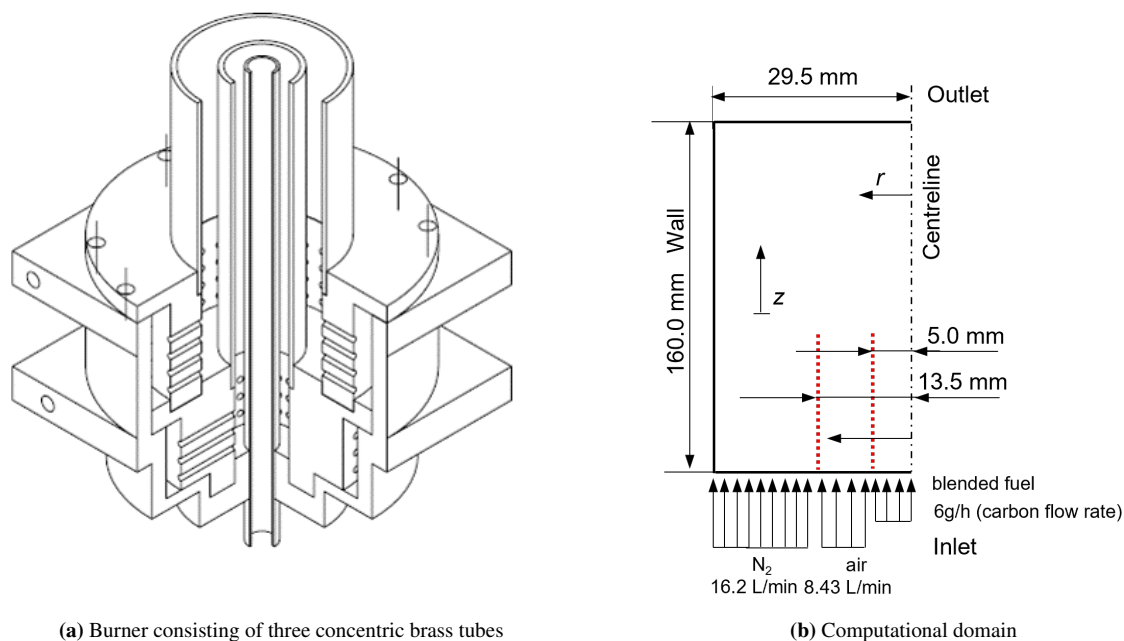
When the local average molecular weight does not vary significantly throughout the reacting flow field, the diffusion term with respect to molecular weight can be neglected. Meanwhile, the thermal diffusion coefficient  $D_k^T$  is also dependent on the difference of molecular weights between species so that thermophoresis term is also neglected when the local average molecular weight remains unchanged. In that situation, the convection term and diffusion term in regard to a scalar can coexist in the numerical implementation of an implicit scheme instead of many additional terms with an explicit scheme, thus improving the computational stability.

$$\frac{\partial \rho \phi_k}{\partial t} + \frac{\partial \rho u_j \phi_k}{\partial x_j} = \frac{\partial}{\partial x_j} \left[ \rho \Gamma_k \frac{\partial \phi_k}{\partial x_j} \right] \tag{4}$$

where  $\phi_k$  is any scalar and  $\Gamma_k$  is the diffusion coefficient.

However, in the simulation of laminar diffusion flames with biodiesel fuels, the local average molecular weight varies notably and the additional diffusion terms make a difference (see Appendix C): the mass diffusion term should be expressed in terms of mole fraction in the transport equations of species and energy. Therefore, the complete forms

are employed in this article (Eq. 2 and Eq. 3).



**Figure 1:** Configuration of the burner and the computational domain

### 2.3. Gaseous chemical mechanism

To gain insight into the process of soot formation in the laminar flame with blended fuels of MD and DBE, a comprehensive chemical mechanism including the combustion of MD and DBE, and PAHs formation is required. However, as the target experiment was the first attempt to study the combustion behaviours of MD-DBE mixture, a well-validated mechanism of MD-DBE cannot be found. In addition, PAHs pathways should also be incorporated in order to model soot formation. Therefore, three sub-mechanisms, respectively, of MD by Sarathy et al. [62], of DBE by Cai et al. [63], and of PAHs by Blanquart and Pitsch [36] will be combined. Another dilemma is faced that systematic measurements of concentrations of key gas species for the MD-DBE mixture combustion in the lab-scale reactors or flames do not exist, thus an alternative but convincing approach to validate the combined MD-DBE-PAH mechanism is proposed below. Here, simulation results of the MD mechanism [62] for the stoichiometric MD-air combustion and the DBE mechanism [63] for the stoichiometric DBE-air combustion in a perfect stirred reactor (PSR) are selected, as *ad-hoc* references for further validation, because both mechanisms have been validated in the respective experiments [62, 63].

The MD mechanism [62], consisting of 648 species and 2998 reactions, reproduces the behaviour of its original detailed mechanism in plugflow and stirred reactors for temperatures of 900 – 1800 K, equivalence ratios of 0.25 – 2.0, and pressures of 1 and 10 bar. This mechanism also well predicts the test data in opposed-flow diffusion flames. The oxidation kinetics of DBE Cai et al. [63], consisting of 328 species and 2236 reactions, was validated in the measurement of ignition delay times and laminar flame speeds in a laminar flow reactor. The sub-mechanism in [64], adopted in the soot modelling of ethylene diffusion flames [50], still accounts for the PAH pathways.

As both the full MD and DBE sub-mechanisms are comprised of a large number of species, they should be simplified for the flame simulation. Using the DRGEP (Directed Relation Graph with Error Propagation) method in the commercial software CHEMKIN PRO, the species shared by the MD, DBE, and PAH sub-mechanisms are selected as the target species for mechanism reduction. The relative tolerance errors for target species range from 10% - 20%, listed in Appendix A. Figure 2 compares the reduced mechanisms with the corresponding full mechanisms for the stoichiometric MD-air and DBE-air combustion, respectively, in a PSR

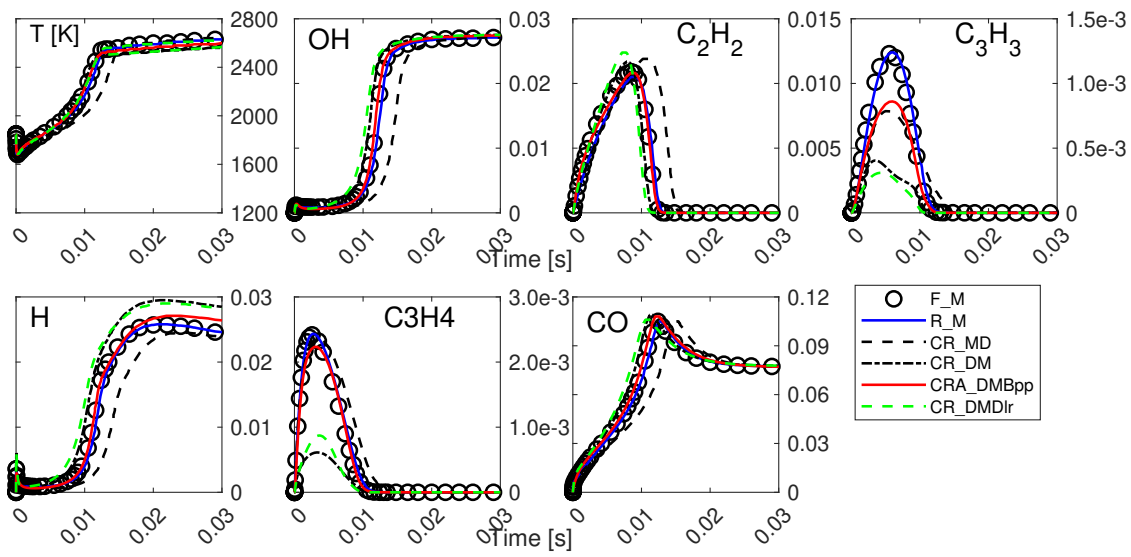
simulation. The mechanisms of full MD, full DBE, reduced MD and reduced DBE are marked as 'F\_M', 'F\_D', 'R\_M', 'R\_D'. It is clear that the reduced mechanisms have replicated the temperature profiles by the full mechanisms, which means ignition delay times are accurately predicted. Mole fractions of major species (OH, H, CO),  $C_2H_2$  and the minor species for PAHs formation ( $C_3H_3$ ,  $C_3H_4$ ) by the reduced mechanisms and full mechanisms coincide.

The combination of various chemical mechanisms is a complicated task. The common species with different names in the three mechanism branches are unified first. The most tricky part is to choose the chemical rate constants of common reactions and the thermochemical properties of common species from different sub-mechanisms. For example, the reduced mechanisms of MD and DBE are combined in two ways: (1) all the chemical rates and thermochemical properties from the MD reduced mechanism (R\_M) are maintained, combined with other reactions and species of the DBE reduced mechanism (R\_D); (2) on the opposite, the DBE reduced mechanism (R\_D) lays the foundation, other reactions and species from MD full mechanism (R\_M) are added in. These two combined mechanisms are labelled as CR\_MD and CR\_DM, respectively. However, Figure 2 also shows that neither of these combined mechanism is accurate in the PSR simulation in two separate stoichiometric conditions: MD-air mixture and DBE-air mixture. A major problem for the CR\_MD mechanism is that ignition delay times are longer in both conditions: the maximum temperature and peaks values of major species (OH,  $C_2H_2$ , H and CO) appear later than the full mechanisms. Another problem for both combined mechanisms is in the prediction of PAHs precursor species,  $C_3H_3$  and  $C_3H_4$ : in the MD-air combustion, the CR\_DM mechanism underestimated the PAH precursor species by about 2/3 of its standard values (prediction by the F\_M mechanism), while the CR\_MD mechanism performs much better, with a nearly accurate  $C_3H_4$  profile. On the opposite, in the DBE-air combustion, the CR\_DM mechanism obtains accurate profiles while the CR\_MD mechanism overpredicts the mole fractions of  $C_3H_3$  and  $C_3H_4$  by around 1 - 2 times.

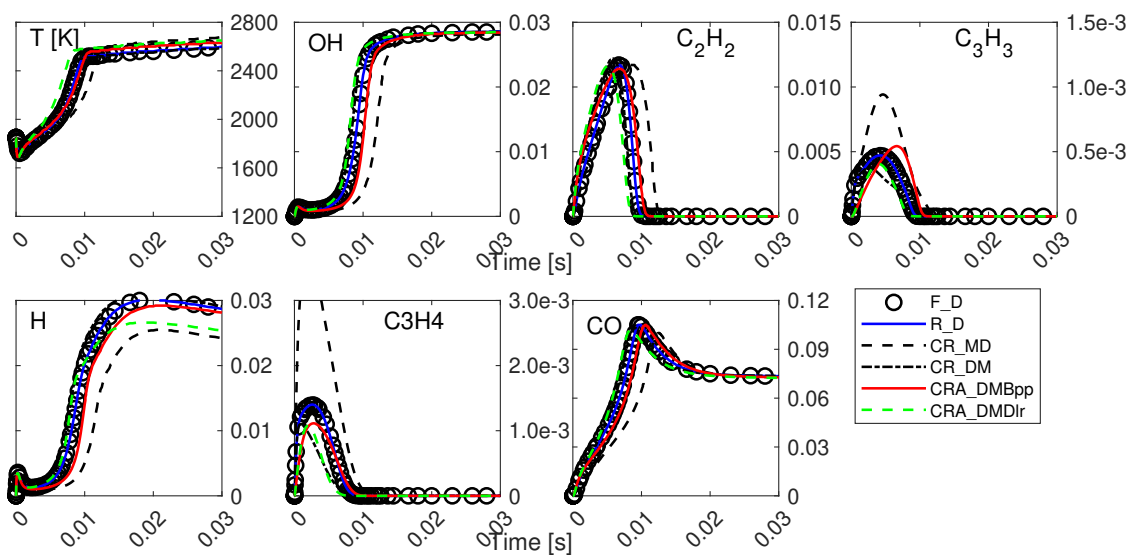
Unfortunately, the combined mechanisms fail to capture the combustion features of the added fuels. Therefore, a way should be considered to neutralise the 'partiality' of combined mechanisms: now starting from the CR\_DM mechanism because it accurately preserves the ignition delay times in both combustion mixtures, it is then combined with the PAH sub-mechanism: all the chemical rates and thermochemical properties from the CR\_DM mechanism are kept, combined with other reactions and species of the PAH sub-mechanism. Now, the reaction rates of common reactions are manually switched related to the monitored  $C_2H_2$ ,  $C_3H_3$ ,  $C_3H_4$ ,  $C_3H_5$  from three different sub-mechanisms. Appendix B summarises the reaction rates of common reactions inherited from the F\_M mechanism or the PAH sub-mechanism, instead of the prioritised F\_D mechanism. Finally, the adjusted combined MD-DBE-PAH mechanism, labelled as CRA\_DMBpp mechanism, is concluded, with 530 species and 3437 chemical reactions. Figure 2 also displays its performance in the PSR combustion. The CRA\_DMBpp mechanism accurately captures the ignition delay times and concentrations of a majority of species. The only evident discrepancy is in the profile of  $C_3H_3$ , which is about 1/3 reduction of the peak value than the F\_M reference in Fig. 2a, but it is still superior to the results of CR\_MD and CR\_DM mechanisms.

As the PAH formation is critical to soot formation, other chemical mechanisms for the PAH formation sub-mechanism are also referred in this study. The DLR mechanism proposed by Dworkin et al. [65] includes 97 gas species and 831 chemical reactions and has been validated for zero- and one-dimensional premixed flame systems. Slavinskaya and Frank [66] and then extended to a sooting ethylene/air diffusion flame in a coflow geometry [65]. By combining the DLR mechanism with the 'CR\_DM' mechanism, and the overlapped chemical rates and thermochemical properties from the mechanism 'CR\_DM', the formed mechanism 'CRA\_DMDlr', similar to the combined and reduced mechanism 'CRA\_DMBpp', also underpredicts  $C_3$  species in the stoichiometric MD-air mixture combustion (see Fig. 2a) with respect to the standard profiles of the full mechanism 'F\_M', and achieves a good agreement of soot precursor species  $C_2H_2$  and  $C_3$  species in the stoichiometric DBE-air mixture combustion.

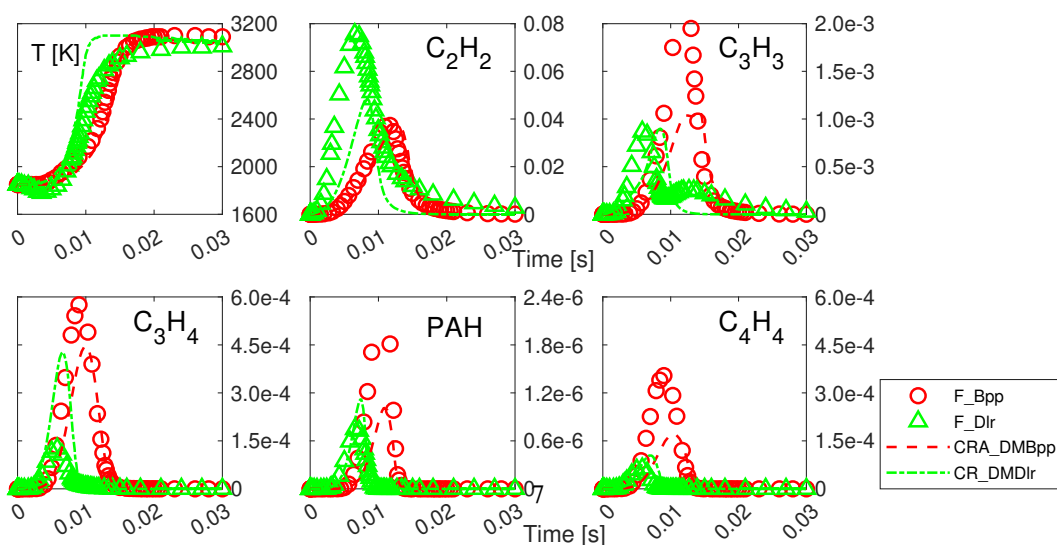
As the PAH species are the major precursors for soot formation modelling, Fig. 2 shows how the concentrations of PAHs are affected as the mechanism 'CR\_DM' is combined with the BPP and DLR mechanisms. As the PAH species are not included in the full MD (F\_M) and DBE (F\_D) mechanisms, a rich ethylene-air combustion in the homogeneous reactor is considered to compare the performance of the combined mechanisms 'CRA\_DMBpp' and 'CRA\_DMDlr', with the full PAH mechanisms BPP (F\_Bpp) and DLR (F\_Dlr). In fact, the BPP and DLR mechanisms show different predictions of the temperature profile and the soot precursors:  $C_2H_2$ ,  $C_3$  and PAHs. The combined mechanisms actually underpredict the soot precursor species, compared with their own full mechanism. This is because the carbon species flows into the added branch 'CRA\_DM'.



(a) stoichiometric MD-air mixture



(b) stoichiometric DBE-air mixture



(c) Ethylene-air mixture ( $\phi = 2$ )

**Figure 2:** Comparison of the simulations with full mechanisms (labelled as F), the reduced mechanism (labelled as R), combined-reduced mechanisms (labelled as CR), the adjusted combined-reduced DBE-MD-PAH mechanism (labelled as CRA) in a perfect stirred reactor

#### 2.4. Soot kinetics model

The population balance equation (PBE) is a statement on the balance of number density in  $(v, \mathbf{x})$ -space, with respect to source and sink processes for the creation and removal of number density [67]. At  $(\mathbf{x}, t)$ ,  $n(\cdot, \mathbf{x}, t)$  is often referred to as the particle size distribution. Considering soot nucleation and aggregation as source/sink processes, the PBE is given by

$$\frac{\partial n(v, \mathbf{x}, t)}{\partial t} + \sum_{j=1}^3 \frac{\partial ((u_j + U_j^T + K_j)n(v, \mathbf{x}, t))}{\partial x_j} + \frac{\partial (G(v, \mathbf{Y})n(v, \mathbf{x}, t))}{\partial v} = \dot{B}(\mathbf{Y})\delta(v - v_0) + \frac{1}{2} \int_0^v \beta(v-w, w)n(v-w, \mathbf{x}, t)n(w, \mathbf{x}, t) dw - \int_0^\infty \beta(v, w)n(v, \mathbf{x}, t)n(w, \mathbf{x}, t) dw, \quad (5)$$

where  $G(v, \mathbf{Y})$  is the combined surface growth and oxidation rate,  $\dot{B}$  represents the nucleation rate,  $v_0$  is the nuclei volume and  $\beta(v, w)$  denotes the aggregation kernel.  $K_j(v, \mathbf{x}, t)$ , moreover, represents the diffusive velocity the  $j$ th coordinate direction in physical space,

$$K_j(v, \mathbf{x}, t) = -\frac{D_p(v, \mathbf{x}, t)}{n(v, \mathbf{x}, t)} \frac{\partial n(v, \mathbf{x}, t)}{\partial x_j}, \quad (6)$$

and  $D_p(v, \mathbf{x}, t)$  represents the kinematic diffusivity of a particle with volume  $v$  [68].  $D_p(v, \mathbf{x}, t)$  is controlled by the diameter  $d_p(v)$  of spherical particles, or the hydrodynamic diameter  $d_h(v)$  for aggregates, as indicated in investigations on the dynamic properties of polymers suspended in a solvent [69].

In the spatial convection term of Eq. (5),  $\mathbf{U}^T(v, \mathbf{x}, t)$  represents the thermophoretic velocity that a particle immersed in a flow with temperature gradients experiences. Depending on the Knudsen number  $\text{Kn}$ , the thermophoretic velocity is computed for the free-molecular ( $\text{Kn} > 10$ ), transitional ( $0.1 < \text{Kn} < 10$ ) or continuum regimes ( $\text{Kn} < 0.1$ ) [70],

$$U_j^T = \begin{cases} -\frac{3\nu}{4\left(1 + \frac{\pi\alpha_m}{8}\right)} \frac{1}{T} \frac{\partial T}{\partial x_j}, & 10 < \text{Kn} \\ -\frac{2C_s\nu\left(\frac{k_g}{k_p} + C_t\text{Kn}\right)}{(1 + 2C_m\text{Kn})\left(1 + \frac{2k_g}{k_p} + 2C_t\text{Kn}\right)} \frac{1}{T} \frac{\partial T}{\partial x_j}, & 0.1 < \text{Kn} < 10 \\ 0, & \text{Kn} < 0.1 \end{cases}, \quad (7)$$

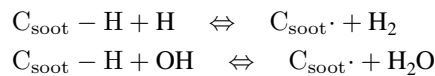
where the accommodation factor  $\alpha_m$  is set to 0.9.

Soot formation processes involve PAHs nucleation, surface growth by adding hydrocarbon species, and coagulation between spherical particles and fractal aggregates. The soot particle nucleation step is considered as the coalescence of two PAH molecules of same species and the collision frequency is given by

$$\dot{B}(\text{PAH}_i) = 4\gamma_i N_A d_i^2 \sqrt{\frac{\pi R_j T}{M_i}} [\text{PAH}_i]^2, \quad (8)$$

where  $R_j$  indicates the gas constant (in  $[\text{kmol} \cdot \text{K}/\text{kJ}]$ ),  $N_A$  is the Avogadro's number,  $d_i$  is the diameter of molecules  $\text{PAH}_i$  and  $M_i$ , represents the molar mass of  $\text{PAH}_i$   $[\text{kg}/\text{kmol}]$ . The PAH species and dimerisation efficiencies  $\gamma_i$  are given in Table 2 [36].

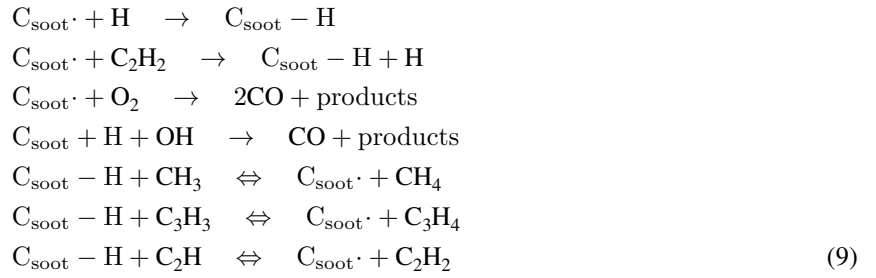
The hydrogen abstraction acetylene addition (HACA) mechanism contributes to the growth of benzenoid polycyclic aromatic hydrocarbons (PAHs) and soot particles, which is modelled as the following reversible reactions on the  $\text{C}_{\text{soot}\cdot}$  (radical sites on dehydrogenated carbon atoms) and the reactive sites  $\text{C}_{\text{soot}}\text{-H}$  (sites bonding hydrogen atoms and carbon atoms):





Species	Molar mass $M_i$ [kg/kmol]	Name	Collision efficiency $\gamma_i$
$C_{10}H_8$	128.17	Naphthalene	0.0010
$C_{12}H_8$	152.20	Acenaphthylene	0.0030
$C_{12}H_{10}$	154.21	Biphenyl	0.0085
$C_{14}H_{10}$	178.24	Phenathrene	0.0150
$C_{16}H_{10}$	202.26	Pyrene	0.0250
		Acphenanthrylene	0.0250
		Fluoranthene	0.0250
$C_{18}H_{10}$	226.28	Cyclo[cd]pyrene	0.0390

**Table 2:** List of PAH species in the nucleation and surface adsorption



A complete list of surface reaction rates of the enhanced HACA mechanism is in Reference [37]. In addition, all PAH molecules in Table 2 can also condense onto the surface of a soot particle with the same collision efficiency  $\gamma_i$  (Table 2). The rate of increase in soot mass per surface area (in  $[\text{kg}/\text{m}^2 - \text{m}^3]$ ) due to PAH adsorption is modelled as

$$\dot{R}(\text{PAH}) = \sum_i \gamma_i \sqrt{\frac{R_j T}{2\pi M_i}} [\text{PAH}_i] M c_i, \tag{10}$$

where  $M c_i$  (in  $[\text{kg}/\text{kmol}]$ ) is the carbon weight per mole for  $\text{PAH}_i$ . The surface growth rate is comprised of the HACA mechanism and the PAH adsorption process

$$G(v) = \frac{A_s(v)}{\rho_s} [\dot{R}(\text{HACA}) + \dot{R}(\text{PAH})], \tag{11}$$

where  $\dot{R}(\text{HACA})$  donates the surface-specific soot growth rate of the HACA mechanism and  $\rho_s = 1800 \text{ kg}/\text{m}^3$  is the density of solid soot. Within an aggregate, neighbouring primary particles are assumed to be in point contact such that the surface area  $A_s(v)$  of an aggregate is estimated as [71]

$$A_s(v) = A_{p,a} N_p(v) \left( 1 - \phi C_{ov} \left( 1 - \frac{1}{N_p(v)} \right) \right). \tag{12}$$

$$A_s(v) = \begin{cases} \pi d_p(v)^2, & N_p < 1 \\ A_{p,a} \left[ 1.72 N_p(v)^{\frac{2}{3}} - 0.72 \right], & 1 < N_p < 10. \\ A_{p,a} N_p(v) \left( 1 - \phi C_{ov} \left( 1 - \frac{1}{N_p(v)} \right) \right), & N_p > 10 \end{cases} \tag{13}$$

Here,  $A_{p,a}$  represents the surface area of a spherical particle with diameter  $d_{p,a}$ ,  $C_{ov} = 0.15$  is an overlapping parameter for neighbouring primary particles and  $\phi$  is defined as the coordination number (number of contacts per primary particles), approximated as 1.3 for large  $N_p(v)$ .

The collision kernel in the aggregation term for the free molecular and continuum regimes, respectively, is de-

scribed according to [40]

$$\beta^{\text{fm}}(v_i, v_j) = 2.2 \sqrt{\frac{\pi k_{\text{B}} T}{\rho_{\text{s}}(v_i + v_j)}} (d(v_i) + d(v_j))^2, \quad (14)$$

$$\beta^{\text{c}}(v_i, v_j) = \frac{2k_{\text{B}} T}{3\mu} \left( \frac{C(v_i)}{d(v_i)} + \frac{C(v_j)}{d(v_j)} \right) (d(v_i) + d(v_j)), \quad (15)$$

where  $k_{\text{B}}$  is the Boltzmann's constant,  $d(v)$  represents either the diameter of gyration of a fractal aggregate  $d_{\text{g}}(v)$  or the diameter of a spherical particle  $d_{\text{p}}(v)$  [50],

$$d_{\text{g}}(v) = \begin{cases} d_{\text{p}}(v), & N_{\text{p}} < 1 \\ d_{\text{p,a}} [-0.1067 N_{\text{p}}^3 + 0.6787 N_{\text{p}}^2 - 1.0219 N_{\text{p}} + 1.45], & 1 < N_{\text{p}} < 4 \\ d_{\text{p,a}} (0.1819 N_{\text{p}} - 4 + 0.6655), & 4 < N_{\text{p}} < 10 \\ d_{\text{p,a}} \left( \frac{N_{\text{p}}}{k_{\text{f}}} \right)^{\frac{1}{D_{\text{f}}}}, & N_{\text{p}} > 10 \end{cases} \quad (16)$$

In order to account for fractal shapes while maintaining a one-dimensional particle property space, we assume a soot particle to be spherical in shape if its volume is less than a critical value  $v_{\text{c1}}$ . On the other hand, it is assumed to be an aggregate consisting of several primary particles with identical volumes  $v_{\text{c1}}$  if the volume of this aggregate is greater than another critical value  $v_{\text{c2}}$ . In between these two limiting cases, soot particles may possess an intermediate shape whose morphological characteristics are detailed at the end of this section. We determined an average primary particle size of  $d_{\text{p,a}} = 30.8$  nm based on TEM images of thermophoretically sampled soot particles in laminar ethylene flames [50]. For this reason, the first critical size is defined as the average size of primary particles  $v_{\text{c1}} = v_{\text{p,a}}$ ; concomitantly, the second critical size is taken as  $v_{\text{c2}} = 10v_{\text{p,a}}$  [71]. The number of primary particles in a fractal-like aggregate can be estimated from the power law [68].

The authors acknowledge that the physics of soot sub-models are not yet fully understood, The research of soot physics is still in progress and thus there is no consensus in the academic community. For example, this work did not consider the recent developments of soot physics, *i.e.*, reversible dimerisation in the nucleation and condensation process [72], internal oxidation of porous nanoparticles [73], and the soot surface reactivity affected by soot ageing [54]. However, these newly understood physics have not been popularised in the modelling studies of soot formation. In fact, we cannot find a universally acknowledged soot kinetics which is utilised by all modellers. However, people have agreed on the general physics of soot formation pathways: soot processes usually include nucleation (the dimerisation of PAHs by [36, 37]), surface growth (HACA mechanism by [38] or ARS aromatic site model by [39], and the PAH adsorptions), multi-regime coagulation/agglomeration by [40] and sintering by [41]. Many numerical studies employed this framework of soot formation. The soot dynamics with the set of empirical parameters for the based on the BPP mechanism (Blanquart et al. [64]) to model the soot formation process in an ethylene diffusion flame (Liu et al. [50]). The simulation results were validated with the line-of-sight soot volume fraction, elastic light scattering (ELS) signals and OH-Planarlaser-induced fluorescence (OH-PLIF) signals. Therefore, in this study, we still used the mechanism of Blanquart et al. [64] and the soot dynamics, called the original soot kinetics 'OR'; as a comparison, the sensitivity analysis of the soot kinetics concerning the contribution of PAHs and  $\text{C}_2\text{H}_2$ , is also conducted by tuning the nucleation, growth and PAH adsorption rates, with the absolute soot volume fraction basically unchanged, called the adjusted soot kinetics 'AD' and listed in Table 3. In addition, parameters studies of the soot kinetics on individual sub-processes of soot formation are also conducted: including increasing the nucleation rate by 10 times, the growth rate by 2 times and the adsorption rate by 10 times, and reducing the average primary particle size  $d_{\text{p,a}}$  to 24.0 nm<sup>1</sup>. These parameter studies are called the soot kinetics 'N10', 'G2', 'A10' and 'D24' respectively.

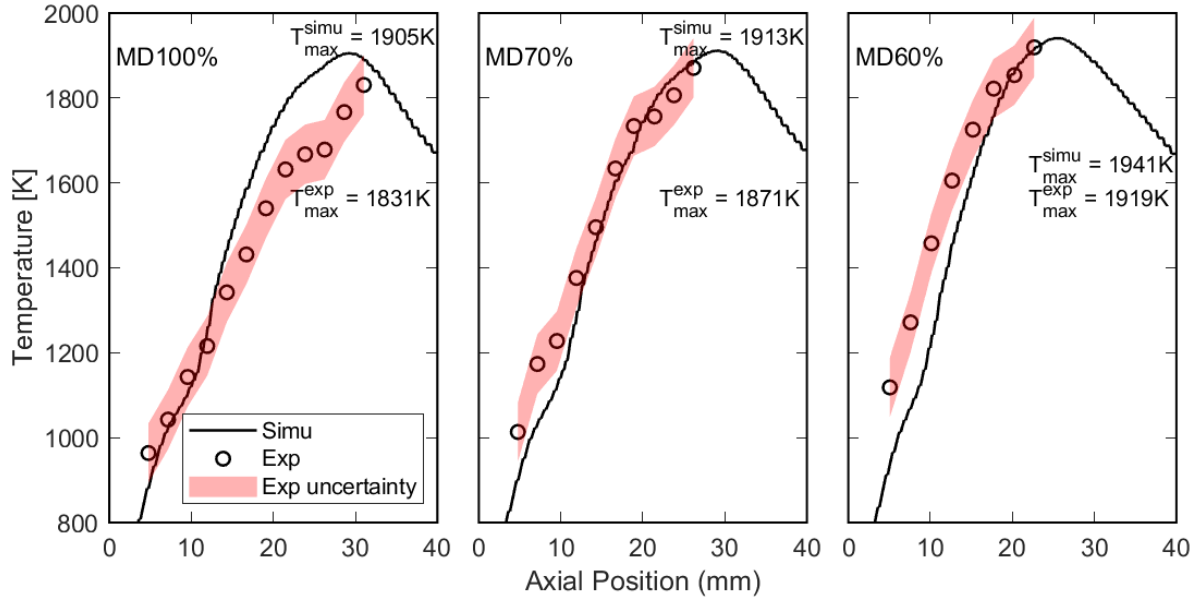
<sup>1</sup>In the dimethyl ether (DME) laminar coflow diffusion flame ([74]), the maximum soot volume fraction is less than 0.1 PPM and the average primary particle size is about ( $d_{\text{p}} = 8\text{-}10$  nm). However, from the measurements of this study for the blended MD/DBE flames, the maximum soot volume fraction is about 1 - 2 PPM, but unfortunately, the average primary particle size was not measured and thus unavailable. The assumption of  $d_{\text{p}} = 30.8\text{nm}$  in this work was estimated from a range of ethylene diffusion flames ([50, 75]) and the maximum soot volume fraction is about 5-10 PPM. But the authors believe that ( $d_{\text{p}}$ ) in this study of the blended MD/DBE flames should probably more approximate to the ones in ethylene diffusion flames rather than the dimethyl ether (DME) laminar coflow diffusion flame ([74])Based on the leverage ratio, the authors also

Case	Abbreviation	Nucleation rate	Growth rate	PAH adsorption rate	Average primary particle $d_{p,a}$
Original	OR	Eq. 8	$\dot{R}$ (HACA)	Eq. 10	30.8 nm
Adjusted	AD	$10 \times \text{Eq. 8}$	$0.5 \times \dot{R}$ (HACA)	$10 \times \text{Eq. 10}$	30.8 nm
Parameter studies	N10	$10 \times \text{Eq. 8}$	$\dot{R}$ (HCCA)	Eq. 10	30.8 nm
	G2	Eq. 8	$2 \times \dot{R}$ (HACA)	Eq. 10	30.8 nm
	A10	Eq. 8	$\dot{R}$ (HACA)	$10 \times \text{Eq. 10}$	30.8 nm
	D24	Eq. 8	$\dot{R}$ (HACA)	Eq. 10	24.0 nm

**Table 3:** Parameter studies on the soot kinetics

### 3. Simulation results and discussion

#### 3.1. Temperature profiles



**Figure 3:** Temperature profiles along the centreline between the experiment [5] and simulations

Figure 3 compares the computed temperature profiles on the centreline with the measurements. For all three flames, in general, the predicted temperature profiles are basically in good agreement with the experimental data, and both the measurements and simulations indicate the highest maximum temperatures with 40% DBE addition. However, the numerical solutions present a smaller difference among the three flames: in the region from 20 mm to 30 mm HAB in the pure MD flame, the predicted temperature is around 100 K to 150 K higher than experimental results; for the 60% MD flame, the temperature is predicted lower prior to 10 mm HAB, which represents that the calculated heat release rate is slower than the experiment. As we found that the predicted soot volume fraction in the centreline is smaller than the measurement (See Sec 3.2), so that the predicted heat radiation is smaller than the reality. This can be a reason for the overprediction of the temperature on the centreline in the simulation for the case 100% MD.

conducted a parameter study on the average primary particle size with  $d_p = 24$  nm, labelled as 'Dp24'. However, the prediction also shows only the moderate difference of soot volume fraction among the three flames with different DBE blend ratio.

### 3.2. Soot volume fraction

Figure 4a and 4b show contour plots of soot volume fraction in the experiment and simulation using the reduced combined mechanism CRA\_DMBpp and the original soot kinetics 'OR'. The two graphs are displayed in the same unit [ppm], implying that the soot model predicts a correct magnitude of soot volume fraction. Essentially, the swallow-tail shape of the soot regions is correctly captured by the simulations. Although the prediction shows the emergence of soot at around 7 mm upstream than in the experiment, the positions of soot decaying are correctly predicted in the flames of 100% MD and 70% MD. Basically, the model has captured the reducing tendency of soot formation in the measurement as the DBE addition increases from 0% to 40%. Quantitatively, however, in Fig.5, the predicted integrated soot volume fraction on the centreline reduces by about 18%-20% as the mole fraction of DBE increases from none to 40%, compared to a much larger reduction of around 55% in the measurement. The simulation with the adjusted soot kinetics 'AD' shows the most soot loading reduction on the centreline as a result of the DBE addition, however, still much lower than the experimental results. In particular, the experiment shows that soot loadings reduce with the addition of DBE, with a significant reduction being as the fraction of MD decreases from 70% to 60%. In the simulation, however, soot is just slightly reduced with the DBE addition. Besides, in the experiment, the position where soot particles entirely burnt out moves forward by 10 mm in the 60% MD case, while in the simulation, it is nearly similar to the case of MD 100%. Another shortcoming is the fact that the sooty wings are overpredicted by the simulation while more soot forms around the centre region in the experiment.

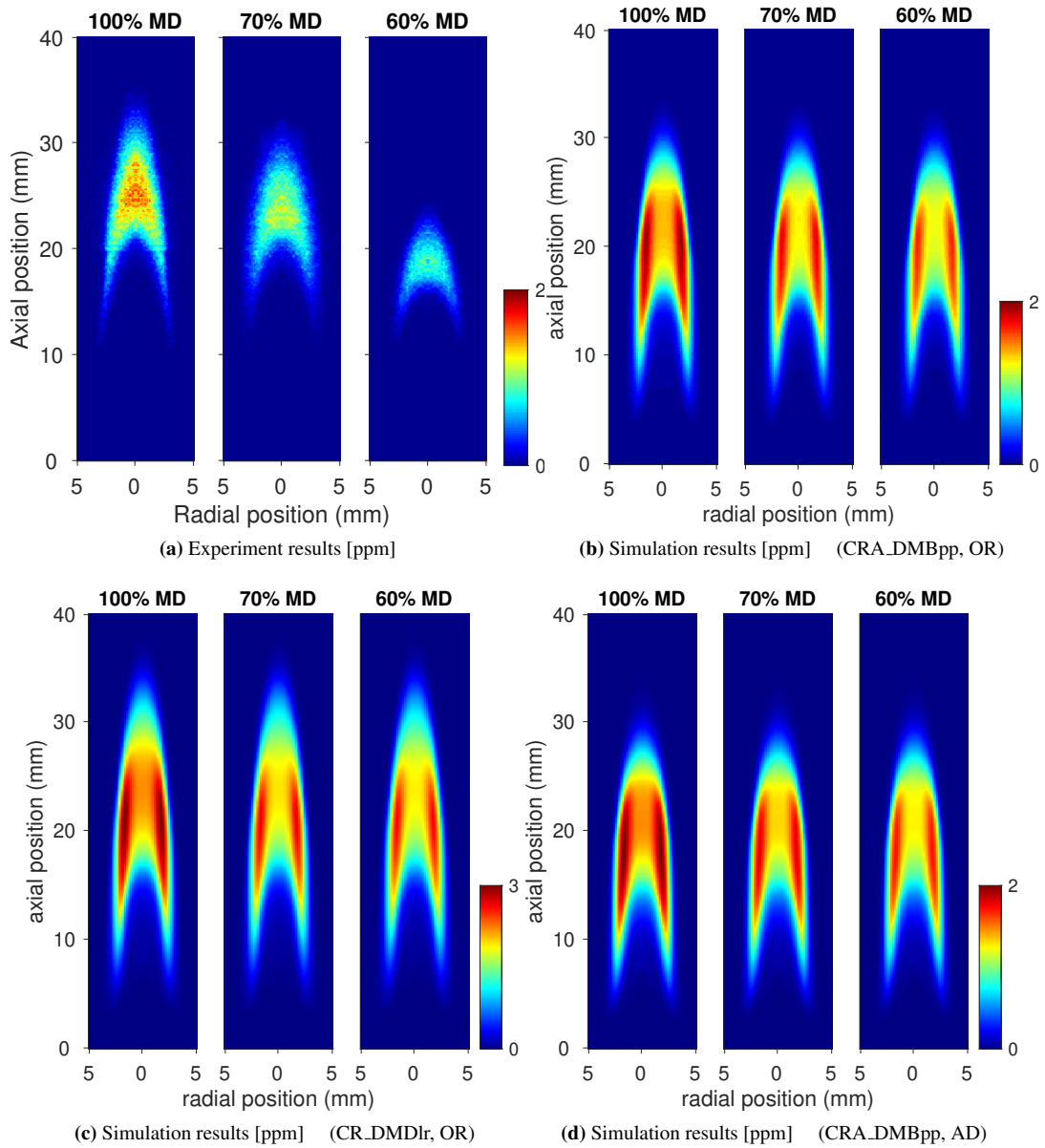
As a comparison, Fig. 4c and 4d show the simulated soot volume fraction using the mechanism CRA\_DMDlr (DLR) and the original soot kinetics 'OR', and using the mechanism CRA\_DMBpp (BPP) and the adjust soot kinetics 'AD', respectively. The simulation with the CRA\_DMDlr shows a larger soot volume fraction (with the legend scale 3 [ppm], see Fig. 4c) than with the CRA\_DMBpp (with the legend scale 2 [ppm], see Fig. 4d). This is because the DLR mechanism predicts higher  $C_2H_2$  concentration but a similar magnitude of  $C_3$ ,  $C_4$  and PAHs than the BPP mechanism (see Fig. 2c), and swallow-tails of the sooting region disappear all above 30 mm HAB. With the adjusted soot kinetics (CRA\_DMBpp, AD, increasing the nucleation and PAH adsorption rate by 10 times while reducing the surface growth rate by twice, see Fig. 4d), the contribution by the PAHs is increased while the contribution by the  $C_2H_2$  growth is reduced. As a result, in Fig. 4d, the soot formation on the flame centre is higher than the soot volume fraction in Fig. 4b. However, by changing the PAH chemical mechanism from BPP to DLR or the soot kinetics by balancing the contributions of PAHs and  $C_2H_2$ , the models still show an earlier emergence of soot formation upstream than in the experiment, less soot formation in the flame centre than on the flame wings and an underpredicted soot reducing capability of the DBE addition as compared to experimental data. Parameters studies on each individual sub-process of soot formation (nucleation rate, growth rate and adsorption rate, case 'N10', 'G2', 'A10') and the average primary particle size  $d_{p,a}$  (case 'D24') are shown Appendix E.

Figure 6a-c plot the radial profiles of the soot volume fraction at different heights above the burner in the three flames in the simulation using the CRA\_DMBpp mechanism and in the experiment. The uncertainty range of the soot volume fraction measured by the 2D LII signals are estimated in Appendix F. To keep the figures easily readable, uncertainty bars of soot volume fraction are included only for the flame of pure MD. Considering the uncertainty in the measurements, the soot volume fraction is basically well predicted at 22.5 mm and 27.5 mm HAB for the 100% MD and 70% MD flames although it is fully overpredicted at 17.5 mm. In the 60% MD flame, soot is completely burned out in the measurement while still exists in the simulation at 22.5 mm and 27.5 mm HAB heights.

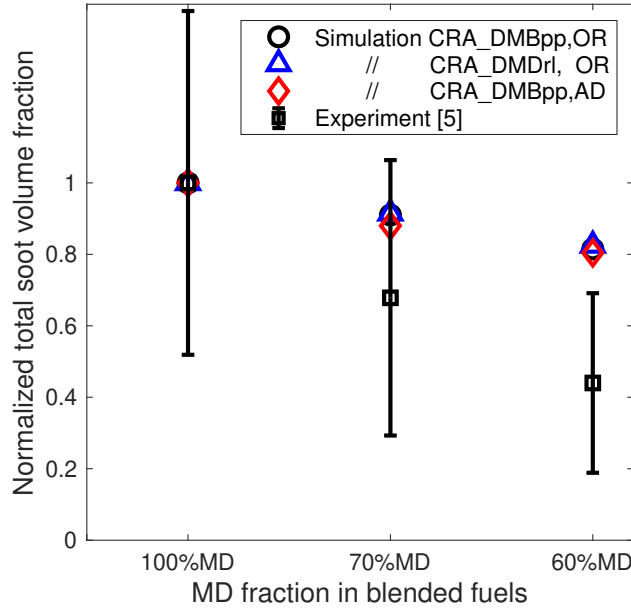
Figure 6d-e also plot the profiles of soot volume fraction  $f_v$  on the centreline and on the pathline of the peak  $f_v$  along the axial direction. The soot is predicted to appear 5-10mm upstream than the measured data on the flame centre of the 100% MD and 70% MD flames, while decays 5-10mm downstream than the measured data. For the 60% MD flame, the soot volume fraction is fully overpredicted on the flame centre and the peak pathline. On the whole, the model just shows the moderate difference of soot volume fraction among the three flames with different DBE blend ratio.

### 3.3. Rates of the nucleation, growth, PAH adsorption and oxidation processes

Figure 7 presents and compare the contributions of the processes of the nucleation, growth, PAHs adsorption and oxidation of the blended cases with the chemical kinetics 'CRA\_DMBpp' and the proposed soot kinetics 'OR'. Profiles at two heights (17.5 mm and 22.5 mm) and the flame centre are shown. In general, the nucleation rate, growth rate and the PAH adsorption rate are reduced as more DBE is blended into the MD medium while the oxidation rate



**Figure 4:** Comparisons of the soot volume fraction in contours between the experiment [5] and simulations



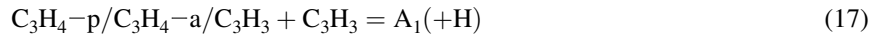
**Figure 5:** Normalised centreline soot loading as a function of the MD mole fraction between the experiment [5] and simulations

is increased. However, the difference of the soot sub-processes is quite small among the three blended cases, which determines the moderate difference of soot formation in the prediction.

### 3.4. Acetylene and PAH concentration

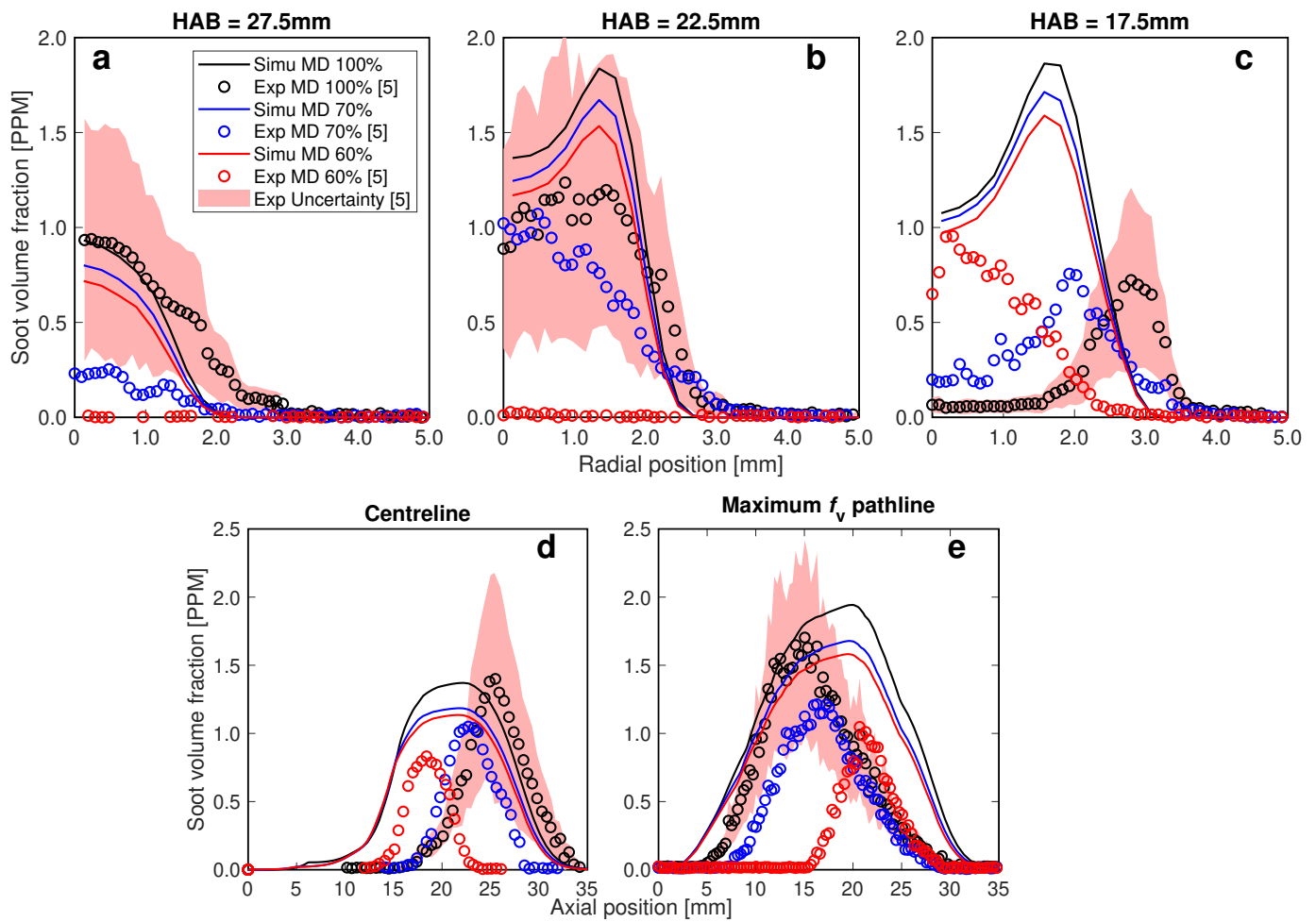
Figure 8 shows contour plots of mole fractions of  $C_2H_2$  and PAH in the simulation of three flames, where the mole fraction of PAH accounts for the sum of species  $A_2$ ,  $A_3$  and  $A_4$ . The population balance modelling of soot formation is based on the PAH-HACA mechanism, therefore, the concentrations of  $C_2H_2$  and PAH significantly affect soot formation. The  $C_2H_2$  concentration does not change with the DBE addition, while the PAH concentration is suppressed by adding DBE in the fuel blends, around 15% quantitatively smaller on the centreline in the 60% MD flame than in the pure MD, according to Fig.9. As  $C_2H_2$  contributes directly to the surface growth while the dimerisation of PAH lead to soot nucleation and adsorption, the soot reduction is mainly caused by a decreasing nucleation and adsorption rate. Here, the reason of only small difference of soot volume fraction among the three flames with different MD-DBE mixture ratios in the simulations can be found: the insignificant difference of soot precursor species ( $C_2H_2$  and PAH) in the combined MD-DBE-PAH mechanism.

Two different kinetics pathways: the combination of propargyl radicals and the reactions of  $C_4H_x$  species with acetylene, contribute to the cyclisation of the first aromatic rings ( $A_1$ ), which is a symbol to demonstrate the PAHs formation.

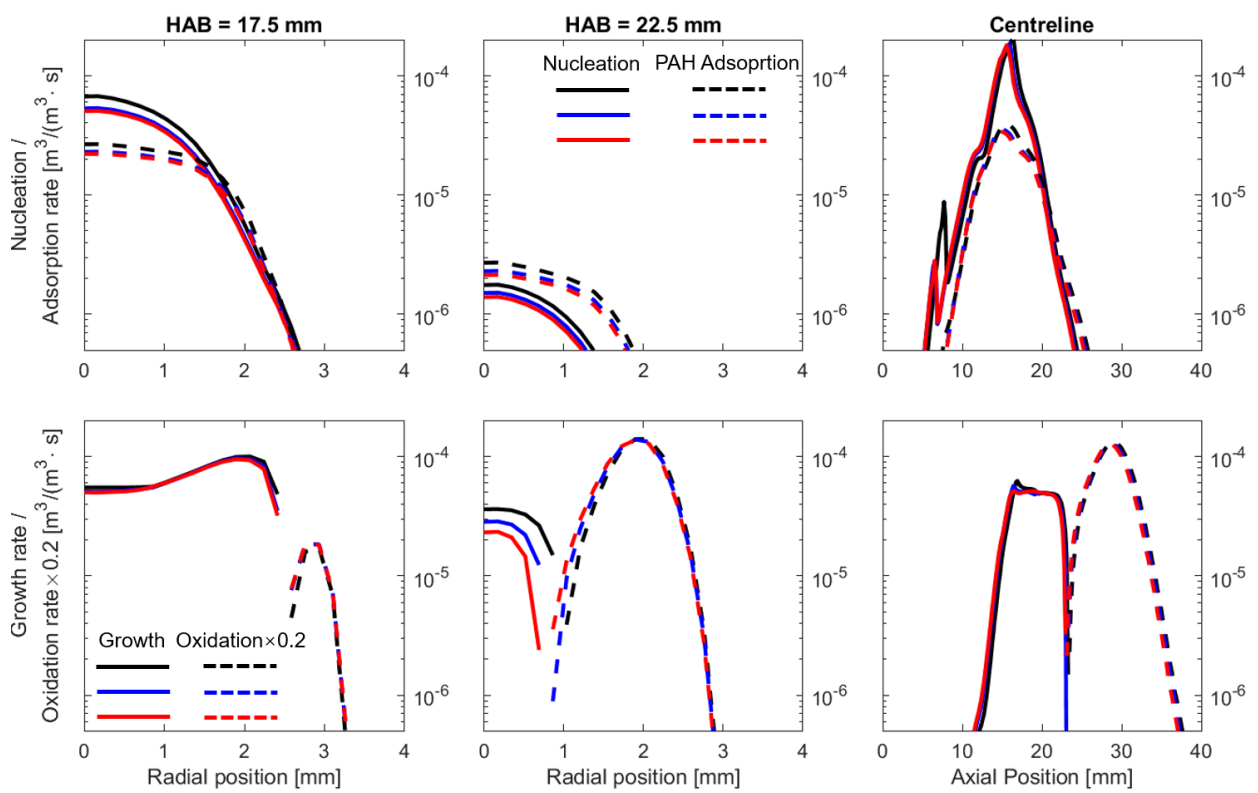


In Fig.9, the concentrations of species  $C_3H_4$ ,  $C_3H_3$  along the centreline show about 5% - 10% reduction as the DBE addition increases from none to 40%, which determines the reduced PAHs formation. For other species in the pathways of PAHs formation, the almost invisible difference can be found in the three flames. This also explains that differences in the PAHs concentration among the three flames are very small, thus leading to a smaller discrepancy of soot formation in the model than in the measurement.

Overall, the soot model can capture the reduction of soot formation by adding DBE into MD fuels. However, two main discrepancies between simulations and measurements arise, namely the early locations of soot occurrence and much slighter suppression of soot in the DBE-added cases.



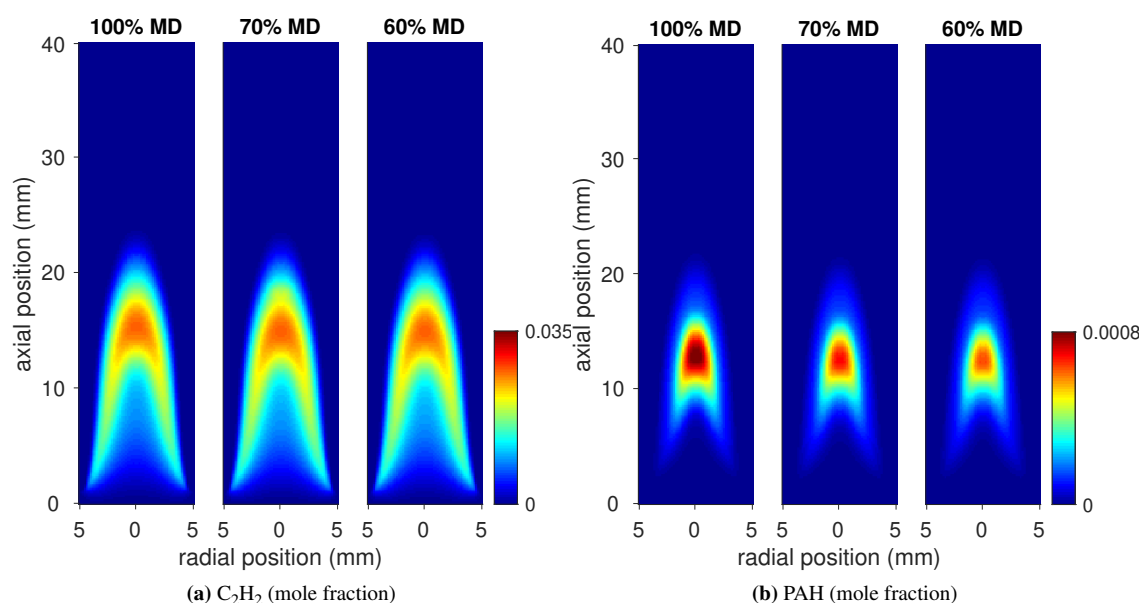
**Figure 6:** Radial profiles of soot volume fractions at different heights above the burner between the experiment [5] and the simulation based on the chemical mechanism 'CRA.DMBpp' and soot kinetics 'OR'



**Figure 7:** Radial profiles of the nucleation, surface growth, PAH adsorption and oxidation rates with the proposed chemical mechanism 'CRA\_DMBpp' and soot kinetics 'OR'



The problem should mainly be attributed to the chemical mechanism: concentrations of soot precursor species ( $C_2H_2$  and PAHs) and  $C_2/C_3/C_4$  molecules for PAH formation are similar in the three flames, although a convincing procedure to combine the MD-DBE-PAH mechanism and validate it (CRA\_DMBpp) in a perfect stirred reactor (see Fig.2) was conducted. In fact, the mechanisms CR\_MD and CR\_DM with the PAH sub-mechanism (not shown in the present paper) have been investigated, respectively, but neither of the combined MD-DBE-PAH mechanisms shows an evident difference between the combustion of MD and DBE in the laminar blended biodiesel flames, which is inadequate to model the suppression of soot formation by adding DBE into the blended fuel. In addition, the empirical parameters therein were calibrated based on ethylene diffusion flames [50]. The soot model could possibly be improved by adjustments based on soot morphological parameters in biodiesel fuel flames.



**Figure 8:** Contour plots of  $C_2H_2$  and PAH in the simulation based on the chemical mechanism CRA\_DMBpp

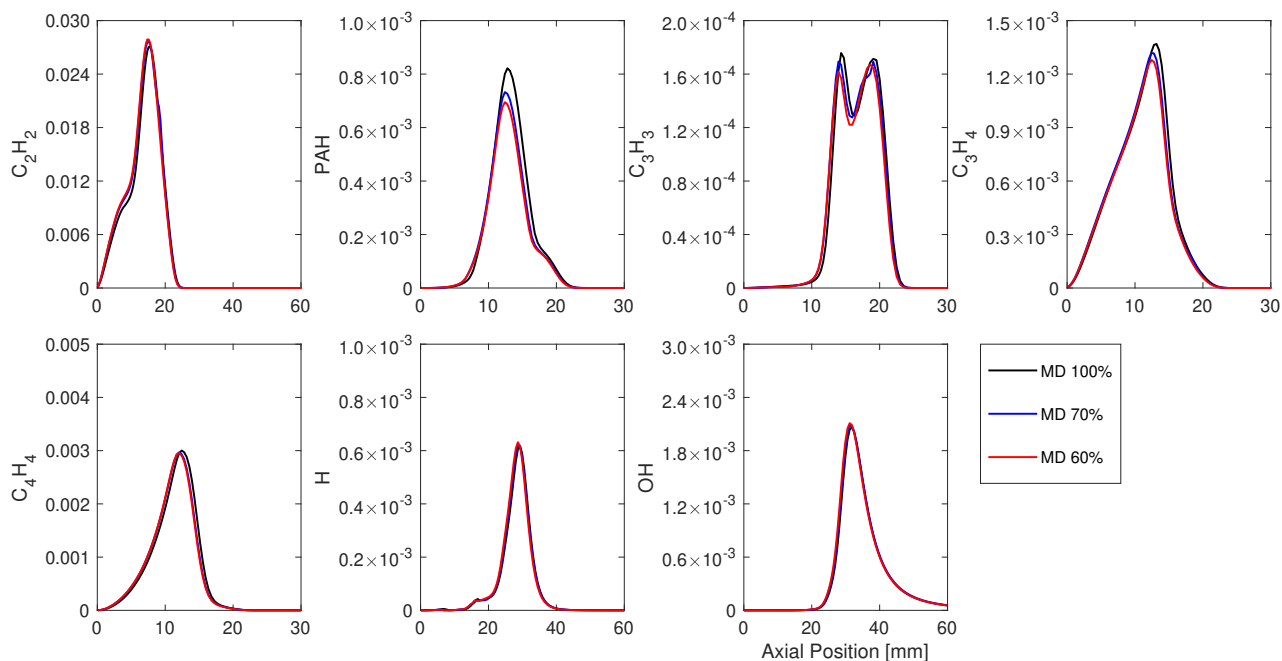
#### 4. Conclusions

The objective of the present paper was to investigate the modelling of soot formation in laminar diffusion flames with different oxygenated surrogate blends of methyl decanoate (MD) and dibutyl ether (DBE). The main research question was whether the soot suppression in flames with different MD/DBE ratios that has been observed experimentally could be predicted by the model.

Three detailed chemical sub-mechanisms, the MD, the DBE and the PAH sub-mechanism, were reduced and combined as the MD-DBE-PAH mechanism with 530 species and 3437 reactions. In the spatially inhomogeneous combustion with fuels of heavy molecular weights, the mass diffusion term should be expressed in term of mole fraction in the transport equations of species and energy, while the molecular thermophoresis process and Dufour effect are negligible. The discretised population balance modelling of soot formation involves the processes of nucleation (PAH dimerisation), surface growth (HACA and PAH adsorption) and coagulation of spherical particles and fractal aggregates.

The predicted temperature profiles are basically in line with the experiment along the flame centrelines. In terms of the position and quantity of soot formation, the swallow-tail shape of the soot occurrence zone and the absolute value of soot production are correctly captured by the simulation. In terms of the soot suppression effect, the model has basically captured the reducing tendency of soot formation in the measurement as the DBE addition increases from 0% to 40%. Concentrations of PAHs and  $C_3$  species contributing to the formation of aromatic rings (PAHs) are reduced as the DBE addition is up to 40%, which is a leading cause of soot suppression. However, on the whole, the

numerical solution featured much smaller differences than those observed in the experiment among laminar flames with different MD/DBE ratios. The reason has been found: the insignificant difference of soot precursor species ( $C_2H_2$  and PAHs) in the combined MD-DBE-PAH mechanism, in which the main error is rooted in the combination process of sub-mechanisms. Given that so far there have been very few attempts to model soot formation in biodiesel flames, especially for the blends with different oxygenated fuels, the results are encouraging but future studies are warranted.



**Figure 9:** The profiles of concentrations of C2, C3, C4 and PAH species on the centreline

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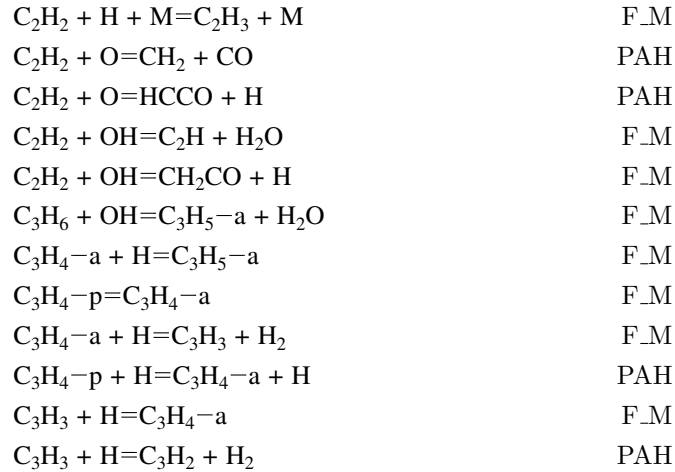
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**Appendix A Target species and corresponding relative tolerance errors in the chemistry reduction for the MD and the DBE mechanisms, using DRGEPSA algorithm**

relative tolerance	target species for chemistry reduction
10%	C <sub>2</sub> H <sub>2</sub> C <sub>3</sub> H <sub>3</sub> DBE CO <sub>2</sub> CO H MD O <sub>2</sub> OH Temperature
15%	C <sub>2</sub> H <sub>3</sub> C <sub>3</sub> H <sub>2</sub> C <sub>3</sub> H <sub>4</sub> -a C <sub>3</sub> H <sub>4</sub> -p C <sub>3</sub> H <sub>5</sub> -a C <sub>4</sub> H <sub>4</sub> C <sub>4</sub> H <sub>5</sub> -n C C <sub>2</sub> H C <sub>2</sub> H <sub>3</sub> CHO C <sub>2</sub> H <sub>5</sub> C <sub>2</sub> H <sub>6</sub> C <sub>3</sub> H <sub>5</sub> -s C <sub>3</sub> H <sub>5</sub> -t
20%	C <sub>3</sub> H <sub>5</sub> O C <sub>3</sub> H <sub>6</sub> C <sub>3</sub> H <sub>8</sub> C <sub>3</sub> H <sub>6</sub> CH CH <sub>2</sub> CHO CH <sub>2</sub> CO CH <sub>2</sub> O CH <sub>2</sub> OH CH <sub>3</sub> CHO CH <sub>3</sub> O CH <sub>3</sub> O <sub>2</sub> CH <sub>3</sub> OH CH <sub>4</sub> H <sub>2</sub> H <sub>2</sub> CC H <sub>2</sub> O H <sub>2</sub> O <sub>2</sub> HCCO HCCOH HCO HO <sub>2</sub> iC <sub>3</sub> H <sub>5</sub> OH iC <sub>3</sub> H <sub>7</sub> iC <sub>4</sub> H <sub>7</sub> iC <sub>4</sub> H <sub>7</sub> O iC <sub>4</sub> H <sub>8</sub> nC <sub>3</sub> H <sub>7</sub> nC <sub>3</sub> H <sub>7</sub> CHO nC <sub>3</sub> H <sub>7</sub> O O

**Appendix B In the CRA\_DMBpp mechanism, the common chemical reactions among three sub-mechanisms F\_M [62], F\_D [63] and PAH [36] whose rate constants are succeeded from F\_M or PAH instead of F\_D**



**Appendix C Evaluation of additional diffusion terms in transport equations**

The molecular weights of gasoline, diesel and biodiesel fuels, are usually in the range of 100 - 250. The transport equations of species and energy are usually numerically implemented in a simple format, by neglecting three additional terms (the effect of molecular weight, thermophoresis and Dufour effect). In this section, these diffusion terms in the species transport equations and the energy equation are evaluated in the laminar flame with pure MD fuel. From Fig.C.1a, the transport equations with respect to H and MD species, respectively, are selected to evaluate the difference of three diffusion components since the molecules H and MD are of the minimum and maximum molecular weights, respectively in the flame.

Three diffusion components, ordinary diffusion term, effect of the average molecular weight and thermophoresis, are marked as 'O', 'M' and 'T' in the legend. The ordinary term oscillates in the region where the species concentrations increase and decrease (the right part of Fig.C.1b). By comparison, the molecular weight term and thermophoresis are negligible in the transport equation of H, but the molecular weight term in the transport equation of MD is about 1/5 of the ordinary term in the transport equation of MD. Therefore, for the transport equations of heavy molecules, the molecular weight term in Eq. 2 should be included.

On the other hand, in Fig.C.1c, additional diffusion components account rednotably in the energy equation, compared with the ordinary diffusion term. 'M', 'T', 'D', 'O' in the legend represent the term of molecular weight effect, thermophoresis, Dufour effect and the ordinary diffusion term, respectively. The term of molecular weight effect is a

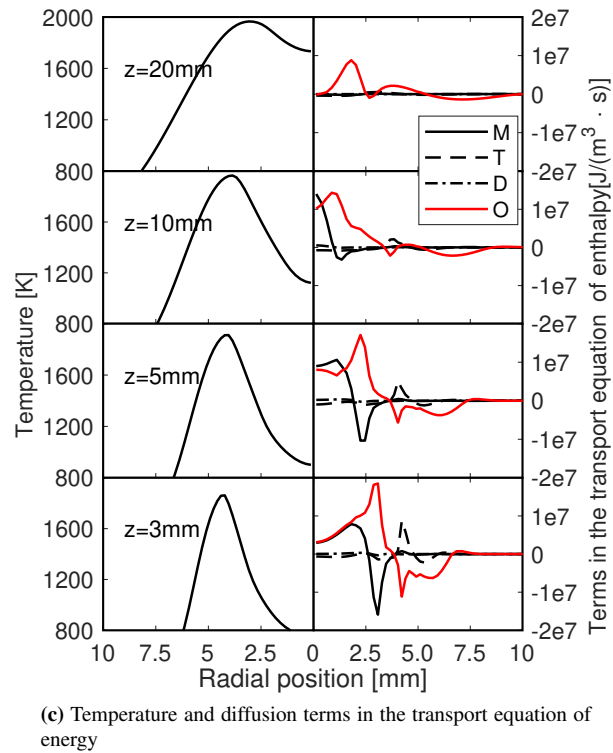
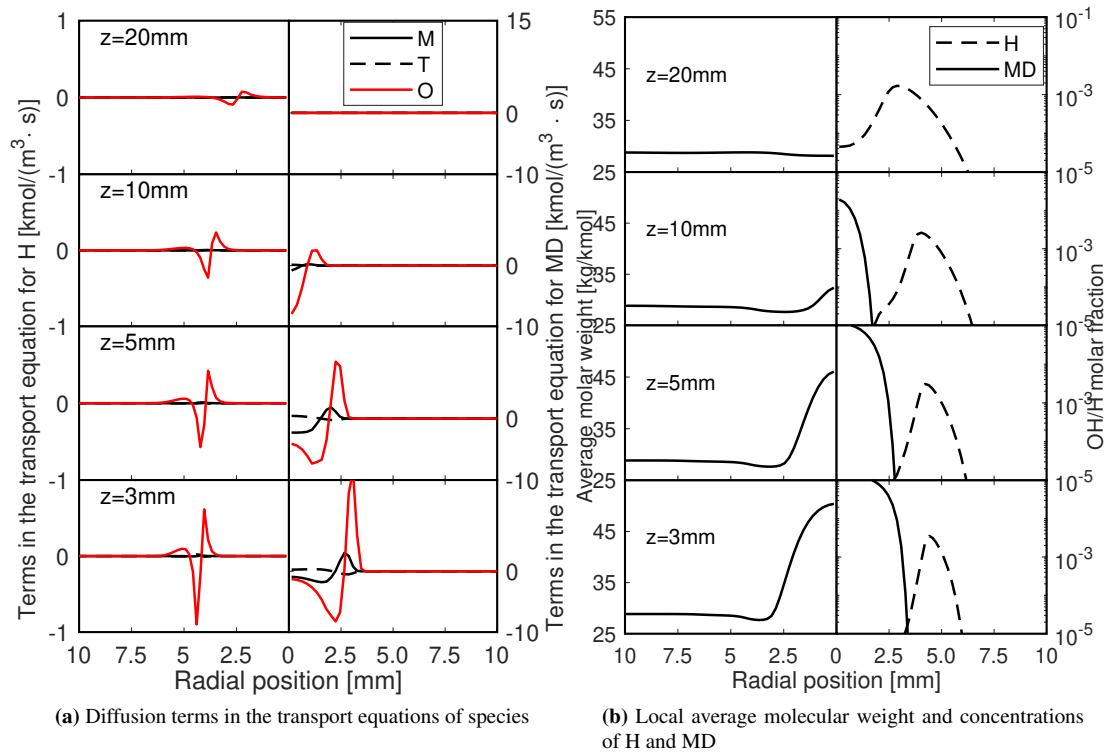
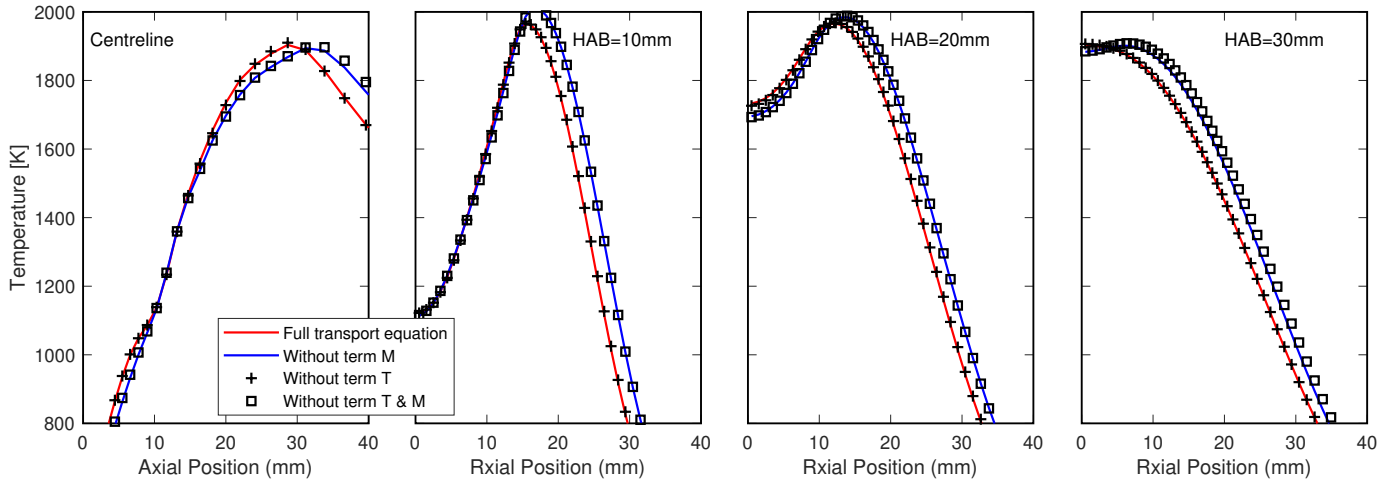


Figure C.1: Comparison of diffusion terms in the transport equations of species and energy

positive value on the centreline close to the jet burner until 10 mm above, corresponding to the region of high average molecular weight in Fig.C.1b. Then the effect becomes negative in the radial direction due to the varying average molecular weight, compensating the positive ordinary diffusion term. Around the reacting zone of peak temperature (the left column in Fig.C.1b) and species source (the right column of Fig.C.1b), thermophoresis effect contributes an opposite equivalence to the ordinary diffusion term, although the absolute values are minor compared with those around the centreline. Different from transport equations of species (Eq. 2), no source term exists in the energy equation in terms of total enthalpy. Therefore, the additional diffusion components largely affect the energy equation, thus the temperature field.

Figure C.2 compares the temperature profiles using four different expressions of transport equations: with full terms, the molecular weight term removed, the thermophoresis term removed, both terms of the molecular weight and thermophoresis removed. By removing the molecular weight term, the temperature has increased by 50 K - 100 K at 30 mm HAB downstream on the centreline and by around 50 K at 15 mm - 30 mm radial locations away from the centreline on different heights above the burner. But thermophoresis term makes no difference. Therefore, the mass diffusion term should be expressed in term of mole fraction in the transport equations of species and energy, while the molecular thermophoresis process and Dufour effect can be negligible.



**Figure C.2:** Temperature profiles using different expressions in the transport equations

#### Appendix D Analysis on the diffusive and thermophoretic velocity of soot

As the effects of secondary diffusion terms in the transport equation of species and energy are elaborated in Appendix C, the effects of the soot diffusion and soot thermophoretic velocity will be discussed in this section. A detailed description on how the soot diffusivity and thermophoretic velocity were modelled is given.

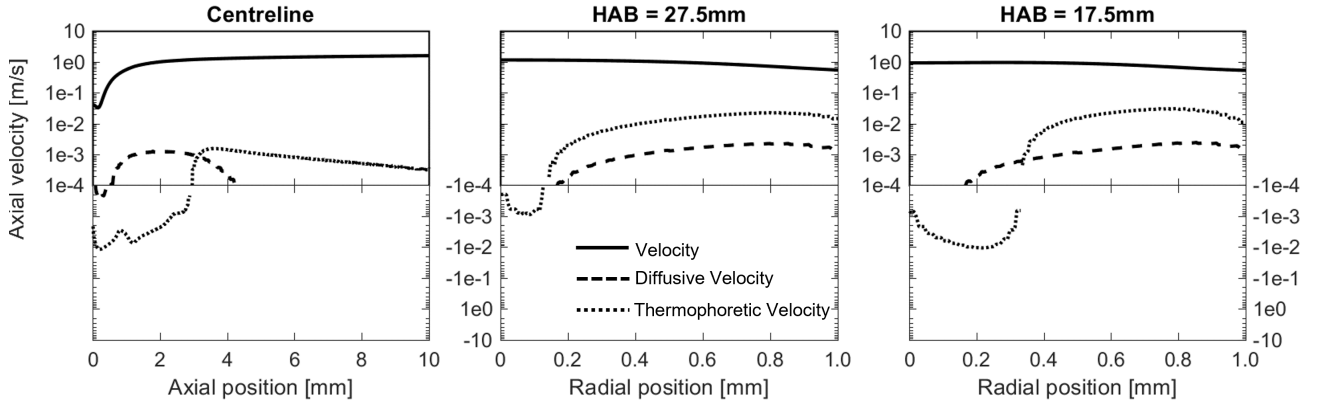
The average diffusivity velocity and thermophoretic velocity of soot particles are displayed as

$$\overline{K_j} = \frac{\sum K_j(vn(v))}{\sum n(v)} \quad (D.1)$$

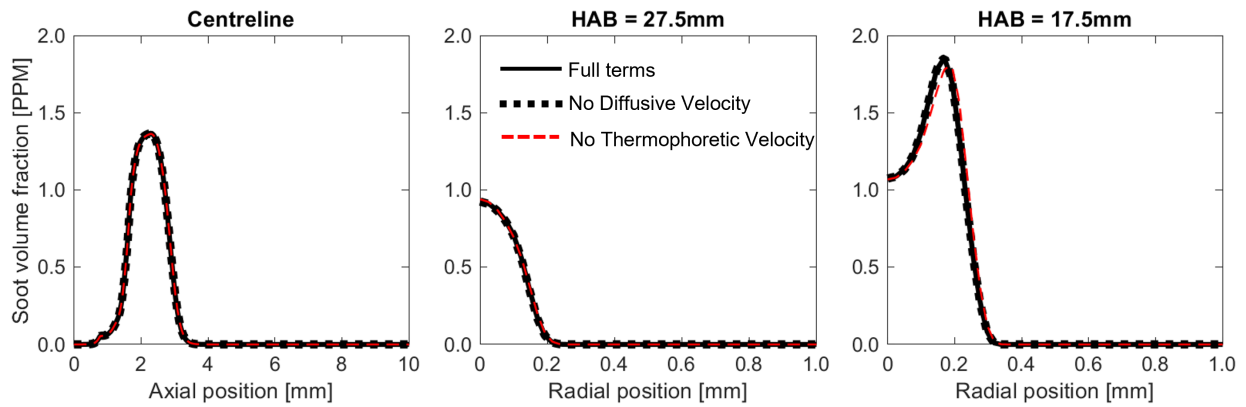
$$\overline{U_j^T} = \frac{\sum U_j^T vn(v)}{\sum n(v)} \quad (D.2)$$

Figure D.1a shows the contributions of the soot diffusion velocity and soot thermophoretic velocity on the flame centreline and at 17.5 mm and 27.5 mm height above the burner. The values of thermophoretic velocity are around 2 orders of magnitude lower than the bulk velocity while the diffusion velocity is around 3 orders of magnitude lower. Therefore, in Figure D.1b, by removing the diffusion velocity, the predicted soot volume fraction is not affected, while

by removing soot thermophoretic velocity, the profile of soot volume fraction changes slightly, particularly at HAB = 17.5 mm.



(a) Mass velocity, diffusive velocity and thermophoretic velocity of soot particles



(b) Soot volume fraction for the cases with the full terms, no diffusive velocity and no thermophoretic velocity

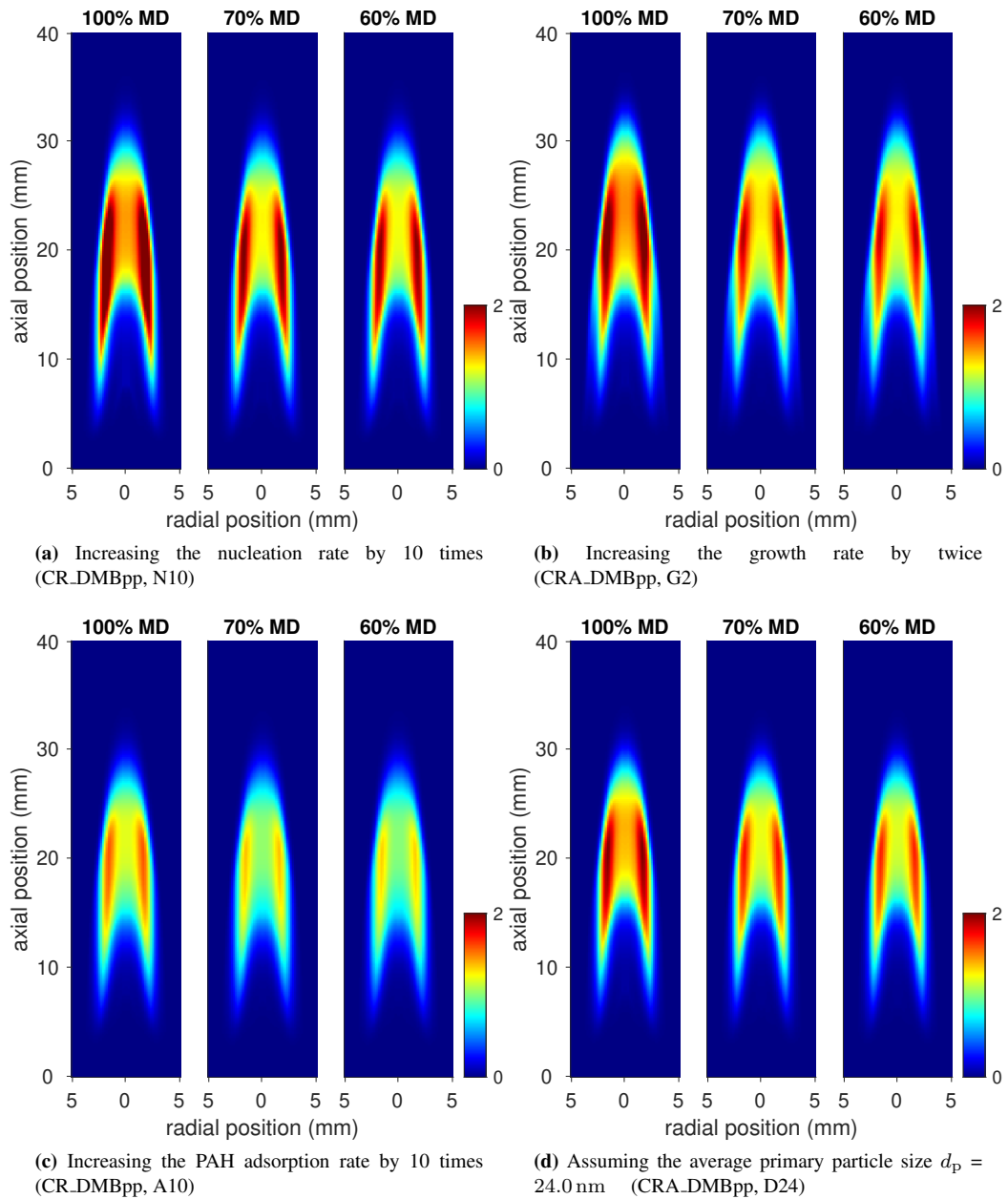
**Figure D.1:** Effects of soot diffusional terms

## Appendix E Parameter study of the nucleation rate, growth rate, adsorption rate and the average primary particle size in the proposed soot formation model

Parameters studies on each individual sub-process of soot formation (nucleation rate, growth rate and adsorption rate, case 'N10', 'G2', 'A10') and the average primary particle size  $d_{p,a}$  (case 'D24') are shown in this appendix.

Figure. E.1 shows the simulated soot volume fraction using different soot kinetics 'N10', 'G2', 'A10' and 'D24', respectively based on the mechanism CRA\_DMBpp (BPP). By increasing the nucleation rate or surface growth rate, individually or reducing the assumed average primary particle size  $d_{p,a}$ , the predicted absolute soot volume fraction has increased, comparing with the original soot kinetics 'OR' (see Fig. 4b). However, the models still show an earlier emergence of soot formation upstream than in the experiment, less soot formation in the flame centre than on the flame wings and an underpredicted soot reducing capability of the DBE addition as compared to experimental data (see Fig. 4a).





**Figure E.1:** Simulation results of contours plots of the soot volume fraction for the parameter study of the nucleation rate, growth rate, adsorption rate and the average primary particle size in the proposed soot formation model

## Appendix F Uncertainty analysis of the local soot volume fraction by using LII measurements

In this study, the local soot volume fraction  $f_v$  is determined by the 2D LII signals  $S_{LII}$  [5, 58],

$$f_v = \frac{S_{LII}}{K_{LII}} \quad (F.1)$$

The calibration constant  $K_{LII}$  is the ratio of the mean collected LII signal at emission signal wavelength per unit soot extinction, and is obtained by connecting the measured extinction signals  $S_{LII}$  to the corresponding integrated soot volume fraction  $f_v$ , as follows,

Therefore, the calibration constant  $K_{LII}$  can be obtained using:

$$K_{LII} = \frac{1}{\ln I_o - \ln I(0)} \frac{6\pi E(m)}{\lambda_e} \int_{-\infty}^{+\infty} S_{LII}(r) dr \quad (F.2)$$

where  $I_o$  represents the intensity of the incident laser beam and  $I(0)$  is the intensity of the transmitted beam passing the sooting flame centre.  $\lambda_e$  is the laser wavelength.  $E(m)$  is the soot absorption function:

$$E(m) = -\text{Im} \left( \frac{m^2 - 1}{m^2 + 2} \right) \quad (F.3)$$

where  $m$  is the complex index of refraction of soot. Based on Eq. F.1, the relative uncertainty of  $f_v$  is

$$\begin{aligned} \frac{\Delta f_v}{f_v} &= \frac{1}{f_v} \left( \frac{\Delta S_{LII}}{K_{LII}} - \frac{S_{LII} \Delta K_{LII}}{K_{LII}^2} \right) \\ &= \frac{\Delta S_{LII}}{S_{LII}} - \frac{\Delta K_{LII}}{K_{LII}} \end{aligned} \quad (F.4)$$

That is

$$\frac{|\Delta f_v|}{f_v} = \frac{|\Delta S_{LII}|}{S_{LII}} + \frac{|\Delta K_{LII}|}{K_{LII}} \quad (F.5)$$

The statistic uncertainty of the LII signals  $\frac{|\Delta S_{LII}|}{S_{LII}}$  were obtained directly from the standard deviation through the statistical analysis of at least 40 LII images.

The systematic errors in  $K_{LII}$  mainly arises from the assumed value for  $E(m)$ . Tian et al. [58] collated values of  $m$  and  $E(m)$  from previous investigations on smoking flames with various laser wavelengths and fuels. For a wavelength of about 632 nm, the values of  $E(m)$  were reported to range from 0.17 to 0.37. In this study,  $E(m)$  is assumed to be 0.26, based on the estimated value of the index of refraction  $m = 1.57 - 0.56i$  ([76]). Hence,

$$\frac{|\Delta K_{LII}|}{K_{LII}} \approx \frac{|\Delta E(m)|}{E(m)} = 0.42 \quad (F.6)$$

The error bar of  $f_v$  has been corrected in Figure 5 and 6, considering both the statistic uncertainty of the 2D LII signals  $S_{LII}$  and the systematic uncertainty of  $E(m)$ .